

INVESTIGATION INTO THE BEHAVIOR OF GEOMETRICALLY
NONLINEAR COMPOSITE ARCHES

THESIS

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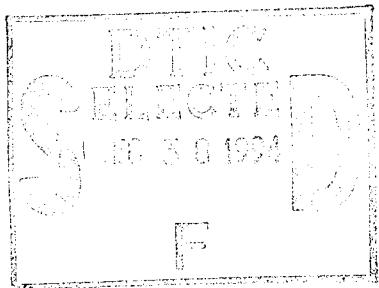
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NONLINEAR COMPOSITE ARCHES

THESIS

Presented to the Faculty of the Graduate School of Engineering
of the Air Force Institute of Technology
Air University
In Partial Fulfillment of the
Requirements for the Degree of
Master of Science in Aeronautical Engineering

John C. Bailey

Captain, USAF

December 1994

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Although only one name is printed on the title page of this thesis, it is by no means the work of just one person. As such, I must thank Dr Anthony Palazotto for pounding the concepts related to nonlinear shell theory into my brain despite great unconscious resistance. I would also like to thank Dr Arnold Mayer (WL/FIV) and Dr Arje Nachman (AFOSR) for sponsoring my efforts. Most importantly, I need to thank my wife Camille and my son Tyler, because these times were no easier on them than they were on me.

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ABSTRACT

This research modifies the existing finite element formulation of a potential energy based large deformation and moderate rotation theory. Hermitian shape functions replace the existing linear bending angle interpolations. Negligible differences between the two formulations indicate the underlying kinematics limit the accuracy, not the finite element interpolations. Using the new program, numerous nonlinear arch geometries are modeled to investigate the effects of arc length and thickness variations. Local and global snapping phenomena are captured as well as through the thickness shear driven nonlinearities.

INVESTIGATION INTO THE BEHAVIOR OF GEOMETRICALLY
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I. Introduction

1.1 Background

Composite shells, plates, arches, and beams are the building blocks for the majority of today's complex aerospace structures. The critical nature of such structures requires accurate prediction of their behavior under any loading conditions. However, analysis beyond the basic linear geometric and material regions is complex and time consuming.

One type of complex shell analysis combines small strains with large displacements and rotations. In this situation, the material remains linearly elastic while the geometry becomes non-linear. Geometric non-linearity causes dramatic structural stiffness changes without significant material changes. Palazotto and Dennis present the Simplified Large Displacement/Rotation (SLR) theory and complementary FORTRAN code to analyze shells experiencing these loading conditions (19). SLR theory determines the equilibrium path of orthotropic shells using a total Lagrangian approach and includes a parabolic transverse shear stress distribution. Equations are solved using the finite element method. The result is a FORTRAN code capable of analyzing linearly elastic behavior of isotropic and laminated orthotropic cylindrical shell structures undergoing geometric non-linearity with large deformations and moderate rotations.

Creaghan reduced SLR theory to one dimension for the analysis of beams and arches (3; 4). The resulting FORTRAN code combines all the

features of Palazotto and Dennis's work with extensible mid-plane strain terms from Smith (25; 26). Creaghan also include displacement control and Riks methods for solving non-linear equations. These modifications allow uninhibited observation of SLR displacement and rotation limits. Creaghan's theory produces accurate solutions through 23 degrees of rotation.

Miller improved Creaghan's rotation limits to approximately 45 degrees (13). Miller replaced Creaghan's small angle approximation for the bending angle (ψ) with a truncated series representation of the tangent function, thereby introducing large rotation kinematic theory to Creaghan's formulation.

The purpose of this thesis is to improve the rotation limit of Miller's code. Miller's adaptation of the finite element method (FEM) will be altered to include higher order shape functions for ψ . Refining the finite element model will improve solutions within the limits of Miller's theory. The resulting FORTRAN code will then be used to analyze arch classification parameters.

1.2 Preview

Thickness and depth primarily determine the behavior of shells and arches. For example, shear plays a large role in thick structures but is negligible in thin shells; therefore, a theory which considers transverse shear should be used for analysis. Also, it is important to note a shallow shell/arch behaves very much like a plate/beam whereas a deep shell/arch does not. Obviously, the geometry of the structure should be classified before analysis begins.

Huang proposes thin arches have a very small physical thickness (h) vs. radius of curvature (r) ratio ($h/r \ll 1$) (10). Miller also

considers the half chord length (c), but his decision value ($c/h < 25$) is arbitrary (13).

Smith proposes a shallow shell has a small depth (δ) vs. half chord length ratio ($\delta/c < 0.3$) (25). Fung and Kaplan propose $\lambda = 2[3(1-\nu^2)]^{.25}(\delta/h)^5$ as a classification criteria, limited to small opening angle ($\alpha \ll 1$) (8). Kaplan revises this definition to remove the opening angle limitation resulting in $\lambda = [12(1-\nu^2)]^{.25}(r/h)^5(\alpha/2)$ (12). Miller uses $\lambda > 8$ to define deep shells (13).

The complex nature of large deformation and rotation theory requires solving numerous simultaneous, non-linear, differential equations. Even with simplifying assumptions, these equations remain complex and extensive. For this reason, most theories employ FEM for solution.

Accurate FEM solutions require proper shape functions. Cook, Gallagher, and Zienkiewicz all discuss shape functions at great length (2; 9; 29). Shape functions interpolate quantities within an element and are generally classified as C^0 or C^1 continuous. All shape functions must be C^0 continuous; that is, the quantity they interpolate must be continuous at each node. If a quantity is represented by C^1 shape functions, the first derivative of the quantity is continuous as well. Obviously, C^1 or Hermitian shape functions are required if a quantity and its first derivative must have interelement continuity. Miller uses linear C^0 shape functions to estimate ψ (13). As such, the first derivative of ψ (ψ_2) is constant within a given element and discontinuous at each node. The current work will employ both quadratic C^0 and Hermitian interpolations of ψ .

1.3 Literature Review

Although primarily concerned with non-linear shell theory, this thesis also addresses arch classification parameters and finite element method shape functions. Therefore, a brief review of these topics is in order.

Kirchhoff's theory is one of the first for analysis of thin plates (22). His assumptions allow two dimensional analysis of three dimensional plate behavior. He assumed an inextensible mid plane and negligible normal strains. Kirchhoff also eliminated transverse shear effects by disallowing cross sectional warping.

Love took the next logical step in the development of shell theory (22). He adapted Kirchhoff's plate theory to curved shells. The Kirchhoff-Love theory obviously neglects transverse shear strain effects. As a result, both Kirchhoff's theory and Love's revision only apply to thin, isotropic plates experiencing small deflections.

Reissner and Mindlin improved Kirchhoff-Love theory through the addition of transverse shear considerations (14; 21). Reissner-Mindlin (RM) theory assumes constant through the thickness shear and, as a result, cross sections normal to the datum surface rotate but do not warp. This first order theory can cause shear locking in a finite element formulation because boundary conditions are not satisfied on the top and bottom of the shell. Shear correction factors minimize this condition.

Reddy adds a parabolic transverse shear strain representation to RM theory to eliminate shear locking (20). Palazotto and Dennis, Creaghan, and Miller employ Reddy's theory (3; 13; 19). As such, it is retained here.

Beam and arch theory spawn an enormous amount of research. As with shell theory, arch theory refinements are directed towards improving displacement and rotation limits. As a baseline for comparison, this thesis retains Miller's theory without changes (13). His theory combines total Lagrangian kinematics with the aforementioned trigonometric representation of the bending angle and vector decomposition of the displacement equations. Miller also includes parabolic transverse shear, mid-plane extensibility, and all non-linear in-plane strain terms. A finite element formulation approximates solutions.

Oliver and Oñate present a total Lagrangian finite element formulation which accurately calculates large displacements and rotations (18). They include a parabolic shear representation and mid plane extensibility. Degenerated shell elements similar to SLR theory are used as well as exact trigonometric relations. This is a stress based theory whereas SLR is strain based.

Surana also uses a vector decomposition of the displacement equations (27). Total Lagrangian kinematics and a finite element representation are employed as well.

Noguchi and Hisada use a finite element model to analyze post buckling shell response (17). Their vector based, total Lagrangian kinematics provide a good comparison to the current derivation.

Mondkar and Powell develop equations of motion based on total Lagrangian kinematics (16). They retain higher order terms and consider mid-plane extension and transverse shear. Solutions for isotropic structures are found using FEM.

Huddleston uses a total Lagrangian formulation with no small angle assumptions to analyze deep, isotropic, circular arches (11).

Transverse shear is neglected while mid plane extension is allowed. Huddleston solves simultaneous, non-linear, first-order differential equations rather than using finite element or finite difference methods.

Sabir and Lock also neglect transverse shear and eliminate some higher order mid-plane strain terms (23). Total Lagrangian kinematics are used in conjunction with a finite element solution method to analyze isotropic, circular arches.

DeDeppo and Schmidt present numerous approaches to solving non-linear circular arch problems (5; 6; 24),. When they consider transverse shear, a first order representation is used. When mid plane stretching is considered, higher order terms are neglected. They use total Lagrangian kinematics with no small angle approximations and finite difference solution method.

Epstein and Murray retain quadratic in-plane strain terms while neglecting transverse shear strain (7). Mid-plane extension is considered with higher order terms retained. Epstein and Murray use finite elements and a total Lagrangian approach to examine only isotropic beams.

Belytschko and Glaum use small angle approximations in an updated Lagrangian basis (1). Mid-plane stretch is allowed but higher order terms are eliminated. Also, they ignore transverse shear.

Minguet and Dugundji also use the co-rotational method; however, Euler angles are used for exact rigid body kinematics (15). Transverse shear and mid-plane extensibility are considered without higher order terms.

II. Theory

2.1 Overview

The current work is based on Miller's kinematic development, constitutive relations, strain relations, and beam energy considerations (13). His work is reviewed here for completeness and to highlight certain inaccuracies in the theory. Miller's FEM formulation is modified to include a higher order shape function for the bending angle. Numerical solution methods and arch geometry definitions are discussed as well.

2.2 Kinematics

Miller derives arch displacement equations in a vector format. The first step considers displacements due to pure bending as shown in Figure 2-1. Points j and k are on the outer surface of the beam and

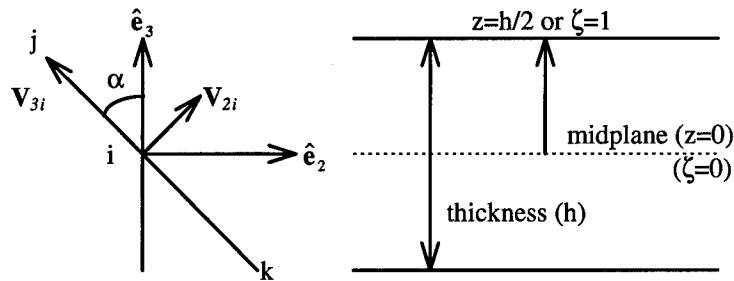


Figure 2-1 Flat Beam Bending Rotation

point i is on the mid plane. The V coordinates are initially parallel to the original system and move with the normal of the cross section during deformation. The \hat{e} coordinates are the global coordinates of the total Lagrangian system. The vector V_{3i} remains parallel to the cross

sectional thickness during bending. The magnitude of V_{3i} equals the beam thickness (h). V_{3i} remains perpendicular to V_{2i} during pure bending and α is the rotation angle about V_{1i} . The natural coordinate ζ is aligned in the \hat{e}_3 direction with magnitude 1 at the top of the beam (positive \hat{e}_3 direction) and -1 at the bottom. V_{3i} is expressed in global coordinates as:

$$V_{3i} = (y_i - y_k)e_2 - (z_j - z_k)e_3 \quad (2-1)$$

with direction cosines:

$$\begin{bmatrix} m_{3i} \\ n_{3i} \end{bmatrix} = \frac{1}{h} \begin{bmatrix} y_j - y_k \\ z_j - z_k \end{bmatrix} \quad (2-2)$$

Eqn(2-2) is substituted into Eqn(2-1), yielding:

$$V_{3i}^n = h(m_{3i}^n e_2 + n_{3i}^n e_3) \quad (2-3)$$

The n superscript denotes any deformed state. The position of any point P on the cross section is described by its location from the beam center line and the location of point i on the mid plane as

$$\begin{bmatrix} y \\ z \end{bmatrix}^n = \begin{bmatrix} y_i \\ z_i \end{bmatrix} + \frac{\zeta}{2} V_{3i}^n \quad (2-4)$$

The total Lagrangian displacement of any point P is:

$$\bar{u}(z) = \begin{bmatrix} y_i \\ z_i \end{bmatrix}^n - \begin{bmatrix} y_i \\ z_i \end{bmatrix}^0 + \frac{\zeta}{2} [V_{3i}^n - V_{3i}^0] \quad (2-5)$$

The superscript 0 denotes the original, undeformed state. Miller assumes vertical displacement (w) is constant through the thickness. As shown in Figure 2-2, P only displaces horizontally due to bending. This assumption implies stretching of the beam normal. Resulting normal strains are neglected. Horizontal displacement is assumed in order to keep w constant through the thickness. The resulting strains become

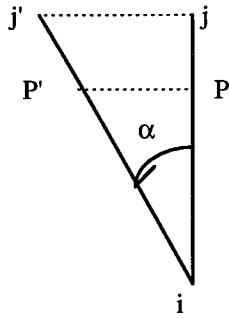


Figure 2-2 Bending Motion

significant as α approaches $\pi/2$. This assumption changes Eqn(2-3) as follows:

$$V_{3i}^n = \frac{h}{\cos(\alpha_i)} \begin{bmatrix} m_{3i} \\ n_{3i} \end{bmatrix} \quad (2-6)$$

where

$$\begin{aligned} m_{3i} &= -\sin(\alpha_i) \\ n_{3i} &= \cos(\alpha_i) \end{aligned}$$

Eqn(2-6) is substituted into Eqn(2-5) at $\alpha_0 = 0$ yielding:

$$\bar{u} = \bar{u}_i + \frac{\zeta}{2} h \left[\frac{1}{\cos(\alpha_i)} \begin{bmatrix} m_{3i} \\ n_{3i} \end{bmatrix}^n - \begin{bmatrix} m_{3i} \\ n_{3i} \end{bmatrix}^0 \right] \quad (2-7)$$

so that

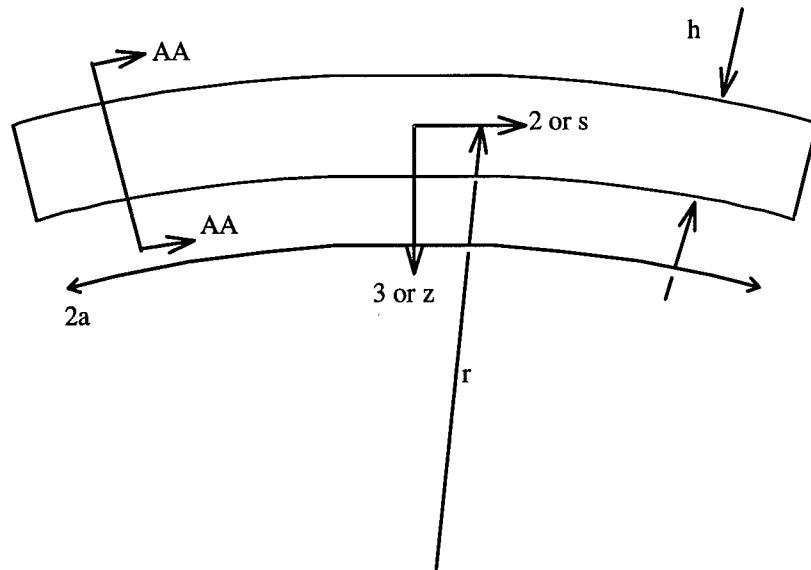
$$V_{3i}^n = \begin{bmatrix} 1 & -\tan(\alpha_i) \\ 0 & 1 \end{bmatrix} V_{3i}^0 = [R] V_{3i}^0$$

where \bar{u}_i represents the motion of point P due to the motion of point i on the midplane and $[R]$ is the bending rotation tensor. The general displacement equation using natural coordinates is

$$\bar{u} = \bar{u}_i + \frac{\zeta}{2} [[R] - [I]] V_{3i}^0 \quad (2-8)$$

where $[I]$ is the 2x2 identity matrix.

Adapting this derivation to curved beams requires consideration of the additional motion due to curvature. Orthogonal curvilinear coordinates are used for arches as shown in figure 2-3. The s direction



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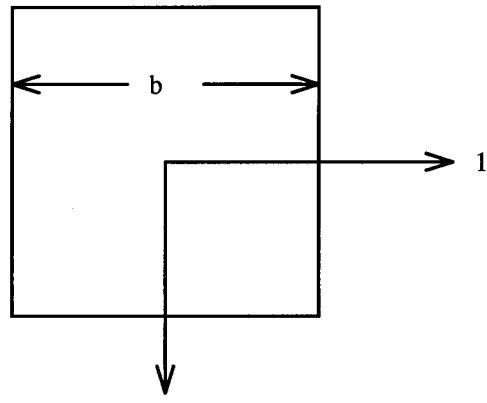


Figure 2-3 Global Arch Coordinates

runs along the arch midplane and the 3 direction points toward the center of curvature.

The \bar{u}_i term in Eqn 2-8 represents the displacement of an arbitrary point caused by non-bending motion. For straight beams, mid-plane motion directly correlates to \bar{u}_i for any arbitrary point. As shown in figure 2-4, the \bar{u}_i becomes thickness dependent in a curvilinear system therefore requiring metric tensor coefficients (19; 22). In this case,

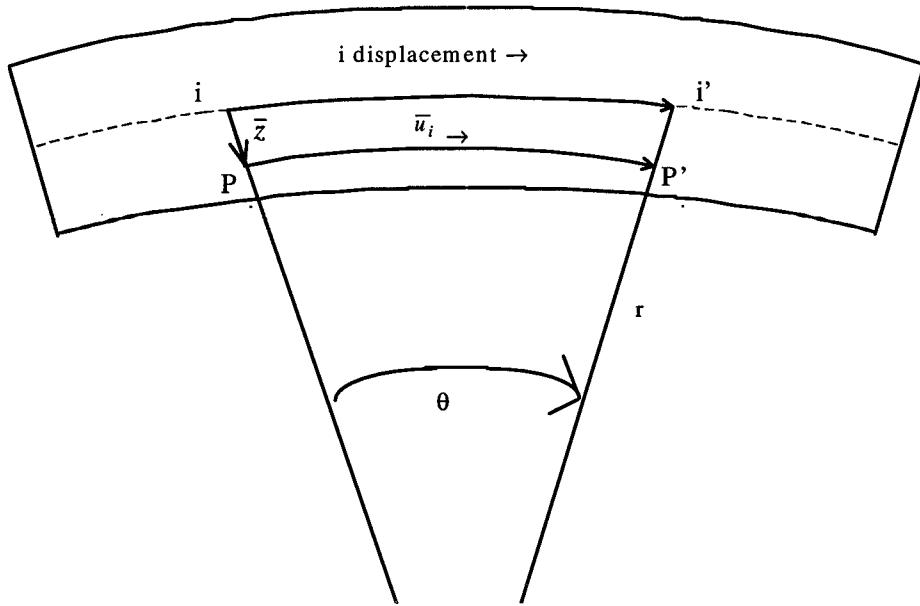


Figure 2-4 Mid Plane Displacement Effects

the motion of point P due to midplane motion is:

$$\bar{u}_i = \begin{bmatrix} v(1 - \frac{z}{r}) \\ w \end{bmatrix} \quad (2-9)$$

where v and w are the motion of point i on the midplane. Using the shell scale factors as did Palazotto and Dennis yields similar results (19).

Figure 2-5 shows the relationship between ψ and slope ($w_{,2}$). A positive ψ is defined as the angle which causes a point with a positive

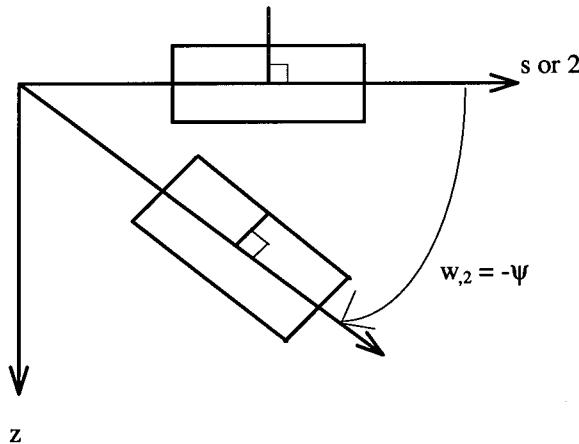


Figure 2-5 Pure Bending Convention

z coordinate to move in the positive s direction. Since α is defined using the right hand rule in a positive direction about the 1 axis, $\psi = -\alpha$.

Expressing Eqn(2-8) and Eqn(2-9) in global coordinates yields these displacement components:

$$u_2 = v \left(1 - \frac{z}{r} \right) + z \tan(\psi) \quad (2-10)$$

$$u_3 = w$$

These are the displacement relationships for bending only. Instead of using a small angle approximation for $\tan(\psi)$, Miller used shell displacement relationships developed by Palazotto and Dennis to retain the tangent function in the u_2 component(13; 19). As a result, the in plane displacement of Eqn(2-10) becomes:

$$u_2(s, z) = v \left(1 - \frac{z}{r} \right) + z \tan(\psi) + z^3 k (\psi + w_{,2}) \quad (2-11)$$

where:

$$k = \frac{4}{3h^2}$$

Similar to ψ , a positive shear angle β causes a point with a positive z coordinate to move in the positive s direction. Figure 2-6 shows the relationship between β and w_{z2} . Mathematically, w_{z2} , β , and ψ are related by:

$$w_{z2} + \psi = -\beta \quad (2-12)$$

Eqn(2-12) is used to find β after w_{z2} and ψ are found using FEM. The sign convention of these three quantities is shown in figure 2-7.

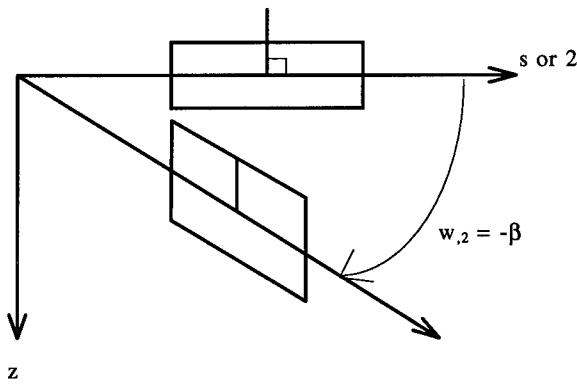


Figure 2-6 Angular Shear convention

The tangent function in Eqn(2-11) is approximated using the following series expansion:

$$\tan(\psi) = \sum_{i=0}^{\infty} -1^{i-1} \frac{2^{2i}(2^{2i}-1)}{2i!} B_{2i} \psi^{2i-1} \quad (2-13)$$

where the B_n 's are Bernoulli numbers and ψ is in radians (12). This series expansion is truncated at two terms yielding:

$$\tan(\psi) = \psi + \frac{1}{3} \psi^3 \quad (2-14)$$

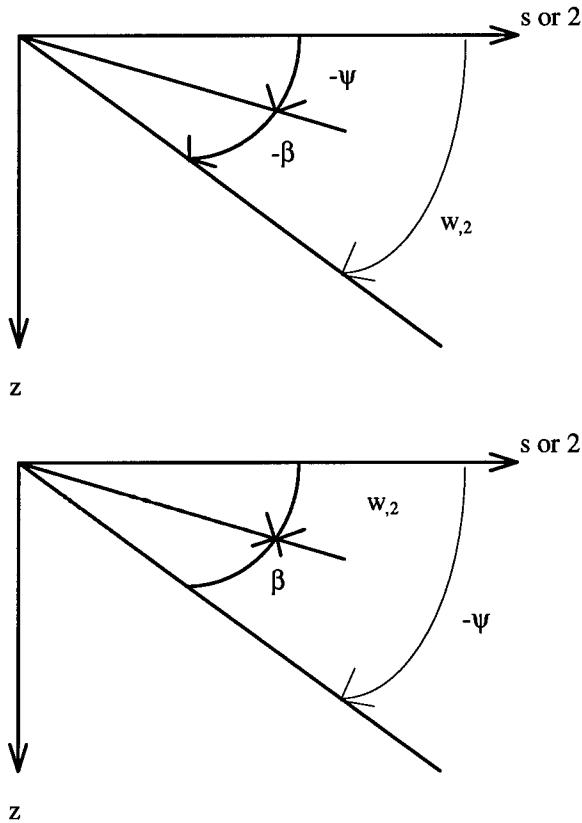


Figure 2-7 Slope, Shear, and Bending Relationship

Substituting Eqn(2-14) into Eqn(2-11) for u_2 , one obtains the final displacement components:

$$u_1 = 0$$

$$u_2(s, z) = v \left(1 - \frac{z}{r} \right) + z(\psi + \frac{1}{3}\psi^3) + z^3 k(\psi + w,2) \quad (2-15)$$

$$u_3(s) = w$$

where v and w are midplane displacements in the 2 and 3 directions, respectively, z is the distance from the midplane, ψ is angle due to bending, and $w,2$ is the slope of the midplane tangent.

2.3 Constitutive Relations

Table 2.1 presents the explicit and contracted notation used for both the stress and strain tensors.

A fiber composite lamina has the local coordinate system defined as shown in figure 2-8. The 1' direction is aligned with the fibers of

Table 2-1 Contraction Definitions

Stress		Strain	
Contracted	Explicit	Contracted	Explicit
σ_1	σ_{11}	ϵ_1	ϵ_{11}
σ_2	σ_{22}	ϵ_2	ϵ_{22}
σ_3	σ_{33}	ϵ_3	ϵ_{33}
σ_4	σ_{23}	ϵ_4	$2\epsilon_{23}$
σ_5	σ_{13}	ϵ_5	$2\epsilon_{13}$
σ_6	σ_{12}	ϵ_6	$2\epsilon_{12}$

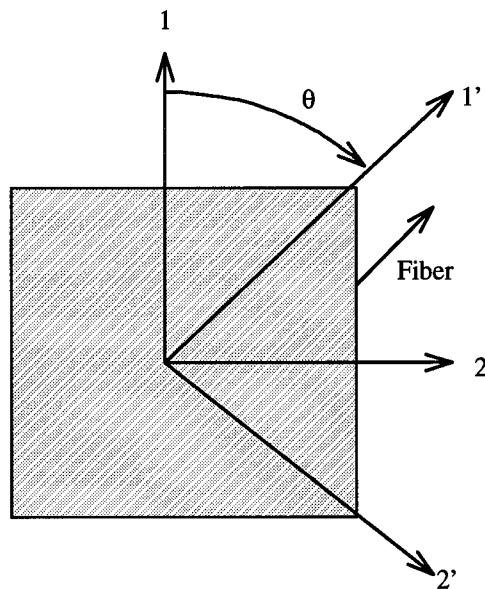


Figure 2-8 Material vs. Global Coordinates

the composite. The 2' and 3' directions are the transverse direction.

In this work, the 3 and 3' directions are aligned and point into the page.

Following these conventions, the stress and strain for a composite lamina are:

$$\begin{bmatrix} \sigma'_1 \\ \sigma'_2 \\ \sigma'_3 \\ \sigma'_4 \\ \sigma'_5 \end{bmatrix} = \begin{bmatrix} Q_{11} & Q_{12} & 0 & 0 & 0 \\ Q_{12} & Q_{22} & 0 & 0 & 0 \\ 0 & 0 & Q_{66} & 0 & 0 \\ 0 & 0 & 0 & Q_{44} & 0 \\ 0 & 0 & 0 & 0 & Q_{55} \end{bmatrix} \begin{bmatrix} \epsilon'_1 \\ \epsilon'_2 \\ \epsilon'_3 \\ \epsilon'_4 \\ \epsilon'_5 \end{bmatrix} \quad (2-16)$$

The Q_{ij} 's represent the ply stiffness expressed in material coordinates, the σ'_i 's are ply stresses, and the ϵ'_i 's are the strains.

These relations assume plane stress conditions exist, therefore σ_3 is negligible. Through the thickness strain (ϵ_3) is found through constitutive relations as shown by Palazotto and Dennis (19). The Q_{ij} 's in terms of engineering constants are:

$$\begin{aligned} Q_{11} &= \frac{E_1}{1-\nu_{12}\nu_{21}} & Q_{12} &= \frac{\nu_{12}E_2}{1-\nu_{12}\nu_{21}} = \frac{\nu_{21}E_1}{1-\nu_{12}\nu_{21}} \\ Q_{22} &= \frac{E_2}{1-\nu_{12}\nu_{21}} & Q_{44} &= G_{23} & Q_{55} &= G_{13} & Q_{66} &= G_{12} \end{aligned} \quad (2-17)$$

In these relations, E_i 's are Young's moduli, G_{ij} 's are shear moduli, and ν_{ij} 's are Poisson's ratios. All values are expressed in material coordinates.

When constructing a laminate composite, the stiffness (Q_{ij}) of each lamina must be converted to global coordinates. This transformation is accomplished using the angle θ defined in figure 2-8 and the transformation equations found in Reddy (20). These equations are presented in Appendix A and give the following constitutive relations for the k th ply:

$$\begin{Bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_6 \\ \sigma_4 \\ \sigma_5 \end{Bmatrix}_k = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} & 0 & 0 \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{26} & 0 & 0 \\ \bar{Q}_{16} & \bar{Q}_{26} & \bar{Q}_{66} & 0 & 0 \\ 0 & 0 & 0 & \bar{Q}_{44} & 0 \\ 0 & 0 & 0 & 0 & \bar{Q}_{55} \end{bmatrix}_k \begin{Bmatrix} \epsilon_1 \\ \epsilon_2 \\ \epsilon_6 \\ \epsilon_4 \\ \epsilon_5 \end{Bmatrix}_k \quad (2-18)$$

where \bar{Q}_{ij} 's represent the plane stress stiffnesses in global coordinates.

The next developmental step requires simplifying the 2-D Eqn(2-18) to 1-D. Because this work examines beams and arches which are narrow in the 1 direction, normal stresses in this direction σ_1 are negligible. Also, because the beam/arch is narrow and traction free at the sides, the in plane shear stress σ_6 is neglected. Stress and strain in the 5 direction (σ_5 and ϵ_5) are neglected because beam twisting is not considered. These assumptions reduce Eqn(2-18) to the following 1-D constitutive relations (3):

$$\begin{aligned} \sigma_{4k} &= \bar{Q}_{44k} \epsilon_{4k} \\ \sigma_{2k} &= \hat{Q}_{2k} \epsilon_{2k} \end{aligned}$$

where (2-19)

$$\hat{Q}_{2k} = \left(\bar{Q}_{22} - \frac{\bar{Q}_{12}^2}{\bar{Q}_{11}} \right)_k$$

Because twisted and torsional stiffness are not considered, Eqn(2-19) is limited to balanced, symmetric lay-ups. This limitation drives \bar{Q}_{16} to zero for the entire laminate.

2.4 Strain Relations

Miller and Creaghan both employ the following relationship between Green strain and shell physical strains developed by Palazotto and Dennis (19).

$$\epsilon_{ij} = \frac{\gamma_{ij}}{h_i h_j} \quad (2-20)$$

In Eqn(2-23), the γ_{ij} 's are Green strain tensor components and the h_i 's are shell scale factors. The previously discussed thickness dependence of \bar{u}_i drives the requirement for shell scale factors. Because only in plane normal and through the thickness shear strains are considered, only the two Green strain tensor terms related to these strains are required.

$$\begin{aligned} \gamma_{22} = & h_2 u_{2,2} + \frac{h_2 u_3}{h_3} h_{2,3} + \frac{h_2 u_1}{h_1} h_{2,1} \\ & + \frac{1}{2} \left(u_{2,2} + \frac{u_3}{h_3} h_{2,3} + \frac{u_1}{h_1} h_{2,1} \right)^2 \\ & + \frac{1}{2} \left(u_{3,2} - \frac{u_2}{h_3} h_{2,3} \right)^2 + \frac{1}{2} \left(u_{1,2} - \frac{u_2}{h_1} h_{2,1} \right)^2 \end{aligned} \quad (2-21)$$

$$\begin{aligned} \gamma_{23} = & \frac{1}{2} (h_3 u_{3,2} + h_2 u_{2,3} - u_2 h_{2,3} - u_3 h_{3,2}) \\ & + \frac{1}{2} \left(u_{2,3} - \frac{u_3}{h_2} h_{2,3} \right) \left(u_{2,2} + \frac{u_3}{h_3} h_{2,3} + \frac{u_1}{h_1} h_{2,1} \right) \\ & + \frac{1}{2} \left(u_{3,2} - \frac{u_2}{h_3} h_{2,3} \right) \left(u_{3,3} + \frac{u_2}{h_2} h_{3,2} + \frac{u_1}{h_1} h_{3,1} \right) \\ & + \frac{1}{2} \left(u_{1,2} - \frac{u_2}{h_1} h_{2,1} \right) \left(u_{1,3} - \frac{u_2}{h_1} h_{3,1} \right) \end{aligned} \quad (2-22)$$

Here, the \bar{u}_i 's come from Eqn(2-15).

The scale factors for a cylinder with curvature in only the 2 or s direction are:

$$\begin{aligned} h_1 &= h_3 = 1 \\ h_2 &= 1 - \frac{z}{r} \end{aligned} \quad (2-23)$$

Since beam width does not affect these scale factors, they also apply to a curved beam or arch. These factors match Eqn(2-9), which were developed graphically from figure 2-4.

Inserting Eqn(2-15), Eqn(2-21), and Eqn(2-23) into Eqn(2-20) yields the in plane normal strain ϵ_2 terms. All terms of Eqn(2-21) are retained because in-plane strain dominates thin beam bending problems. Representing the scale factors with truncated Taylor series simplifies expansion of Eqn(2-20). The resulting sixty expressions are shown in Palazotto and Dennis (19). For this work, only the following nine of the sixty scale factor approximations are required:

$$\begin{aligned} \frac{1}{h_2} &\approx 1 + zc & \frac{h_{2,3}}{h_2 h_3} &\approx -c - c^2 z & \frac{1}{h_2^2} &\approx 1 + 2zc \\ \frac{h_{2,3}}{h_2^2 h_3} &\approx -c - 2c^2 z & \frac{h_{2,3}^2}{h_2 h_3} &\approx c^2 + c^3 z & \frac{h_{2,3}^2}{h_2^2} &\approx c^2 + 2c^3 z \\ h_{2,3}^2 &\approx c^2 & \frac{h_{2,3}^2}{h_2} &\approx c^2 + c^3 z & \frac{h_{2,3}}{h_2^2} &\approx -c - 2c^2 z \end{aligned} \quad (2-24)$$

where $c = 1/r$. In plane normal strain is expressed in terms of the mid plane strain (ϵ_2^0) and functions independent of Z (χ_{2i}):

$$\epsilon_2 = \epsilon_2^0 + \sum_{p=1}^7 z^p \chi_{2p} \quad (2-25)$$

The mid plane strain and χ_{2i} 's are defined as:

$$\begin{aligned}
\epsilon_2^0 &= v_{,2} - wc + 0.5(v_{,2}^2 + w_{,2}^2 + v^2 c^2 + w^2 c^2) + vw_{,2}c - v_{,2}wc \\
\chi_{21} &= \psi_{,2} - wc^2 + w_{,2}^2 c + w^2 c^3 - c^2(v_{,2}w - vw_{,2}) + v\psi c^2 + v_{,2}\psi_{,2} \\
&\quad - c(\psi_{,2}w - \psi w_{,2}) + \psi_{,2}\psi^2 + v_{,2}\psi_{,2}\psi^2 - wc\psi_{,2}\psi^2 \\
\chi_{22} &= \psi_{,2}c + 0.5(\psi_{,2}^2 + \psi^2 c^2) + v\psi c^3 - 2c^2(\psi_{,2}w - \psi w_{,2}) + \psi_{,2}v_{,2}c - 1.5(c^4 v^2 + c^2 v_{,2}^2) \\
&\quad - c^2 v_{,2} + 2c^3(v_{,2}w - vw_{,2}) + c(\psi_{,2}\psi^2 + \psi_{,2}\psi^2 v_{,2}) + \psi_{,2}\psi^2 \\
\chi_{23} &= k(w_{,22} + \psi_{,2}) + c\psi_{,2}^2 + \psi^2 c^3 + kv_{,2}(w_{,22} + \psi_{,2}) + vkc^2(w_{,2} + \psi) - wkc(w_{,22} + \psi_{,2}) \\
&\quad + w_{,2}kc(w_{,2} + \psi) + c^5 v^2 + c^3 v_{,2}^2 - 2c^2(c^2 v\psi + v_{,2}\psi_{,2}) \\
\chi_{24} &= kc(w_{,22} + \psi_{,2}) + vkc^3(w_{,2} + \psi) + 2kc^2(-ww_{,22} - w_{,2}\psi + w_{,2}^2 + w\psi_{,2}) \\
&\quad + k\psi_{,2}(w_{,22} + \psi_{,2}) + \psi kc^2(w_{,2} + \psi) + v_{,2}kc(w_{,22} + \psi_{,2}) + k\psi_{,2}\psi^2(1 + w_{,22}) \\
\chi_{25} &= 2kc[\psi_{,2}(w_{,22} + \psi_{,2}) + \psi c^2(w_{,2} + \psi) - cv_{,2}(w_{,22} + \psi_{,2}) - c^3 v(w_{,2} + \psi) + \psi_{,2}\psi^2 w_{,22}] \\
\chi_{26} &= \frac{k^2}{2}[w_{,22}^2 + 2w_{,22}\psi_{,2} + \psi_{,2}^2 + c^2(w_{,2}^2 + 2w_{,2}\psi + \psi^2)] \\
\chi_{27} &= k^2 c[(w_{,22} + \psi_{,2})^2 + c^2(w_{,2} + \psi)^2]
\end{aligned} \tag{2-26}$$

In these equations, terms with combinations of c , ψ , $\psi_{,2}$, and $w_{,2}$ of order 5 and higher are neglected as higher order terms.

Because in-plane stress and strain dominate thin beam bending, through the thickness shear strain (ϵ_4) is developed using only the linear terms of Eqn(2-22). Combining these linear terms with Eqn(20) and Eqn(23) yields the following expression for through the thickness shear strain:

$$\epsilon_4 = 2\epsilon_{23} = \frac{1}{1 - \frac{z}{r}} \left[u_{3,2} + (1 - \frac{z}{r})u_{2,3} + \frac{u_2}{r} \right] \tag{2-27}$$

Small angle approximations are used in the global displacement equations because shear strain is linear, resulting in:

$$\begin{aligned}
u_1 &= 0 \\
u_2(s, z) &= v \left(1 - \frac{z}{r} \right) + z(\psi) + z^3 k(\psi + w_{,2}) \\
u_3(s) &= w
\end{aligned} \tag{2-28}$$

Substituting Eqn(2-28) into Eqn(2-27) using $k = 4/(3h^2)$, one obtains:

$$\epsilon_4 = \frac{1}{1 - \frac{z}{r}} \left[w_{,2} + \psi - \frac{4z^2}{h^2}(\psi + w_{,2}) + \frac{8z^3}{3h^2 r}(\psi + w_{,2}) \right] \tag{2-29}$$

The scale factor term in front of Eqn(2-29) is approximated with a truncated binomial expansion:

$$\frac{1}{1-\frac{z}{r}} = 1 + \frac{z}{r} + \frac{z^2}{r^2 \left(1-\frac{z}{r}\right)} + \dots \approx 1 \quad (2-30)$$

The z^3 is also assumed negligible since $z \leq h/2$, leaving:

$$\varepsilon_4 = \varepsilon_4^0 + z^2 \chi_{42}$$

where (2-31)

$$\varepsilon_4^0 = w_{,2} + \psi \text{ and } \chi_{42} = \frac{-4}{h^2} (w_{,2} + \psi)$$

as the expressions for through the thickness shear strain.

2.5 Beam Potential Energy

The potential energy Π_p is defined as the sum of the internal strain energy (U) and the external work (V):

$$\Pi_p = U + V \quad (2-32)$$

The internal strain energy is defined as:

$$U = \int_{vol} W^* dV \quad (2-33)$$

where W^* is the strain energy density function:

$$W^* = \frac{1}{2} a_{ijkl} \varepsilon_{ij} \varepsilon_{kl} \quad (2-34)$$

The a_{ijkl} term is an elasticity tensor.

In this case, the internal strain energy is made up of two components, the in-plane strain (U_1) and the through the thickness shear strain (U_2):

$$U_1 = \frac{1}{2} b \int_l^{\frac{h}{2}} \int_{-\frac{h}{2}}^{\frac{h}{2}} \hat{Q}_{2k} \varepsilon_2^2 dz ds \quad (2-35)$$

$$U_2 = \frac{1}{2} b \int_l^{\frac{h}{2}} \bar{Q}_{44k} \varepsilon_4^2 dz ds \quad (2-36)$$

Using Eqn(2-25) :

$$\varepsilon_2^2 = (\varepsilon_2^0)^2 + 2 \sum_{p=1}^7 z^p \chi_{2p} \varepsilon_2^0 + \sum_{p=1}^7 (z^p \chi_{2p})^2 \quad (2-37)$$

Now U_1 is divided into three parts, each representing a strain component from Eqn(2-37) :

$$U_1 = \frac{1}{2} b \int_l^{\frac{h}{2}} (\mu_1 + \mu_2 + \mu_3) \quad (2-38)$$

where:

$$\begin{aligned} \mu_1 &= \int_{-\frac{h}{2}}^{\frac{h}{2}} \hat{Q}_{2k} (\varepsilon_2^0)^2 dz = A (\varepsilon_2^0)^2 \\ \mu_2 &= \int_{-\frac{h}{2}}^{\frac{h}{2}} 2 \hat{Q}_{2k} (\varepsilon_2^0) \sum_{p=1}^7 \chi_{2p} z^p dz = 2 \varepsilon_2^0 B \chi_{21} + D \chi_{22} + E \chi_{23} + F \chi_{24} + G \chi_{25} + H \chi_{26} + I \chi_{27} \\ \mu_3 &= \int_{-\frac{h}{2}}^{\frac{h}{2}} \hat{Q}_{2k} \left(\sum_{p=1}^7 \chi_{2p} z^p \right)^2 dz \\ &= D \chi_{21}^2 + 2E \chi_{21} \chi_{22} + F(2 \chi_{21} \chi_{23} + \chi_{22}^2) + 2G(\chi_{21} \chi_{24} + \chi_{22} \chi_{23}) + 2H(\chi_{21} \chi_{25} + \chi_{22} \chi_{24}) \quad (2-39) \\ &\quad + 2I(\chi_{21} \chi_{26} + \chi_{22} \chi_{25} + \chi_{23} \chi_{24}) + J(2 \chi_{21} \chi_{27} + 2 \chi_{22} \chi_{26} + 2 \chi_{23} \chi_{25} + \chi_{24}^2) \\ &\quad + 2K(\chi_{22} \chi_{27} + \chi_{23} \chi_{26} + \chi_{24} \chi_{25}) + L(2 \chi_{23} \chi_{27} + 2 \chi_{24} \chi_{26} + \chi_{25}^2) + 2P(\chi_{24} \chi_{27} + \chi_{25} \chi_{26}) \\ &\quad + R(2 \chi_{25} \chi_{27} + \chi_{26}^2) + 2S \chi_{26} \chi_{27} + T \chi_{27}^2 \end{aligned}$$

The μ_i 's of Eqn(2-39) have been integrated through the thickness such that:

$$(A, B, D, E, F, G, H, I, J, K, L, P, R, S, T) = \int_{-\frac{h}{2}}^{\frac{h}{2}} \hat{Q}_{2k} (1, z, z^2, z^3, z^4, z^5, z^6, z^7, z^8, z^9, z^{10}, z^{11}, z^{12}, z^{13}, z^{14}) dz \quad (2-40)$$

All other terms of Eqn(2-39) are independent of thickness. If composite materials are used, the balanced symmetric lay up drives the integration of any odd powers of z to zero.

Strain energy due to through the thickness shear is similarly derived. In this case:

$$U_2 = \frac{1}{2} b \int_l^h \mu_s ds$$

where:

$$\begin{aligned} \mu_s &= \int_{-\frac{h}{2}}^{\frac{h}{2}} \bar{Q}_{44k} [(\varepsilon_4^0)^2 + 2\varepsilon_4^0 \chi_{42} z^2 + (\chi_{42})^2 z^4] dz \\ &= AS(\varepsilon_4^0)^2 + 2DS\varepsilon_4^0 \chi_{42} + FS(\chi_{42})^2 \end{aligned}$$

Again, the elasticity terms (AS,DS,FS) are found by integrating through the thickness:

$$[AS, DS, FS] = \int_{-\frac{h}{2}}^{\frac{h}{2}} \bar{Q}_{44k} [1, z^2, z^4] dz \quad (2-42)$$

The strain expressions of Eqn(2-25) and Eqn(2-31) are divided into linear and nonlinear parts to facilitate placing them into the preceding strain energy formulations. This is done by first defining the displacement gradient vector \mathbf{d} such that:

$$\mathbf{d}^T = \{ v \quad v_{,2} \quad w \quad w_{,2} \quad w_{,22} \quad \psi \quad \psi_{,2} \} \quad (2-43)$$

Now the in plane strain can be expressed as:

$$\begin{aligned} \varepsilon_2^0 &= \mathbf{L}_0^T \mathbf{d} + \frac{1}{2} \mathbf{d}^T \mathbf{H}_0 \mathbf{d} \\ \chi_{2p} &= \mathbf{L}_p^T \mathbf{d} + \frac{1}{2} \mathbf{d}^T \mathbf{H}_p \mathbf{d} \end{aligned} \quad (2-44)$$

where $p = 1,2,3,4,5,6,7$. The \mathbf{L}_i 's are column vectors and the \mathbf{H}_i 's are symmetric matrices. Combining the \mathbf{L}_i 's and \mathbf{H}_i 's with the displacement gradient vector creates the linear and nonlinear strain terms, respectively. This is demonstrated for the mid plane strain term (ε_2^0):

$$\begin{aligned}
\boldsymbol{\varepsilon}_2^0 = & \{ 0 \quad 1 \quad -c \quad 0 \quad 0 \quad 0 \quad 0 \} \begin{Bmatrix} v \\ v_{,2} \\ w \\ w_{,2} \\ w_{,22} \\ \Psi \\ \Psi_{,2} \end{Bmatrix} \\
& + \frac{1}{2} \{ v \quad v_{,2} \quad w \quad w_{,2} \quad w_{,22} \quad \Psi \quad \Psi_{,2} \} \begin{Bmatrix} c^2 & 0 & 0 & c & 0 & 0 & 0 \\ 0 & 1 & -c & 0 & 0 & 0 & 0 \\ 0 & -c & c^2 & 0 & 0 & 0 & 0 \\ c & 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{Bmatrix} \begin{Bmatrix} v \\ v_{,2} \\ w \\ w_{,2} \\ w_{,22} \\ \Psi \\ \Psi_{,2} \end{Bmatrix}
\end{aligned} \tag{2-45}$$

The through the thickness shear strain (ε_4) expressions are less complicated due to the previous assumptions about linearity and small angle approximations:

$$\begin{aligned}
\boldsymbol{\varepsilon}_4^0 &= \mathbf{S}_0^T \mathbf{d} \\
\chi_{42} &= \mathbf{S}_2^T \mathbf{d}
\end{aligned} \tag{2-46}$$

Appendix A contains the complete \mathbf{H} , \mathbf{L} , and \mathbf{S} matrices.

Miller's derivation of the \mathbf{H}_i matrices contains non linear terms of Ψ whereas Creaghan's \mathbf{H}_i 's contained only constants. Miller treats the \mathbf{H}_i 's as constants to maintain similarity with Creaghan and Palazotto and Dennis. This linearization is accomplished by substituting the previous incremental value of Ψ into the current \mathbf{H}_i 's (3; 13; 19). Although this assumption simplifies the impending variational calculus, it limits the accuracy of Miller's theory because the behavior of the bending angle is not properly modeled and leads to inexact equilibrium equations. This may be a major reason the theory is limited to moderate rotations (46 degrees). Subsequent research should investigate this assumption.

The total beam strain energy can now be expressed as:

$$U = \frac{1}{2} b \int_I \mathbf{d}^T \left[\hat{\mathbf{K}} + \frac{\hat{\mathbf{N}}_1(\mathbf{d})}{3} + \frac{\hat{\mathbf{N}}_2(\mathbf{d})}{6} \right] \mathbf{d} ds \quad (2-47)$$

where $\hat{\mathbf{K}}$ is a matrix of constants, $\hat{\mathbf{N}}_1$ is a matrix of the linear functions of \mathbf{d} , and $\hat{\mathbf{N}}_2$ is a matrix containing quadratic terms from \mathbf{d} . This expression is formed by substituting Eqns(2-44) and (2-46) into Eqns(2-38), (2-39), (2-41), and (2-42) as outlined by Palazotto and Dennis (18). This form of the equation simplifies finite element formulation. As previously stated, the higher order terms of ψ are treated as constants in $\hat{\mathbf{N}}_1$ and $\hat{\mathbf{N}}_2$. The $\hat{\mathbf{K}}$, $\hat{\mathbf{N}}_1$, and $\hat{\mathbf{N}}_2$ matrices are listed in Appendix A.

2.6 Finite Element Formulation

The beam finite element used for the current formulation is shown in figure 2-9. The element has three nodes with degrees of freedom (DOF) v , w , w_2 , ψ , and ψ_2 at the end nodes and only v at the middle node. This element is similar to Miller's except for the addition of the ψ_2 degree of freedom at the end nodes.

The additional DOF facilitates Hermitian shape functions for the bending angle (ψ). This element maintains C^1 continuity for w and ψ and their derivatives and C^0 continuity for v . Previous efforts only maintained C^0 continuity for ψ .

Adding two DOF also mandates additional boundary conditions. This element experiences clamped, hinged, free, and symmetric boundary conditions and ψ_2 must be defined for each. Because a physical

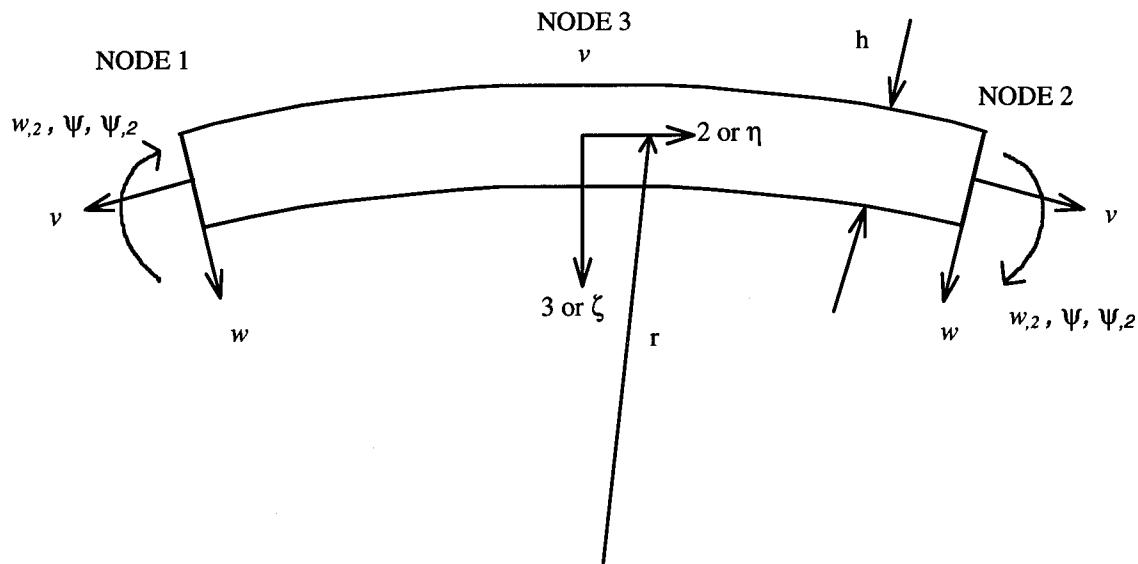


Figure 2-9 Beam Finite Element

interpretation of Ψ_2 is impossible, the values for Ψ_2 under these conditions are derived mathematically from:

$$\Psi_2 \approx \frac{\partial^2 w}{\partial s^2} \approx \frac{M}{EI} \quad (2-48)$$

Based on this assumption, $\Psi_2 = 0$ at any boundary condition incapable of supporting a bending moment. Table 2-2 lists the resulting boundary conditions used for the DOF at nodes 1 and 2. For the current work, v at node 3 is never specified.

Table 2-2 Specified Boundary Conditions

	v	w	w ₂	Ψ	Ψ_2
clamped	0	0	0	0	free
hinged	0	0	free	free	0
free	free	free	free	free	0
point of symmetry	0	0	0	0	free

The values of the DOF $\mathbf{q}(\eta)$ at any point can be expressed in natural coordinates in terms of the element's DOF using:

$$q(\eta) = \begin{bmatrix} v \\ \Psi \\ \Psi_{,2} \\ w \end{bmatrix} = \begin{bmatrix} Q_1 & 0 & 0 & 0 & 0 & Q_3 & Q_2 & 0 & 0 & 0 & 0 \\ 0 & H_{11} & H_{12} & 0 & 0 & 0 & 0 & H_{21} & H_{22} & 0 & 0 \\ 0 & H_{11,\eta} & H_{12,\eta} & 0 & 0 & 0 & 0 & H_{21,\eta} & H_{22,\eta} & 0 & 0 \\ 0 & 0 & 0 & H_{11} & H_{12} & 0 & 0 & 0 & 0 & H_{21} & H_{22} \\ 0 & 0 & 0 & H_{11,\eta} & H_{12,\eta} & 0 & 0 & 0 & 0 & H_{21,\eta} & H_{22,\eta} \end{bmatrix} \begin{bmatrix} v^{(1)} \\ \Psi^{(1)} \\ \Psi_{,2}^{(1)} \\ w^{(1)} \\ w_{,2}^{(1)} \\ v^{(3)} \\ \Psi^{(2)} \\ \Psi_{,2}^{(2)} \\ w^{(2)} \\ w_{,2}^{(2)} \end{bmatrix} \quad (2-49)$$

where the shape functions are:

$$\begin{aligned} Q_1 &= \frac{1}{2}(\eta^2 - \eta) & Q_2 &= \frac{1}{2}(\eta^2 + \eta) & Q_3 &= (1 - \eta^2) \\ H_{11} &= \frac{1}{4}(2 - 3\eta + \eta^3) & H_{12} &= \frac{a}{4}(1 - \eta - \eta^2 + \eta^3) \\ H_{21} &= \frac{1}{4}(2 + 3\eta - \eta^3) & H_{22} &= \frac{a}{4}(-1 - \eta + \eta^2 + \eta^3) \end{aligned} \quad (2-50)$$

The displacement gradient vector \mathbf{d} of Eqn(2-43) must be related to the DOF before using the beam energy equation, Eqn(2-47). The conversion is done in natural coordinates using the following equation:

$$\mathbf{d}(\eta) = \mathbf{Dd} = \begin{bmatrix} Q_1 & 0 & 0 & 0 & 0 & Q_3 & Q_2 & 0 & 0 & 0 & 0 \\ Q_{1,\eta} & 0 & 0 & 0 & 0 & Q_{3,\eta} & Q_{2,\eta} & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & H_{11} & H_{12} & 0 & 0 & 0 & 0 & H_{21} & H_{22} \\ 0 & 0 & 0 & H_{11,\eta} & H_{12,\eta} & 0 & 0 & 0 & 0 & H_{21,\eta} & H_{22,\eta} \\ 0 & 0 & 0 & H_{11,\eta\eta} & H_{12,\eta\eta} & 0 & 0 & 0 & 0 & H_{21,\eta\eta} & H_{22,\eta\eta} \\ 0 & H_{11} & H_{12} & 0 & 0 & 0 & 0 & H_{21} & H_{22} & 0 & 0 \\ 0 & H_{11,\eta} & H_{12,\eta} & 0 & 0 & 0 & 0 & H_{21,\eta} & H_{22,\eta} & 0 & 0 \end{bmatrix} \begin{bmatrix} v^{(1)} \\ \Psi^{(1)} \\ \Psi_{,2}^{(1)} \\ w^{(1)} \\ w_{,2}^{(1)} \\ v^{(3)} \\ \Psi^{(2)} \\ \Psi_{,2}^{(2)} \\ w^{(2)} \\ w_{,2}^{(2)} \end{bmatrix} \quad (2-51)$$

where \mathbf{D} is the shape function matrix used to relate the \mathbf{d} to the DOF.

By multiplying by the inverse of the Jacobian matrix (\mathbf{J}), Eqn(2-51) is converted to global coordinates.

$$\mathbf{d}(\mathbf{s}) = \mathbf{J}^{-1} \mathbf{d}(\mathbf{h}) = \mathbf{J}^{-1} \mathbf{D} \mathbf{q} = \mathbf{T} \mathbf{q} \quad (2-52)$$

where \mathbf{J}^{-1} is defined as:

$$\mathbf{J}^{-1} = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & \frac{1}{a} & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{a} & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{a^2} & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & \frac{1}{a} \end{bmatrix} \quad (2-53)$$

where a is half the element length.

Now the beam strain energy from Eqn(2-47) based on the DOF is:

$$U = \frac{1}{2} b \int_l \mathbf{q}^T \mathbf{T}^T \left[\hat{\mathbf{K}} + \frac{\hat{\mathbf{N}}_2}{3} + \frac{\hat{\mathbf{N}}_2}{6} \right] \mathbf{T} \mathbf{q} ds \quad (2-54)$$

Integrating along the beam length yields new terms used for a tangent stiffness matrix:

$$\mathbf{K} = b \int_l \mathbf{T}^T \hat{\mathbf{K}} \mathbf{T} ds \cdot \mathbf{N}_1 = b \int_l \mathbf{T}^T \hat{\mathbf{N}}_1 \mathbf{T} ds \cdot \mathbf{N}_2 = b \int_l \mathbf{T}^T \hat{\mathbf{N}}_2 \mathbf{T} ds \quad (2-55)$$

Substituting Eqn(2-55) into Eqn(2-54), one obtains this element's potential energy equation:

$$\Pi_p = \frac{1}{2} \mathbf{q}^T \left[\mathbf{K} + \frac{\mathbf{N}_1(\mathbf{q})}{3} + \frac{\mathbf{N}_2(\mathbf{q})}{6} \right] \mathbf{q} - \mathbf{q}^T \mathbf{R} \quad (2-56)$$

where \mathbf{R} is a vector representation of external forces.

Defining the beam equilibrium equation requires taking the first variation of Eqn(2-56) and setting it equal to zero:

$$\delta\Pi_p = \delta\mathbf{q}^T \left[\left[\mathbf{K} + \frac{\mathbf{N}_1(\mathbf{q})}{2} + \frac{\mathbf{N}_2(\mathbf{q})}{3} \right] \mathbf{q} - \mathbf{R} \right] = \delta\mathbf{q}^T [\mathbf{F}(\mathbf{q})] = 0 \quad (2-57)$$

where \mathbf{F} is the resultant equilibrium equations. The first variation is accomplished with respect to the vertical displacement (w) because the bending angle ψ is taken to be a linearized function and the membrane displacement (v) is a linear function. This is where the theory goes wrong. The first variation should be accomplished with respect to the bending angle which includes a third order approximation as well as the vertical displacement. The linearization of ψ limits the exactness of the equilibrium equations as discussed in section 2.5.

Because $\delta\mathbf{q}$ is arbitrary and independent, $\mathbf{F}(\mathbf{q}) = 0$ becomes the set of algebraic nonlinear equilibrium equations. To solve these equations, Eqn(2-57) is expanded in a truncated Taylor series such that:

$$\mathbf{F}(\mathbf{q} + \delta\mathbf{q}) \approx \mathbf{F}(\mathbf{q}) + \frac{\partial \mathbf{F}}{\partial \mathbf{q}} \delta\mathbf{q} = 0$$

or:

$$\frac{\partial \mathbf{F}}{\partial \mathbf{q}} \delta\mathbf{q} = -\mathbf{F}(\mathbf{q})$$

Substituting the expression for \mathbf{F} and its derivatives into Eqn(2-58) yields:

$$\mathbf{K}_T \delta\mathbf{q} = - \left[\mathbf{K} + \frac{\mathbf{N}_1}{2} + \frac{\mathbf{N}_2}{3} \right] \mathbf{q} + \mathbf{R}$$

where:

$$\mathbf{K}_T = \left[\mathbf{K} + \mathbf{N}_1 + \frac{\mathbf{N}_2}{3} \right]$$

Eqn(2-59) is the final equation used to find the equilibrium values for the 11 DOF. In any case, the nonlinear stiffness matrices vary with load and displacement throughout the structure.

2.7 Numerical Solution Methods

Modifying Eqn(2-59) to include all elements of a finite element model, one obtains:

$$\sum_{j=1}^n \left[b \int_I \mathbf{T}^T \left(\hat{\mathbf{K}} + \hat{\mathbf{N}}_1 + \hat{\mathbf{N}}_2 \right) \mathbf{T} ds \right] \delta \mathbf{q} = - \sum_{j=1}^n \left[b \int_I \mathbf{T}^T \left(\hat{\mathbf{K}} + \frac{\hat{\mathbf{N}}_1}{2} + \frac{\hat{\mathbf{N}}_2}{3} \right) \mathbf{T} ds \right] \mathbf{q} + \mathbf{R} \quad (2-60)$$

where n is the number of elements in the model and j is the integration over the length of any particular element. The global load vector \mathbf{R} contains as many rows as DOF. The \mathbf{q} and $\delta \mathbf{q}$ vectors are global displacement arrays constructed from the elemental displacement vectors.

Eqn(2-60) is integrated using Gauss quadrature. Although exact integration would require 7th order Gauss quadrature, 5th order quadrature is used with negligible difference (8; 13).

Miller employed the Newton-Raphson solution methods which are retained for the current work. Either the displacement control method developed by Dennis and Palazotto or a modified Riks-Wempner technique is used to generate equilibrium paths (13; 19; 27). Both techniques are briefly reviewed here.

The displacement control method iterates load values based on incremental displacements until equilibrium is achieved. The initial displacement as well as each displacement increment are prescribed. This method transverses horizontal limit points similar to point A in figure 2-10; however, vertical limits such as point B cannot be evaluated because no unique solution exists at that displacement. Figure 2-11 graphically depicts the displacement control method. Starting at an equilibrium point, the displacement is incremented and a load determined using:

$$[\mathbf{K} + \mathbf{N}_1(\mathbf{q}_{n-1}) + \mathbf{N}_2(\mathbf{q}_{n-1}^2)] \delta \mathbf{q}_{n-1} = - \left[\mathbf{K} + \frac{\mathbf{N}_1(\mathbf{q}_{n-1})}{2} + \frac{\mathbf{N}_2(\mathbf{q}_{n-1}^2)}{3} \right] \mathbf{q}_{n-1} + \mathbf{R}_1 \quad (2-61)$$

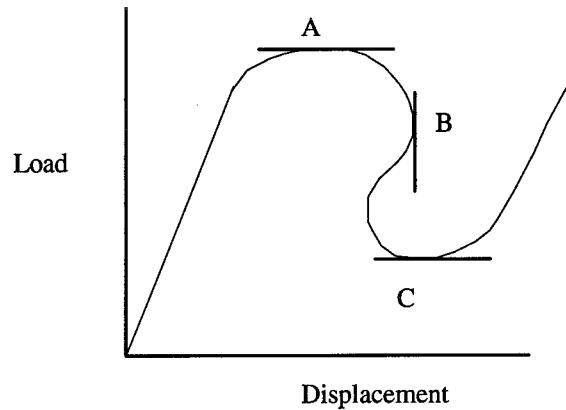


Figure 2-10 Typical Load vs. Displacement Curve

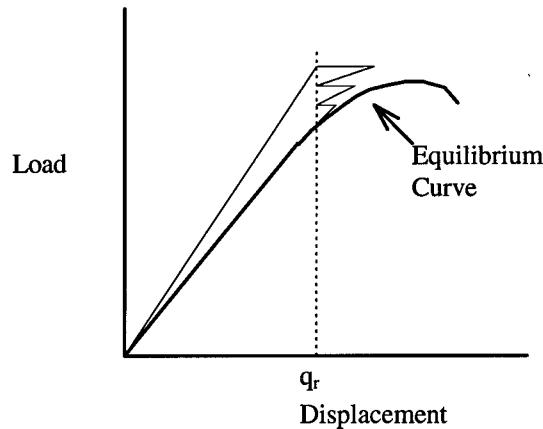


Figure 2-11 Displacement Control

Eqn(2-61) comes from Palazotto and Dennis (19). Iteration of Eqn(2-61) is continued until convergence, for a given displacement increment, satisfies equilibrium

The Tsai and Palazotto modified Riks-Wempner technique transverses limit points the displacement control method cannot (27). This technique, which is referred to as the Riks method, modifies the equilibrium equation such that:

$$F(\mathbf{q}, \lambda) = \left(\mathbf{K} + \frac{\mathbf{N}_1}{2} + \frac{\mathbf{N}_2}{3} \right) \mathbf{q} - \lambda \mathbf{R} \quad (2-62)$$

where λ is a user specified loading parameter. Eqn(2-62) is expanded for a solution similar to Eqn(2-56), resulting in:

$$\mathbf{K}_T \delta \mathbf{q}_i = \delta \Delta_i \mathbf{R} - \mathbf{F}(\mathbf{q}_i, \lambda_i) \quad (2-63)$$

A search radius, Δl is defined as the additional constraint required for Eqn(2-63) to be a solvable system. This value is defined as:

$$\Delta l^2 = \Delta \mathbf{q}_{i+1}^T \Delta \mathbf{q}_{i+1} + \Delta \lambda_{i+1}^2 \mathbf{R}^T \mathbf{R} \quad (2-64)$$

The search radius establishes the distance of the new equilibrium point from the current equilibrium point. Assuming Δl is small enough to accurately capture the equilibrium path, it may be approximated using:

$$\Delta l^2 = \Delta \mathbf{q}_{i+1}^T \Delta \mathbf{q}_{i+1} \quad (2-65)$$

This definition causes an unsymmetrical global stiffness matrix.

Reducing $\delta \mathbf{q}_i$ to two terms restores the required symmetry:

$$\delta \mathbf{q}_i = \delta \mathbf{q}_{i1} + \delta \lambda_i \delta \mathbf{q}_{i2}$$

where (2-66)

$$\delta \mathbf{q}_{i1} = -\mathbf{K}_T^{-1} \mathbf{F}(\mathbf{q}_i, \lambda_i), \quad \delta \mathbf{q}_{i2} = -\mathbf{K}_T^{-1} \mathbf{R}$$

Each incremental displacement and load increment values are based on the results of the previous increment and a small change δ :

$$\begin{aligned} \Delta \mathbf{q}_{i+1} &= \Delta \mathbf{q}_i + \delta \mathbf{q}_{i+1} \\ \Delta \lambda_{i+1} &= \Delta \lambda_i + \delta \lambda_i \end{aligned} \quad (2-67)$$

Substituting Eqn(2-67) into Eqn(2-65) yields:

$$a \delta \lambda_i^2 + b \delta \lambda_i + c = 0$$

where (2-68)

$$\begin{aligned} a &= \delta \mathbf{q}_{i2}^T \delta \mathbf{q}_{i2}, \quad b = 2 \delta \mathbf{q}_{i2}^T (\Delta \mathbf{q}_i + \delta \mathbf{q}_i) \\ c &= (\Delta \mathbf{q}_i + \delta \mathbf{q}_i)^T (\Delta \mathbf{q}_i + \delta \mathbf{q}_i) - \Delta l^2 \end{aligned}$$

The preceding system of equations converges when $\delta \mathbf{q}_i$ becomes smaller

than a user defined value. After convergence at load step m , the total displacement \mathbf{q}_m , the total load parameter λ_m , and the incremental load step Δl_m are calculated using:

$$\begin{aligned}\mathbf{q}_m &= \mathbf{q}_{m-1} + \Delta \mathbf{q}_m \\ \lambda_m &= \lambda_{m-1} + \Delta \lambda_m \\ \Delta l_m &= \Delta l_{m-1} \frac{N_s}{N_{m-1}}\end{aligned}\quad (2-69)$$

where N_s is a user defined number of iteration prediction and N_{m-1} the number of iterations required for convergence of step $m-1$. Once the search radius for a step is defined, the initial load parameter is found using:

$$\Delta \lambda_1 = \pm \frac{\Delta l_m}{(\delta \mathbf{q}_{i2}^T \delta \mathbf{q}_{i2})^{1/2}} \quad (2-70)$$

2.8 Step by Step Riks Algorithm

This Riks method example is presented by Creaghan and Miller and is repeated here for clarity (3; 13). Figure 2-12 graphically depicts the process. For each Riks solution, the estimated number of iterations (N_s), total number of steps (m), the incremental displacement limit (\mathbf{q}_i), and the initial load parameter (λ_0) are user prescribed parameters.

1. First increment, first iteration: compute only the constant stiffness matrix, \mathbf{K} .
2. First iteration, subsequent increments: compute $\delta \mathbf{q}_{i2} = -\mathbf{K}_T^{-1} \mathbf{R}$. After the first increment, \mathbf{K}_T is composed from \mathbf{q}_{m-1} nonlinear displacement terms.
3. Each iteration, each increment: compute $\Delta \mathbf{q}_i = \Delta \lambda_i \delta \mathbf{q}_{i2}$. For the first increment, $\Delta \lambda_i = \lambda_0$, which is a program input. Current examples

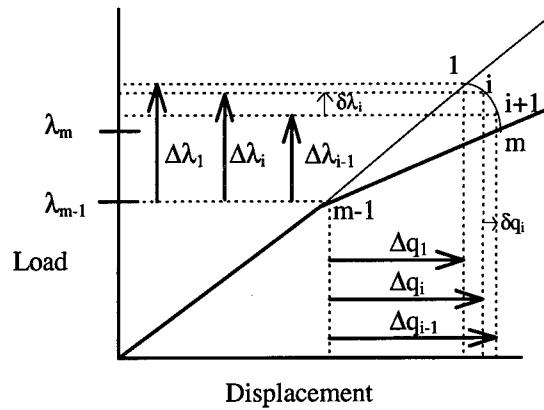


Figure 2-12 Riks Method

used $\lambda_0 = 0.1$ as did Creaghan and Miller with acceptable results (3; 13). For subsequent increments, $\Delta\lambda_i = \lambda_i$.

4. Each iteration, each increment: compute in order:

$$\begin{aligned}\delta\mathbf{q}_i &= -\mathbf{K}_T^{-1} \mathbf{F}(\mathbf{q}_i, \lambda_i) \\ \delta\mathbf{q}_i &= \delta\mathbf{q}_i + \delta\lambda_i \delta\mathbf{q}_{i+1} \\ \Delta\mathbf{q}_{i+1} &= \Delta\mathbf{q}_i + \delta\mathbf{q}_{i+1}\end{aligned}\quad (2-71)$$

5. Compute $\Delta l = (\Delta\mathbf{q}_{i+1}^T \Delta\mathbf{q}_{i+1})$.
6. Update \mathbf{K}^T with $\mathbf{q}_i = \mathbf{q}_{m-1} + \Delta\mathbf{q}_i$.
7. Solve Eqn(2-68) for $\pm\delta\lambda_i$. For complex roots, return to step 2 and arbitrarily adjust Δl_m , which changes $\Delta\lambda_i$.
8. Choose $\pm\delta\lambda_i$ based on which value yields a positive θ in:

$$\begin{aligned}\delta\mathbf{q}_i &= \delta\mathbf{q}_i \pm \delta\lambda_i \delta\mathbf{q}_{i+1} \\ \theta &= (\Delta\mathbf{q}_i + \delta\mathbf{q}_i) \Delta\mathbf{q}_i\end{aligned}\quad (2-72)$$

If both are positive, then $\delta\lambda_i = -c/b$.

9. Update the displacement and loading parameters using:

$$\begin{aligned}\Delta\mathbf{q}_{i+1} &= \Delta\mathbf{q}_i + \delta\mathbf{q}_{i+1} \\ \Delta\lambda_{i+1} &= \Delta\lambda_i + \delta\lambda_i\end{aligned}\quad (2-73)$$

10. Check for convergence. If no convergence, return to step 2. upon convergence, update the displacement, load parameter, and search radius for the next increment:

$$\begin{aligned}\mathbf{q}_m &= \mathbf{q}_{m-1} + \Delta \mathbf{q}_m \\ \lambda_m &= \lambda_{m-1} + \Delta \lambda_m \\ \Delta l_m &= \Delta l_{m-1} \frac{N_s}{N_{m-1}}\end{aligned}\quad (2-74)$$

where N_s is a user defined number of iteration prediction and N_{m-1} the number of iterations required for convergence of step $m-1$.

The current work uses $N_s = 2.5$.

11. Compute the loading parameter for the first iteration of the next increment:

$$\Delta \lambda_1 = \pm \frac{\Delta l_m}{(\delta \mathbf{q}_{i2}^T \delta \mathbf{q}_{i2})^{1/2}} \quad (2-75)$$

12. Return to step 2 to begin the next increment.

2.9 Arch Geometry Considerations

As previously discussed, geometry plays a significant role in the arch behavior. Figure 2-13 presents a typical arch geometry. Miller

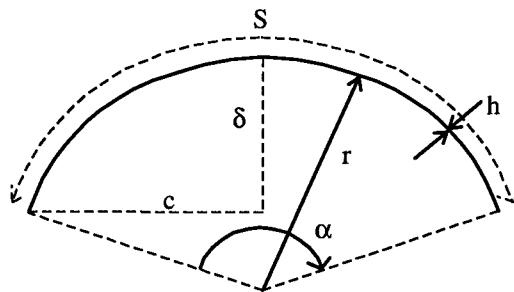


Figure 2-13 Arch Geometry Definitions

considers an arch thick when (12):

$$\frac{c}{h} < 25 \quad (2-76)$$

He also defines a deep arch using:

$$\lambda \geq 8$$

where (2-77)

$$\lambda = \left(12(1-v^2)\right)^{1/4} \left(\frac{r}{h}\right)^{1/2} \frac{\alpha}{2}$$

These classification will be used when comparing the current work with Miller's.

III. Results and Discussion

3.1 Overview

This chapter is divided into two sections. In the first section, the current work is compared to Miller's work to assess the effects of the Hermitian shape functions. The second section presents the results of extensive arch analysis using the current program.

3.2 Comparison

3.2.1 Clamped Isotropic Shallow Thin Arch. The first problem considered was originally presented by Belytschko and Glaum(1) and is

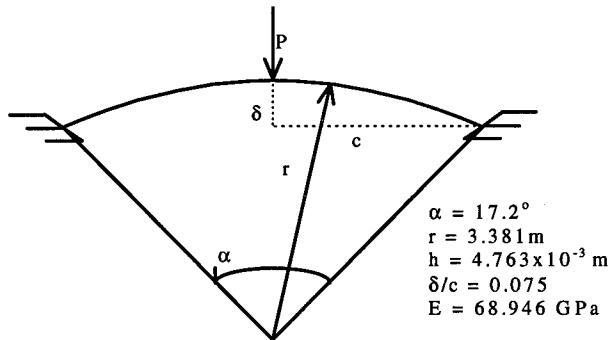


Figure 3-1 Clamped Isotropic Shallow Thin Arch

presented here in figure 3-1. Miller and Creaghan also analyze this structure(3; 13). Because the current work is an evolution of Miller's, comparisons are only made with his results.

This first example combines large displacements and geometric nonlinearity with small rotations. As such, it is used to validate the current work and demonstrate that the updated shape functions do not corrupt the program's capabilities. The structure is symmetrically

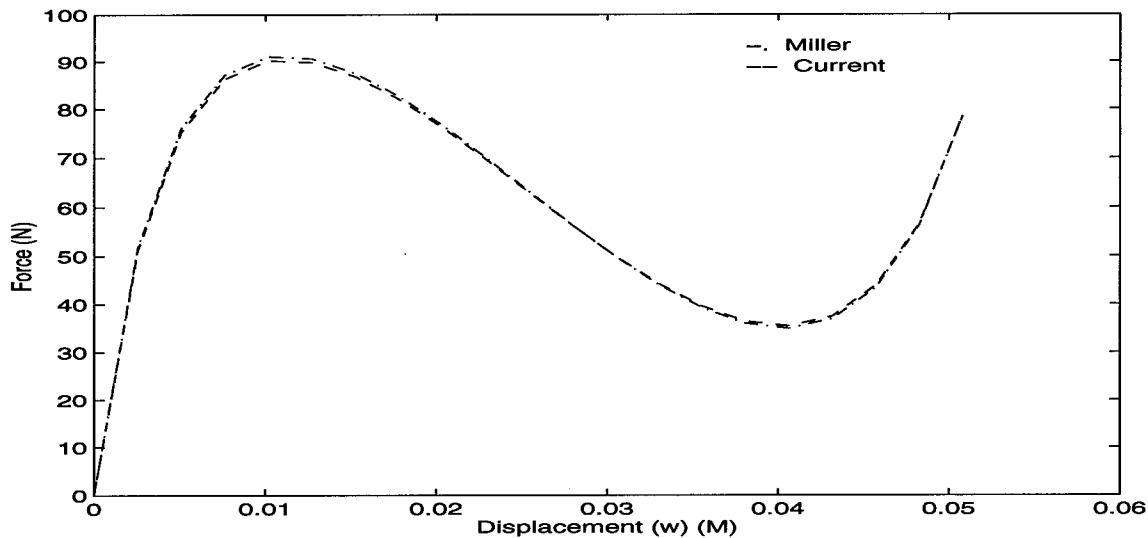


Figure 3-2 Comparison for Clamped Shallow Thin Arch

modeled using 17 elements. Miller's model contained 81 active degrees of freedom (DOF) while the current work used 99. Both formulations use displacement control with a convergence tolerance of 0.1% and maximum of 200 iterations per displacement increment. Twenty increments are used with an initial displacement of 2.54mm. Convergence is based on:

$$\frac{\sqrt{\sum_i (q_r^i)^2} - \sqrt{\sum_i (q_{r-1}^i)^2}}{\sqrt{\sum_i (q_1^i)^2}} \times 100\% \leq TOL \quad (3-1)$$

where q_r^i , q_{r-1}^i , and q_1^i are elements of the global displacement vector, for the r^{th} , $(r - 1)^{\text{th}}$, and the first load increment, $i = 1$ to n degrees of freedom, and TOL is the user defined tolerance (19).

Figure 3-2 compares the vertical displacement (w) at node 8 (the point of symmetry where P is applied) for each solution method. The maximum difference is less than 1.0% at 10.16mm of displacement. This figure is attained using:

$$\text{difference} = \left| \frac{\text{Miller's} - \text{Current}}{\text{Miller's}} \right| \times 100 \quad (3-2)$$

This confirms correct introduction of Hermitian shape functions into the finite element formulation.

Examining the derivative of the bending angle ($\psi_{,2}$) demonstrates the effect of the higher order shape function. Figure 3-3 compares ($\psi_{,2}$) for the current work with Miller's results. The linear shape

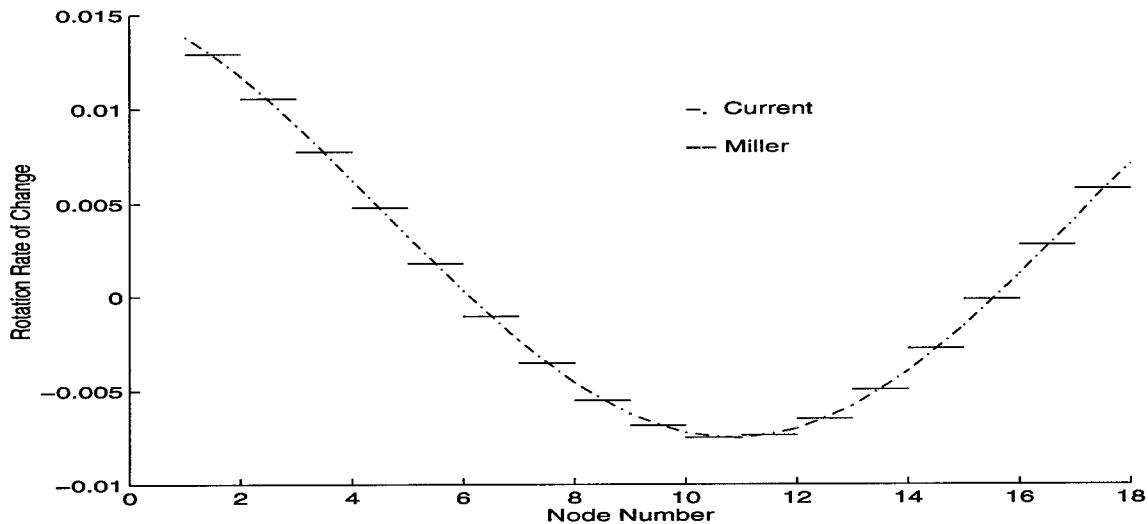


Figure 3-3 Clamped Shallow Thin Arch Rotation Rate of Change Comparison

function of the previous effort create a curve with step discontinuities at each node. The current work provides a continuous curve for the bending angle rate of change expression. Thus, the higher order shape function is indeed properly incorporated and does not degrade the capabilities of the previous effort.

To examine the efficiency of the two programs, the number of elements used to model the structure is incrementally reduced until the solutions deviated from the "approved" solution presented in figure 3-2 by more than 1.0%. Both formulations were run using the same setup as the initial solution with the exception of the reduced number of elements. Using 12 elements and 56 active DOF, Miller's program

deviated 0.848%. Using this model, his program required 23.1 seconds to solve using a Sun Sparc20 workstation. The current work required only 7 elements and 39 active DOF to achieve similar results, using the same convergence parameters. This setup required only 17.3 seconds to run on the same workstation, a 25.1% reduction.

3.2.2 Cantilever Isotropic Thin Beam. Figure 3-4 presents the second problem examined. This structure experiences large deflections

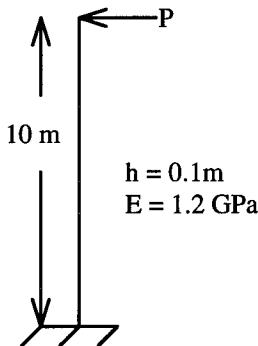


Figure 3-4 Cantilever Isotropic Beam

and moderate rotations. Hsiao and Hou initially examine this beam using an updated Lagrangian formulation (14). The structure is modeled with 40 elements providing 200 active DOF for Miller's solution and 240 for the current program. Displacement control was used with an initial displacement of 0.15m, 58 increments, a limit of 100 iterations per increment, and a tolerance of 0.5%. Figure 3-5 compares the tip displacement (w) solution of the current method to Miller's. Because this structure experiences large rotations, the current work should deviate from Miller's if the bending angle shape functions are the limiting factor of the previous work. However, the two solutions are almost identical. This is the first indication that the limit of the

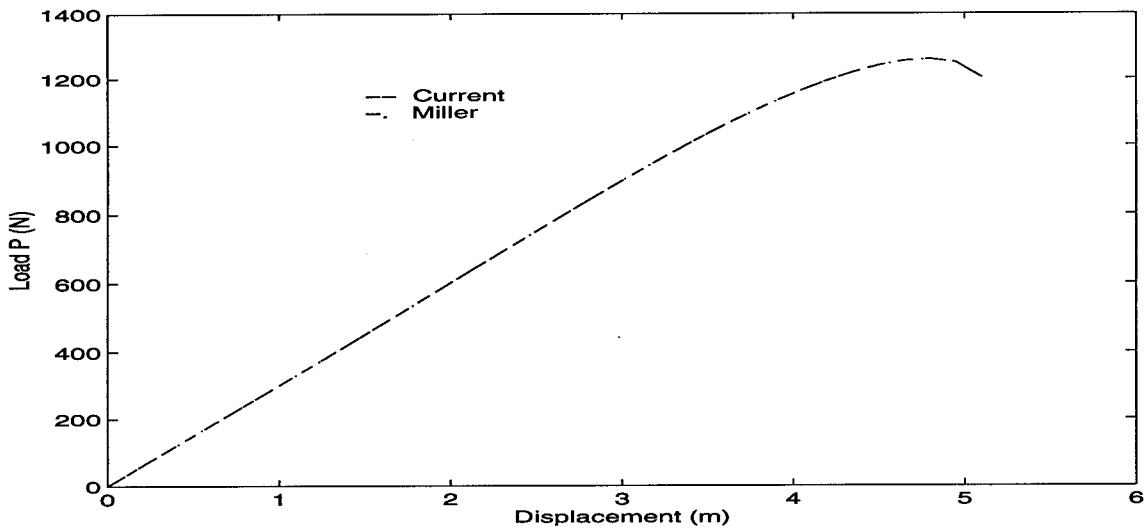


Figure 3-5 Cantilever Isotropic Beam End Displacement Comparison

previous effort is not in the finite element formulation, but in the derivation of the equilibrium equations.

The derivative of the bending angle ($\psi_{,2}$) is examined for the entire beam at maximum load. Figure 3-6 presents these results. As with the first example, the current effort generates a continuous curve

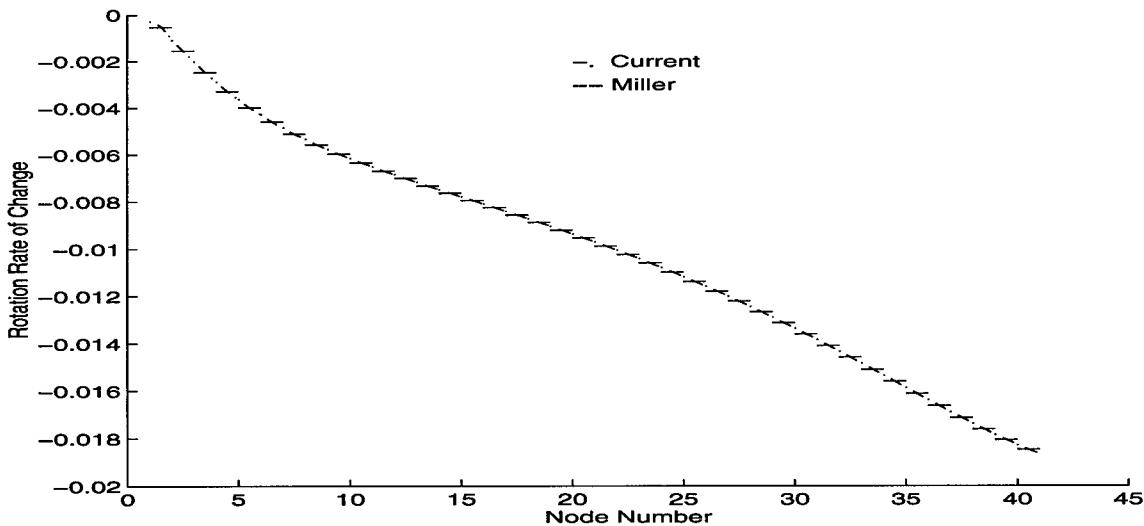


Figure 3-6 Cantilever Beam Rotation Rate of Change Comparison

whereas Miller's results are not. This indicates the Hermitian shape functions do properly estimate the bending angle and its first derivative.

The efficiency was also examined using the procedure outlined in section 3.2.1. Miller's program required 14 elements and 70 active DOF to match the solution generated using 40 elements. This model required 45.4 seconds to solve. The current work required fewer elements (13) but more DOF (78). This caused the current program to run 70.7 seconds before reaching a solution.

3.2.3 Very Deep Hinged Clamped Isotropic Arch. The following example combines large rotations and large displacements. The unsymmetrical boundary conditions combine with the very deep geometry, as shown in figure 3-7, to present a formidable test of any large

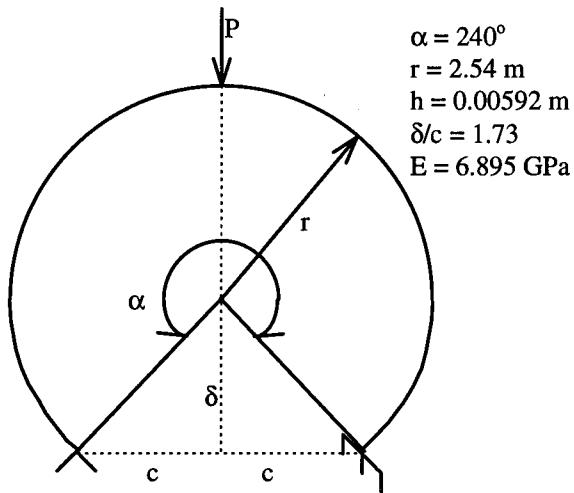


Figure 3-7 Very Deep Hinged Clamped Isotropic Arch

rotation and large displacement theory. The arch was modeled with 100 elements and the displacement control method was used. A maximum of 100 iterations was specified for each of the 200 increments, and the tolerance was set at 2.0%.

The comparison with Miller's work is shown in Figure 3-8. The programs produce nearly identical results up to the collapse load of

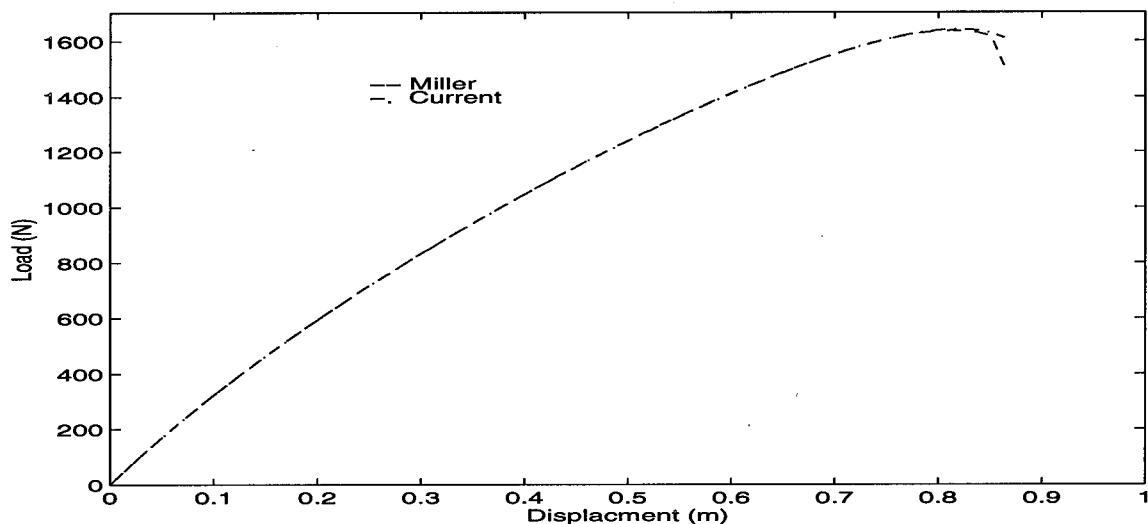


Figure 3-8 Load vs. Displacement for a Very Deep Hinged Clamped Arch

1.720 kn. and a displacement of 0.826 m. At this point the solutions diverge. However, Miller discovered solutions generated after collapse were unreliable due to large energy changes within the elements (13). As such, solutions reached after collapse are neglected.

The rate of change of the bending rotation (ψ_2) is presented in figure 3-9 and shows the same results as the previous two examples. Again, C^1 continuity of the bending angle is attained. Even with the higher order continuity, the current model does not provide a better solution. These results clearly indicate a limitation caused by the linearization of the rotation angle (ψ) during derivation of the large rotation kinematics.

Although the rotation limits have not been improved, the current work does address another shortcoming of the previous efforts. At large deflections, Miller noted a propensity for elements near the hinged

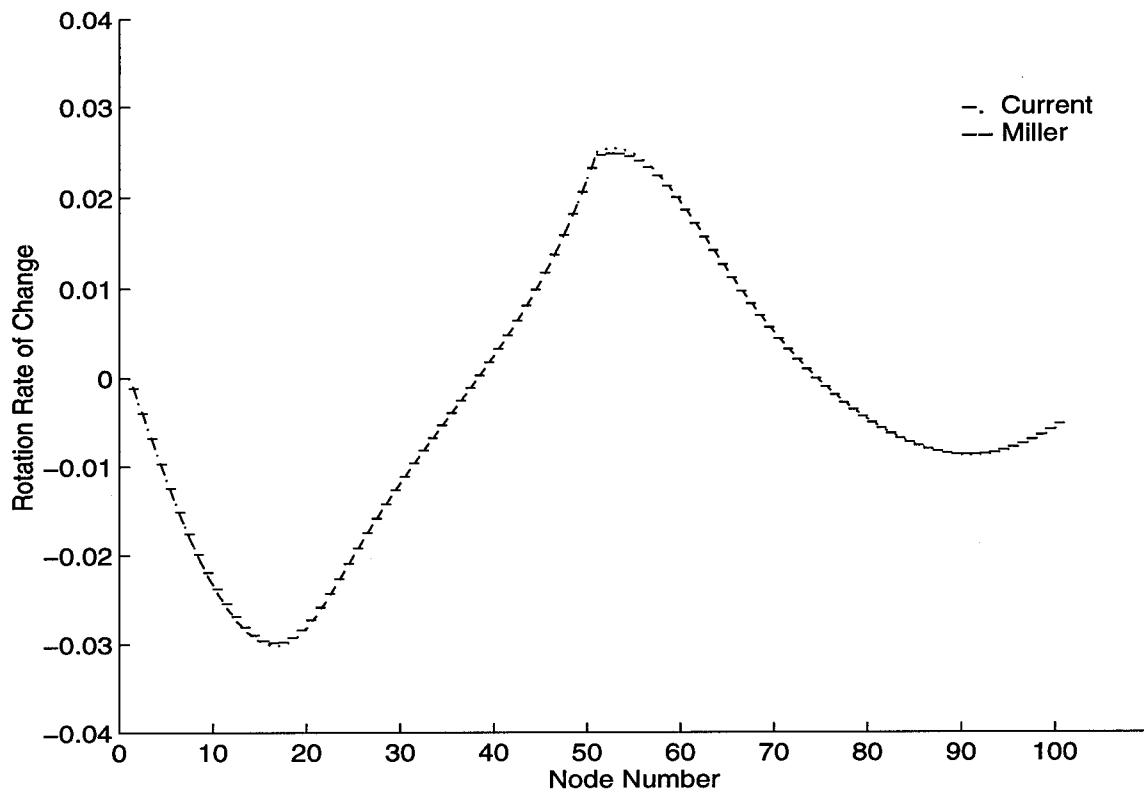


Figure 3-9 Rotation Rate of Change Comparison

boundary to "kink over" (13). Figure 3-10 shows the undeformed and deformed shapes created by each model at 35 inches of vertical deflection at the middle node. Despite providing similar solutions for the displacement at the middle of the arch, the kinking phenomenon Miller noticed is nonexistent in the current solution. As a result, displacements near the hinged boundary are significantly different. For this reason, the current model should be used for investigating large rotation problems.

Another advantage of the current formulation is that erroneous solutions are eliminated. In this example, Miller's formulation continued providing solutions after the model experienced element

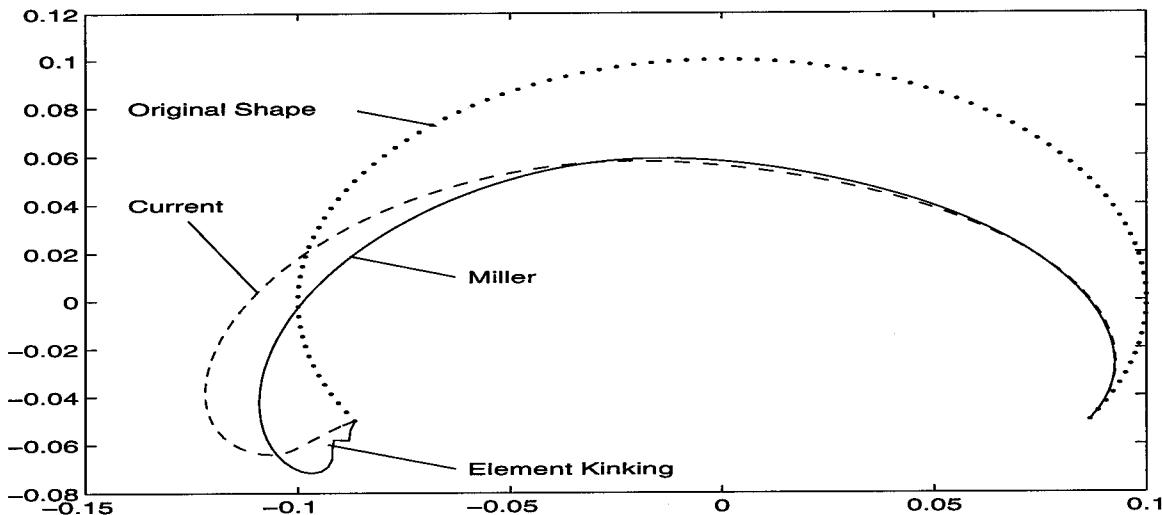


Figure 3-10 Deformed Shapes for Clamped Hinged Very Deep Arch

kinking despite the unstable nature of the structure (13). The current program does not converge on a solution beyond this deflection level.

The elimination of element kinking and inaccurate solutions clearly demonstrate the advantage of using Hermitian shape functions for interpolation of the bending angle (ψ) when modeling arches with large displacements and large rotations.

For this example, the current program required 98 elements with 586 active DOF to maintain an accurate solution. Miller's program only required 88 elements and 440 active DOF. As a result, the current formulation ran for 1049 seconds versus 805 seconds for the previous effort. Clearly, the trade off for C^1 continuity is a finer mesh and larger number of DOF, which in turn require more time to solve.

3.3 Arch Evaluation

This section makes use of the newly developed curved element to characterize certain geometric parameters. Two classes of composite arches are evaluated. The first group investigates how the arc

length (S) affects the collapse load (P_c). The second group evaluates the effect of changing the thickness ratio (h/r) versus P_c . In either case, simply supported composite arches are symmetrically modeled using the current program. All loads were applied and displacements (w) measured at the point of symmetry. The arches are constructed of a $[0/90]_s$ lay-up of AS4-3501-6 graphite epoxy. Each ply is 0.25mm thick, for a total thickness of 1mm. The lamina material properties are: $E_1=142$ GPa; $E_2=9.8$ GPa; $G_{12}=G_{13}=6.0$ GPa; $G_{23}=4.8$ GPa; and $\nu_{12}=0.3$. These arches experience large deformations and moderate rotations. Riks method is used for all simulations with a tolerance of 0.5%, a maximum of 500 increments, and a limit of 100 iterations per increment. The initial load parameter (λ_0) was set at 0.1, with the number of iterations estimated as 2.5 (N_s), and a maximum allowable load change (λ_{\max}) of 2.0.

3.3.1 Constant h/r , Variable Arc Length. For this portion of the evaluation, five different thickness ratios are considered. Each h/r was established by varying the radius while holding the thickness at 1mm. For each h/r the arc length was varied to determine how boundary location affects arch behavior. Table 3-1 presents the specifications of each arch analyzed for this section along with the depth and thickness parameters used for evaluation. Representative figures are presented in Appendix B.

Table 3-1 includes two classification parameters for thickness and depth evaluation. For thickness, either h/r or c/h may be used. The c/h parameter takes opening angle into account and is more accurate as the opening angle increases. Either δ/c or λ are used for depth classification. λ is calculated using $\lambda = [12(1-\nu^2)] \cdot 25 \cdot (r/h) \cdot 5 \cdot (\alpha/2)$.

Table 3-1 Arch Investigation Matrix, Constant h/r , $h = 0.001m$

h/r		S (m)	r (m)	c/h		δ/c		λ	
0.1	thick	0.01	0.01	4.79	thick	0.26	shallow	2.94	shallow
0.1	thick	0.015	0.01	6.82	thick	0.39	deep	4.41	shallow
0.1	thick	0.025	0.01	9.49	thick	0.72	deep	7.36	shallow
0.1	thick	0.05	0.01	5.98	thick	3.01	deep	14.71	deep
0.05	thick	0.015	0.02	7.33	thick	0.18	shallow	3.12	shallow
0.05	thick	0.02	0.02	9.59	thick	0.26	shallow	4.16	shallow
0.05	thick	0.025	0.02	11.70	thick	0.32	deep	5.20	shallow
0.05	thick	0.03	0.02	13.63	thick	0.39	deep	6.24	shallow
0.05	thick	0.05	0.02	19.98	thick	0.72	deep	10.40	deep
0.05	thick	0.075	0.02	19.08	thick	1.36	deep	15.61	deep
0.05	thick	0.1	0.02	11.97	thick	3.01	deep	20.81	deep
0.01	thin	0.025	0.1	12.47	thick	0.06	shallow	2.33	shallow
0.01	thin	0.05	0.1	24.74	thick	0.13	shallow	4.65	shallow
0.01	thin	0.075	0.1	36.63	thin	0.19	shallow	6.98	shallow
0.01	thin	0.1	0.1	47.94	thin	0.26	shallow	9.31	deep
0.01	thin	0.15	0.1	68.16	thin	0.39	deep	13.96	deep
0.01	thin	0.2	0.1	84.15	thin	0.55	deep	18.61	deep
0.005	thin	0.05	0.2	24.93	thick	0.06	shallow	3.29	shallow
0.005	thin	0.075	0.2	37.28	thin	0.09	shallow	4.93	shallow
0.005	thin	0.1	0.2	49.48	thin	0.13	shallow	6.58	shallow
0.005	thin	0.15	0.2	73.25	thin	0.19	shallow	9.87	deep
0.005	thin	0.2	0.2	95.89	thin	0.26	shallow	13.16	deep
0.005	thin	0.3	0.2	136.3	thin	0.39	deep	19.74	deep
0.0025	thin	0.05	0.4	24.98	thick	0.03	shallow	2.33	shallow
0.0025	thin	0.075	0.4	37.45	thin	0.05	shallow	3.49	shallow
0.0025	thin	0.1	0.4	49.87	thin	0.06	shallow	4.65	shallow
0.0025	thin	0.15	0.4	74.56	thin	0.09	shallow	6.98	shallow
0.0025	thin	0.2	0.4	98.96	thin	0.13	shallow	9.31	deep
0.0025	thin	0.4	0.4	191.8	thin	0.26	shallow	18.61	deep
0.0025	thin	0.6	0.4	272.6	thin	0.29	deep	27.92	deep

Similar to the thickness classification parameters, as α increases, the λ parameter is preferred because it directly accounts for the opening angle.

Figures 3-11 presents a typical P_c vs. w curve. The critical load is defined as the local peak indicated by point A. In the cases where no local peak exists, it is assumed the structure does not collapse but has post buckling features. Figure 3-12 presents the P_c v S data for $h/r = 0.1$. As arc length approaches zero, P_c smoothly approaches

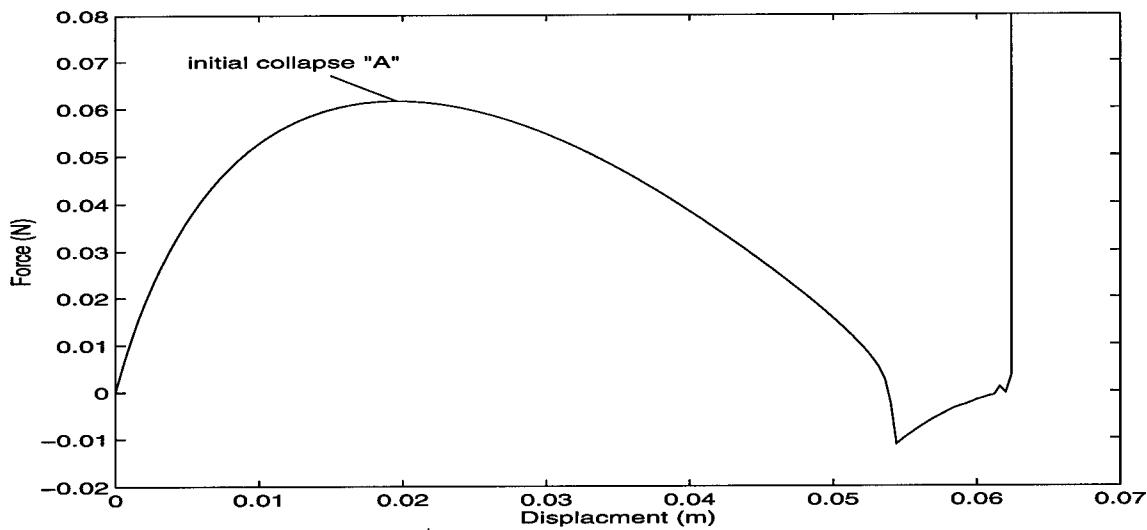


Figure 3-11 Typical Arch P_c vs. w Curve

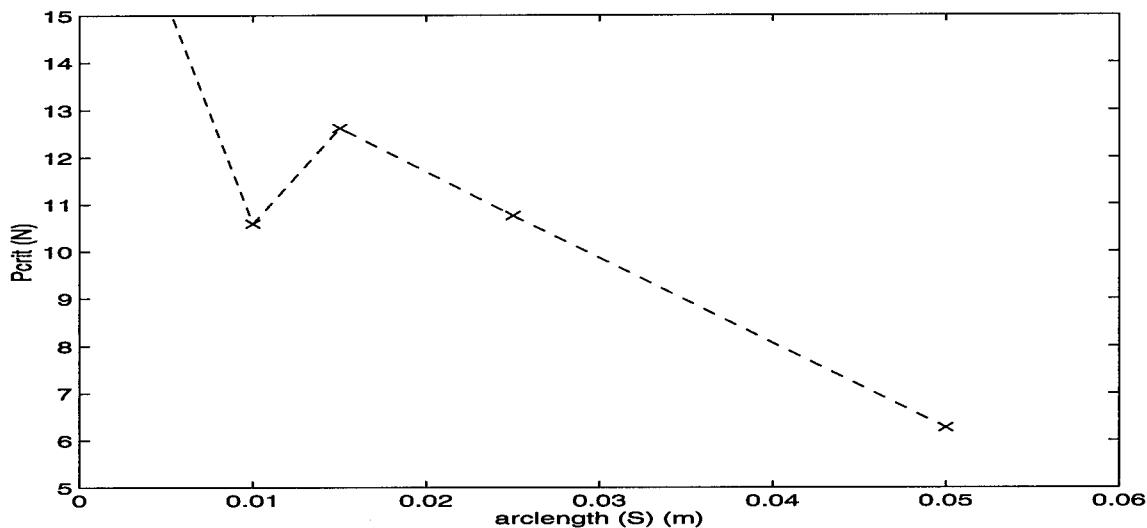


Figure 3-12 Critical Load Comparison for $h/r = 0.1$

infinity except for the decrease at $S = 0.01\text{m}$. Further examination indicates this arch experiences local snapping and does not totally collapse (total collapse is defined as global snapping). That is, the arch geometry changes but is immediately able to support a load with

minimal deflection increases. Figure 3-13 presents the load displacement curve for an arch with arc length of 0.01m and thickness ratio of 0.1. As indicated, the arch initially collapses at 10.059 N at

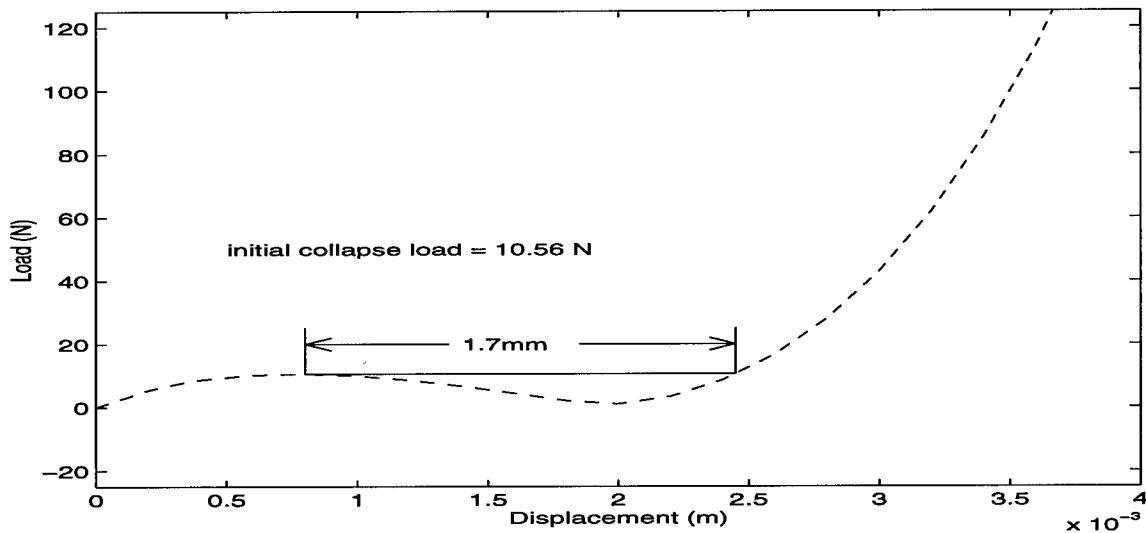


Figure 3-13 Load v Displacement for $S = 0.01\text{m}$, $h/r = 0.1$

a displacement of 0.75mm. This arch is able to support the same load again after the displacement increases only 1.7mm to 2.45mm. Figure 3-14 presents the actual arch shape at the collapse displacement. The counterflexure points indicate the point on the arch where the radius changes signs. Outside these points, the arch is concave downward and between the counterflexure points the arch is concave upward. The distance between the points is indicative of the stability the arch. As the distance increases, stability decreases along with the ability to support a load. As indicated in figure 3-14, the counterflexure points of the $S = 0.01\text{m}$, $h/r = 0.1$ arch are separated by 1.5mm or 15% of the arc length. This small separation, combined with the minimal displacement change, indicate that a local snapping occurs.

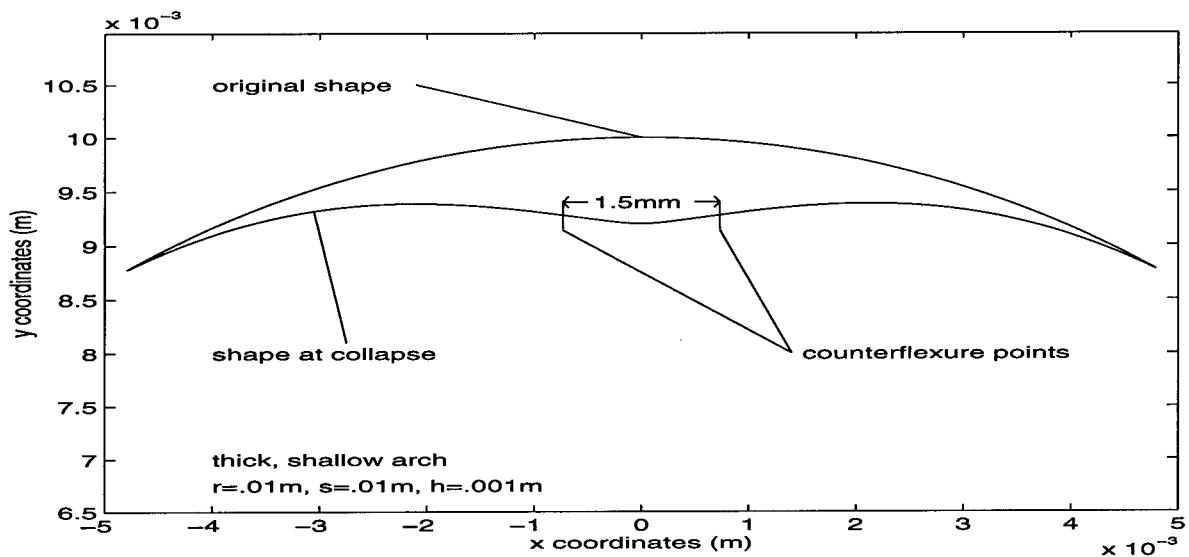


Figure 3-14 Undeformed and Collapsed Arch Shapes; $S = 0.01\text{m}$, $h/r = 0.1$

To demonstrate the total collapse of an arch, the typical load displacement curve for an arch with $S = 0.025\text{m}$ and $h/r = 0.1$ is considered. This curve is presented in figure 3-15. This structure initially collapses at a load of 10.76N and a displacement of 2.72mm .

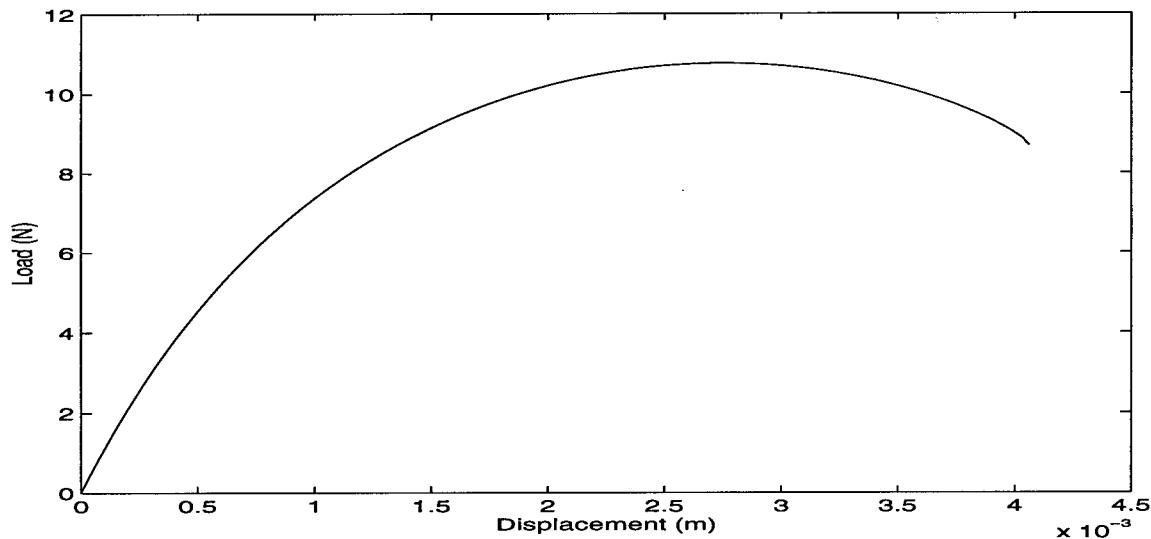


Figure 3-15 Load v Displacement for $S = 0.025\text{m}$, $h/r = 0.1$

As shown, the arch does not recover after the initial collapse point, indicating a global snapping or total collapse of the structure.

The undeformed shape of this arch is presented in figure 3-16 along with the shape at collapse. The counterflexure points are

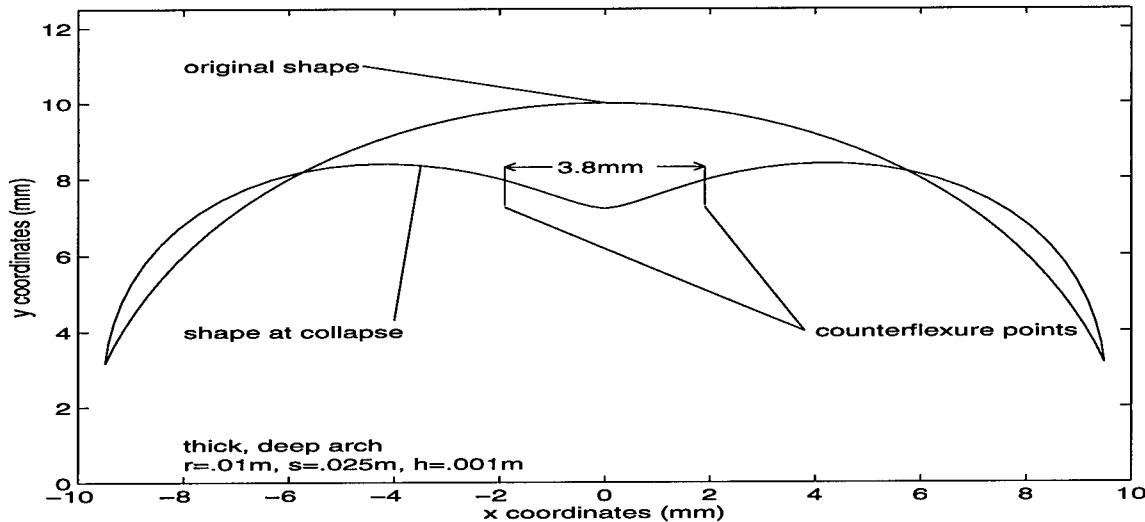


Figure 3-16 Undefomed and Collapsed Arch Shapes; $S = 0.025\text{m}$, $h/r = 0.1$

separated by 3.8mm (15.2% of S) at collapse. Both this arch ($S = 0.025$, $h/r = 0.1$) and the previous arch ($S = 0.01$, $h/r = 0.1$) have similar counterflexure separation distances (15% of S) and collapse loads (10.5N).

Although both arches collapse at similar loads, the longer arch displaces almost 12% of its length before collapsing versus 7% for the shorter arch. Also, the longer arch does not recover after initial collapse, indicating a that global snapping occurs.

Similar results were observed for thickness ratios (h/r) of 0.05, 0.01, 0.005, and 0.0025. These results are summarized in table 3-2 and presented graphically in Appendix B.

Table 3-2 Constant h/r Results

h/r	S (m)	P_c (N)	w_c (mm)	recover	post snap disp (mm)	type of snapping
0.1	0.01	10.587	0.7471	yes	1.70292	local
0.1	0.015	12.616	1.2727	no	n/a	global
0.1	0.025	10.761	2.7196	no	n/a	global
0.1	0.05	6.2709	7.059	no	n/a	global
0.05	0.015	4.8981	0.8508	yes	1.9892	local
0.05	0.02	5.8055	1.2548	no	n/a	global
0.05	0.025	5.6238	1.7891	no	n/a	global
0.05	0.03	5.1597	2.385	no	n/a	global
0.05	0.05	3.5331	5.7071	no	n/a	global
0.05	0.075	2.7036	12.055	no	n/a	global
0.05	0.1	1.7821	14.187	no	n/a	global
0.01	0.025	0.3848	0.66	yes	0.645	local
0.01	0.05	0.67085	1.6038	yes	0.808	local
0.01	0.075	0.52102	3.23	no	n/a	global
0.01	0.1	0.40145	5.5168	no	n/a	global
0.01	0.15	0.26823	11.795	no	n/a	global
0.01	0.2	0.19992	19.769	no	n/a	global
0.005	0.05	0.20175	0.97761	yes	2.31239	local
0.005	0.075	0.2405	1.81	yes	5.6552	local
0.005	0.1	0.1984	2.9864	no	n/a	global
0.005	0.15	0.13688	6.4393	no	n/a	global
0.005	0.2	0.10279	10.941	no	n/a	global
0.005	0.3	0.06824	23.472	no	n/a	global
0.0025	0.05	0.04979	0.6667	yes	0.6532	local
0.0025	0.075	0.07853	0.9867	yes	2.8736	local
0.0025	0.1	0.08843	1.654	yes	4.7291	local
0.0025	0.15	0.06757	3.366	no	n/a	global
0.0025	0.2	0.05178	5.829	no	n/a	global
0.0025	0.4	0.02605	21.968	no	n/a	global
0.0025	0.6	0.0133	47.795	no	n/a	global

The results presented in table 3-2 highlight two important trends in arch behavior. First, the arches that recover after collapse, or experience local snapping, are generally classified as shallow structures, based on the specifications listed in table 3-1. It is interesting to note that for smaller radii ($r = 0.01m$ and $0.02m$), the δ/c parameter correctly classifies shallow arches as those that experience local snapping, but as the radius increases, the λ parameter provides a more accurate prediction.

The second trend is for P_c to increase as h/r increases for a given arc length. This indicates the influence of through the thickness shear, as will be described in the next section.

3.3.2 Constant Arc Length, Variable h/r . Table 3-3 presents the specifications of each arch analyzed for this section along with the depth and thickness parameters used for evaluation.

Table 3-3 Arch Investigation Matrix, Constant S , $h = 0.001m$

S (m)	h/r		r (m)	c/h		δ/c		λ	
0.025	0.1	thick	0.01	9.49	thick	0.72	deep	7.36	shallow
0.025	0.05	thick	0.02	11.70	thick	0.32	deep	5.20	shallow
0.025	0.01	thin	0.1	12.47	thick	0.06	shallow	2.33	shallow
0.05	0.1	thick	0.01	5.98	thick	3.01	deep	14.71	deep
0.05	0.05	thick	0.02	18.98	thick	0.72	deep	10.40	deep
0.05	0.01	thin	0.1	24.74	thick	0.13	shallow	4.65	shallow
0.05	0.005	thin	0.2	24.93	thick	0.06	shallow	3.29	shallow
0.075	0.05	thick	0.02	19.08	thick	1.36	deep	15.61	deep
0.075	0.01	thin	0.1	36.63	thin	0.19	shallow	6.98	shallow
0.075	0.005	thin	0.2	37.28	thin	0.09	shallow	4.93	shallow
0.075	0.0025	thin	0.4	37.45	thin	0.05	shallow	3.49	shallow
0.1	0.05	thick	0.02	11.97	thick	3.01	deep	20.81	deep
0.1	0.01	thin	0.1	47.94	thin	0.26	shallow	9.31	deep
0.1	0.005	thin	0.2	49.48	thin	0.13	shallow	6.58	shallow
0.1	0.0025	thin	0.4	49.87	thin	0.06	shallow	4.65	shallow
0.15	0.01	thin	0.1	68.16	thin	0.39	deep	13.96	deep
0.15	0.005	thin	0.2	73.25	thin	0.19	shallow	9.87	deep
0.15	0.0025	thin	0.4	74.56	thin	0.09	shallow	6.97	shallow
0.2	0.01	thin	0.1	84.15	thin	0.55	deep	18.61	deep
0.2	0.005	thin	0.2	95.89	thin	0.26	shallow	13.16	deep
0.2	0.0025	thin	0.4	98.96	thin	0.13	shallow	9.31	deep

Figure 3-17 presents a P_c vs. h/r curve for arches 0.025m long. This curve was constructed by varying the radius of curvature while maintaining a thickness of 1mm. As shown in figure 3-18, as h/r decreases due to the increasing radius, the depth of the arch decreases or becomes more like a flat beam.

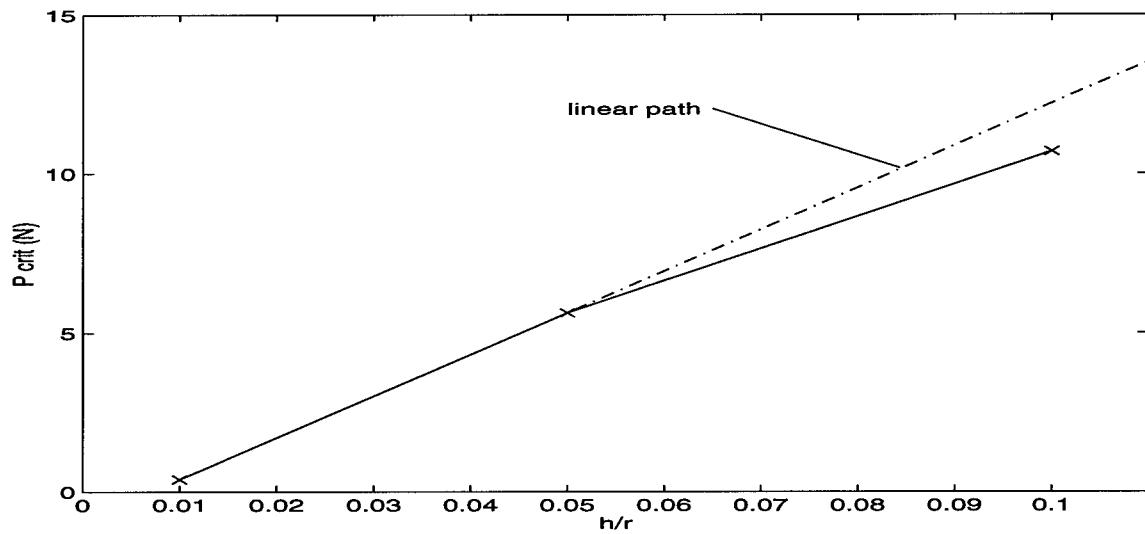


Figure 3-17 P_c vs. h/r for $S = 0.025m$

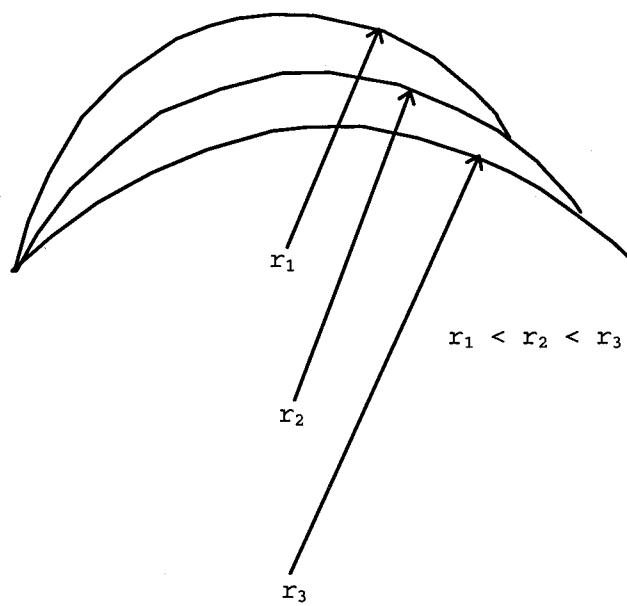


Figure 3-18 Effects of Increasing Radius for a Constant Arc Length

As expected, P_c increases as h/r increases; however, as indicated in figure 3-17, the rate of change of the critical load decreases as the thickness ratio increases (radius decreases). All three arches used to

generate the curve are considered thick based on $c/h < 25$. As such, the deviations observed in the P_c vs. h/r curve demonstrate the nonlinear effects caused by significant through the thickness shear stresses experienced by thick beams and arches.

Figure 3-19 presents the P_c vs. h/r for an arc length of 0.2m. A slight variation in the critical load rate of change was observed. In this case, all three $S = 0.2\text{m}$ arches were thin arches, based on the $c/h > 25$ criteria presented in table 3-3. As such, the P_c vs. h/r curve

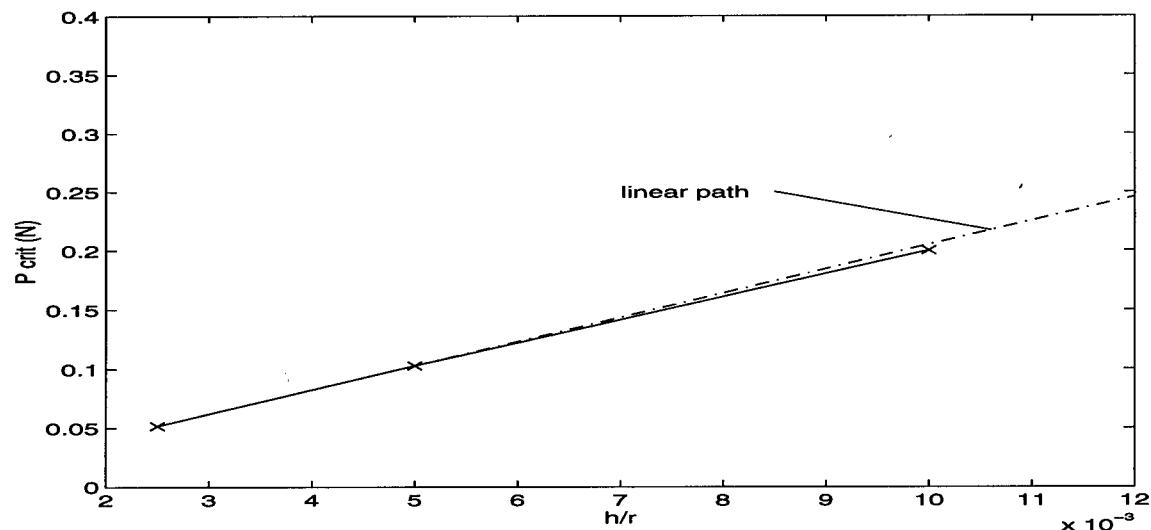


Figure 3-19 P_c vs. h/r for $S = 0.2\text{m}$

indicates minimal nonlinear behavior caused by through the thickness shear.

The critical displacement (w_c) for these arches was also examined. Figure 3-20 shows the w_c vs. h/r curve for $S = 0.025\text{m}$. This curve deviates from the linear path as indicated. Figure 3-21 presents the w_c vs. h/r curve for $S = 0.2\text{m}$. In both cases, as the arches modeled become

deeper, the rate of change of the critical displacement decreases, regardless of the presence of through the thickness shear effects.

The analysis of arches with constant arc length and thickness and changing radius of curvature demonstrates the nonlinearities introduced as depth and through the thickness shear increase.

Similar P_c vs. h/r and w_c vs. h/r curves were constructed for arc lengths of 0.05m, 0.075m, 0.1m, and 0.15m. These curves further exemplify the relationships between critical load and thickness and critical displacement and depth. Appendix C contains these curves.

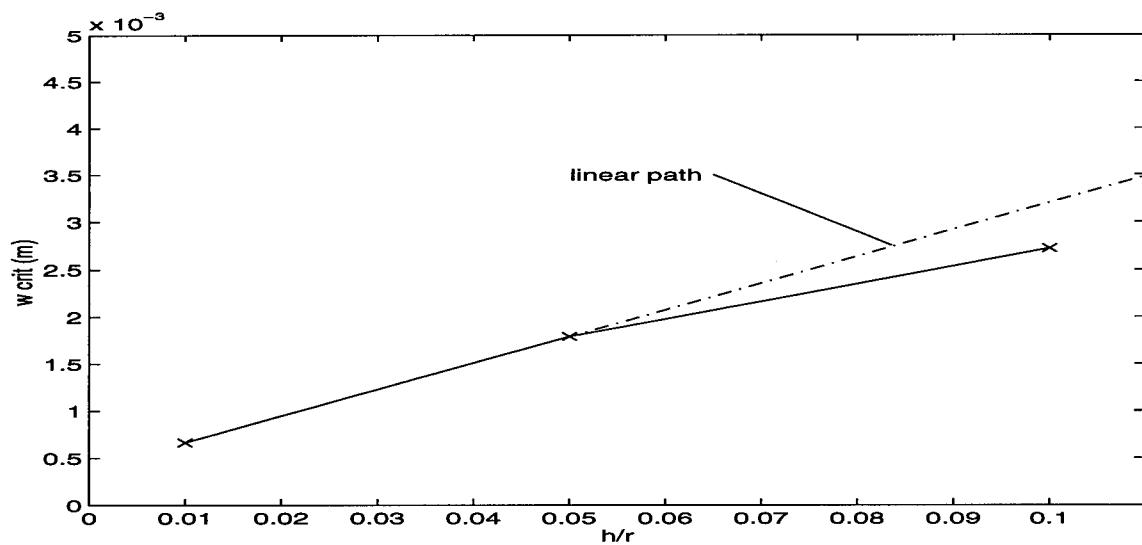


Figure 3-20 w_c vs. h/r for $S = 0.025m$

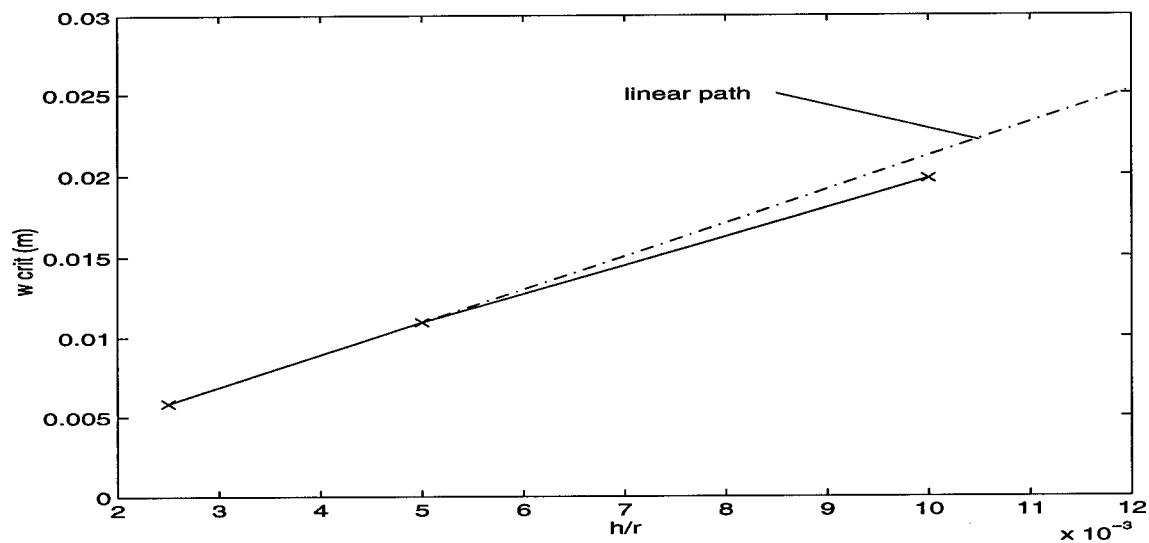


Figure 3-21 w_c vs. h/r for $S = 0.2\text{m}$

IV. Conclusions and Recommendations

4.1 Summary and Conclusions

This effort successfully modified the existing finite element formulation of a potential energy based large deformation and moderate rotation theory. Comparison with the unmodified formulation indicated proper incorporation of Hermitian shape functions for the bending angle (ψ). Despite the higher order shape functions, the rotation limit of the current work remained the same as the previous effort. Although the limits were unchanged, the current formulation did eliminate element kinking experienced in previous efforts.

Numerous arch configurations were modeled to analyze their behavior with respect to arc length and thickness variations. As the arc length decreased for a given thickness ratio (h/r), the critical load (P_c) approached infinity except for a minor decrease as the arch became flatter. This dip in the P_c vs. S curve comes about because global snapping becomes a local phenomena before snapping disappears altogether. The decrease in critical load does not represent a structural failure, but a geometry change which does not affect the load carrying capability of the arch. An arch experiencing local snapping recovers its structural integrity with only a small displacement increase. On the other hand, after global snapping, an arch experiences extremely large displacements before recovering. The extent of snapping (local or global) can be accurately predicted by examining the depth of an arch. Deep arches experience total collapse while shallow arches only undergo local snapping.

Varying the radius of curvature of an arch while holding the thickness and arc length constant demonstrated independence of depth and thickness. Compared to an analysis where through the thickness shear is neglected, these stresses decreased the critical load of both shallow and deep arches as the thickness ratio increased. Similarly, as the depth increased, nonlinearities decreased critical displacement for thick and thin beams alike. Thus, the critical load is dependent on the thickness and the critical displacement is a function of the depth of the arch.

4.2 Recommendations

The current model is limited to 45 degrees of bending rotation just as the previous effort was. This indicates the previous finite element formulation was sufficient and the underlying equilibrium equations are limiting the accuracy of the program beyond the bending rotation limits. As such, the equilibrium equations should be rederived taking the first variation of the potential energy equation with respect to the bending angle (ψ) as well as vertical displacement (w). Such a derivation could then be used with the current finite element formulation to increase bending angle limit.

Appendix A

A.1 Q_{ij} Transformations of Eqn(2-18)

$$\begin{aligned}
 \bar{Q}_{11} &= Q_{11} \cos^4 \theta + 2(Q_{12} + 2Q_{66}) \sin^2 \theta \cos^2 \theta + Q_{22} \sin^4 \theta \\
 \bar{Q}_{12} &= (Q_{11} + Q_{22} - 4Q_{66}) \sin^2 \theta \cos^2 \theta + Q_{12} (\cos^4 \theta + \sin^4 \theta) \\
 \bar{Q}_{22} &= Q_{11} \sin^4 \theta + 2(Q_{12} + 2Q_{66}) \sin^2 \theta \cos^2 \theta + Q_{22} \cos^4 \theta \\
 \bar{Q}_{16} &= (Q_{11} - Q_{12} - 2Q_{66}) \sin \theta \cos^3 \theta + (Q_{12} - Q_{22} + 2Q_{66}) \sin^3 \theta \cos \theta \\
 \bar{Q}_{26} &= (Q_{11} - Q_{12} - 2Q_{66}) \sin^3 \theta \cos \theta + (Q_{12} - Q_{22} + 2Q_{66}) \sin \theta \cos^3 \theta \\
 \bar{Q}_{66} &= (Q_{11} + Q_{22} - 2Q_{12} - 2Q_{66}) \sin^2 \theta \cos^2 \theta + Q_{66} (\cos^4 \theta + \sin^4 \theta) \\
 \bar{Q}_{44} &= Q_{55} \sin^2 \theta + Q_{44} \cos^2 \theta \\
 \bar{Q}_{45} &= (Q_{44} - Q_{55}) \cos \theta \sin \theta \\
 \bar{Q}_{55} &= Q_{44} \sin^2 \theta + Q_{55} \cos^2 \theta
 \end{aligned}$$

A.2 L, S, and H Matrices in Eqn(2-44) and Eqn(2-46)

$$\begin{aligned}
 L_0^T &= \{0 \ 1 \ -c \ 0 \ 0 \ 0 \ 0\} \\
 L_1^T &= \{0 \ 0 \ -c^2 \ 0 \ 0 \ 0 \ 1\} \\
 L_2^T &= \{0 \ -c^2 \ 0 \ 0 \ 0 \ 0 \ c\} \\
 L_3^T &= \{0 \ 0 \ 0 \ 0 \ k \ 0 \ k\} \\
 L_4^T &= \{0 \ 0 \ 0 \ 0 \ ck \ 0 \ ck\} \\
 L_5^T &= \{0 \ 0 \ 0 \ 0 \ 0 \ 0 \ 0\} \\
 L_6^T &= \{0 \ 0 \ 0 \ 0 \ 0 \ 0 \ 0\} \\
 S_1^T &= \{0 \ 0 \ 0 \ 0 \ 0 \ 0 \ 0\} \\
 S_2^T &= \{0 \ 0 \ 0 \ 3k \ 0 \ 3k \ 0\} \\
 H_0 &= \begin{bmatrix} c^2 & 0 & 0 & c & 0 & 0 & 0 \\ 0 & 1 & -c & 0 & 0 & 0 & 0 \\ 0 & -c & c^2 & 0 & 0 & 0 & 0 \\ c & 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}
 \end{aligned}$$

$$\begin{aligned}
H_1 &= \begin{bmatrix} 0 & 0 & 0 & c^2 & 0 & c^2 & 0 \\ 0 & 0 & -c^2 & 0 & 0 & 0 & 1+\psi^2 \\ 0 & -c^2 & 2c^3 & 0 & 0 & 0 & -c(1+\psi^2) \\ c^2 & 0 & 0 & 2c & 0 & c & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ c^2 & 0 & 0 & c & 0 & 0 & \psi \\ 0 & 1+\psi^2 & -c(1+\psi^2) & 0 & 0 & \psi & 0 \end{bmatrix} \\
H_2 &= \begin{bmatrix} -3c^4 & 0 & 0 & -2c^3 & 0 & c^3 & 0 \\ 0 & -3c^2 & 2c^3 & 0 & 0 & 0 & c(1+\psi^2) \\ 0 & 2c^3 & 0 & 0 & 0 & 0 & -2c^2 \\ -2c^3 & 0 & 0 & 0 & 0 & 2c^2 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ c^3 & 0 & 0 & 2c^2 & 0 & c^2 & c\psi \\ 0 & c(1+\psi^2) & -2c^2 & 0 & 0 & c\psi & 1+2\psi^2 \end{bmatrix} \\
H_3 &= \begin{bmatrix} 2c^5 & 0 & 0 & kc^2 & 0 & -2c^4 + kc^2 & 0 \\ 0 & 2c^3 & 0 & 0 & k & 0 & -2c^2 + k \\ 0 & 0 & 0 & 0 & -kc & 0 & -kc \\ kc^2 & 0 & 0 & 2kc & 0 & kc & 0 \\ 0 & k & -kc & 0 & 0 & 0 & 0 \\ -2c^4 + kc^2 & 0 & 0 & kc & 0 & 2c^3 & 0 \\ 0 & -2c^2 + k & -kc & 0 & 0 & 0 & 2c \end{bmatrix} \\
H_4 &= \begin{bmatrix} 0 & 0 & 0 & kc^3 & 0 & kc^3 & 0 \\ 0 & 0 & 0 & 0 & kc & 0 & kc \\ 0 & 0 & 0 & 0 & -2kc^2 & 0 & -2kc^2 \\ kc^3 & 0 & 0 & 4kc^2 & 0 & 3kc^2 & 0 \\ 0 & kc & -2kc^2 & 0 & 0 & 0 & k(1+\psi^2) \\ kc^3 & 0 & 0 & 3kc^2 & 0 & 2kc^2 & 0 \\ 0 & kc & -2kc^2 & 0 & k(1+\psi^2) & 0 & 2k(1+\psi^2) \end{bmatrix} \\
H_5 &= \begin{bmatrix} 0 & 0 & 0 & -2kc^4 & 0 & -2kc^4 & 0 \\ 0 & 0 & 0 & 0 & -2kc^2 & 0 & -2kc^2 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ -2kc^4 & 0 & 0 & 0 & 0 & 2kc^3 & 0 \\ 0 & -2kc^2 & 0 & 0 & 0 & 0 & 2kc(1+\psi^2) \\ -2kc^4 & 0 & 0 & -2kc^3 & 0 & 4kc^3 & 0 \\ 0 & -2kc^2 & 0 & 0 & 2kc(1+\psi^2) & 0 & 4kc \end{bmatrix}
\end{aligned}$$

$$H_6 = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & k^2 c^2 & 0 & k^2 c^2 & 0 \\ 0 & 0 & 0 & 0 & k^2 & 0 & k^2 \\ 0 & 0 & 0 & k^2 c^2 & 0 & k^2 c^2 & 0 \\ 0 & 0 & 0 & 0 & k^2 & 0 & k^2 \end{bmatrix}$$

$$H_7 = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2k^2 c^3 & 0 & 2k^2 c^3 & 0 \\ 0 & 0 & 0 & 0 & 2k^2 c & 0 & 2k^2 c \\ 0 & 0 & 0 & 2k^2 c^3 & 0 & 2k^2 c^3 & 0 \\ 0 & 0 & 0 & 0 & 2k^2 c & 0 & 2k^2 c \end{bmatrix}$$

A.3 Element Stiffness Matrices of Eqn(2-47)

$$\begin{aligned} \hat{K} = & AL_0 L_0^T + D(L_0 L_2^T + L_2 L_0^T + L_1 L_1^T) \\ & + F(L_0 L_4^T + L_4 L_0^T + L_1 L_3^T + L_3 L_1^T + L_2 L_2^T) \\ & + H(L_2 L_4^T + L_4 L_2^T + L_3 L_3^T) + JL_4 L_4^T + ASS_0 S_0^T \\ & + DS(S_0 S_2^T + S_2 S_0^T) + FSS_2 S_2^T \end{aligned}$$

$$\begin{aligned} \hat{N}_1 = & A(L_0 d^T H_0 + d^T L_0 H_0 + H_0 d L_0^T) \\ & + D(L_0 d^T H_2 + L_2 d^T H_0 + L_1 d^T H_1 + d^T L_0 H_2 + d^T L_2 H_0 + d^T L_1 H_1 + H_2 d L_0^T + H_0 d L_2^T + H_1 d L_1^T) \\ & + F\left(L_0 d^T H_4 + L_4 d^T H_0 + L_1 d^T H_3 + L_3 d^T H_1 + L_2 d^T H_2 + d^T L_0 H_4 + d^T L_4 H_0 + d^T L_1 H_3\right. \\ & \left.+ d^T L_3 H_1 + d^T L_2 H_2 + H_4 d L_0^T + H_0 d L_4^T + H_3 d L_1^T + H_1 d L_3^T + H_2 d L_2^T\right) \\ & + H\left(L_0 d^T H_6 + L_1 d^T H_5 + L_2 d^T H_4 + L_3 d^T H_3 + L_4 d^T H_2 + d^T L_0 H_6 + d^T L_1 H_5 + d^T L_2 H_4\right. \\ & \left.+ d^T L_3 H_3 + d^T L_4 H_2 + H_6 d L_0^T + H_5 d L_1^T + H_4 d L_2^T + H_3 d L_3^T + H_2 d L_4^T\right) \\ & + J\left(L_1 d^T H_7 + L_2 d^T H_6 + L_3 d^T H_5 + L_4 d^T H_4 + d^T L_1 H_7 + d^T L_2 H_6 + d^T L_3 H_5 + d^T L_4 H_4\right. \\ & \left.+ H_7 d L_1^T + H_6 d L_2^T + H_5 d L_3^T + H_4 d L_4^T\right) \\ & + L(L_3 d^T H_7 + L_4 d^T H_4 + d^T L_3 H_3 + d^T L_4 H_4 + H_7 d L_3^T + H_4 d L_4^T) \end{aligned}$$

$$\begin{aligned}
\hat{N}_2 = & A \left(H_0 d d^T H_0 + \frac{1}{2} d^T H_0 d H_0 \right) \\
& + D \left(H_0 d d^T H_2 + \frac{1}{2} H_2 d d^T H_0 + H_1 d d^T H_1 \right. \\
& \left. + \frac{1}{4} d^T H_0 d H_2 + \frac{1}{4} d^T H_2 d H_0 + \frac{1}{2} d^T H_1 d H_1 \right) \\
& + F \left(H_0 d d^T H_4 + \frac{1}{2} H_4 d d^T H_0 + H_1 d d^T H_3 + H_3 d d^T H_1 + H_2 d d^T H_2 \right. \\
& \left. + \frac{1}{4} d^T H_0 d H_4 + \frac{1}{4} d^T H_4 d H_0 + \frac{1}{2} d^T H_1 d H_3 + \frac{1}{2} d^T H_3 d H_1 + \frac{1}{2} d^T H_2 d H_2 \right) \\
& + H \left(\frac{1}{2} H_0 d d^T H_6 + \frac{1}{2} H_6 d d^T H_0 + H_1 d d^T H_5 + H_5 d d^T H_1 + H_2 d d^T H_4 + H_4 d d^T H_2 + H_3 d d^T H_3 \right. \\
& \left. + \frac{1}{4} d^T H_0 d H_6 + \frac{1}{4} d^T H_6 d H_0 + \frac{1}{2} d^T H_1 d H_5 + \frac{1}{2} d^T H_5 d H_1 + \frac{1}{2} d^T H_2 d H_4 + \frac{1}{2} d^T H_4 d H_2 + \frac{1}{2} d^T H_3 d H_3 \right) \\
& + J \left(H_1 d d^T H_7 + H_7 d d^T H_1 + H_2 d d^T H_6 + H_6 d d^T H_2 + H_3 d d^T H_5 + H_5 d d^T H_3 + H_4 d d^T H_4 \right. \\
& \left. + \frac{1}{2} d^T H_1 d H_7 + \frac{1}{2} d^T H_7 d H_1 + \frac{1}{2} d^T H_2 d H_6 + \frac{1}{2} d^T H_6 d H_2 + \frac{1}{2} d^T H_3 d H_5 + \frac{1}{2} d^T H_5 d H_3 + \frac{1}{2} d^T H_4 d H_4 \right) \\
& + L \left(H_3 d d^T H_7 + H_7 d d^T H_3 + H_4 d d^T H_6 + H_6 d d^T H_4 + H_5 d d^T H_5 \right. \\
& \left. + \frac{1}{2} d^T H_3 d H_7 + \frac{1}{2} d^T H_7 d H_3 + \frac{1}{2} d^T H_4 d H_6 + \frac{1}{2} d^T H_6 d H_4 + \frac{1}{2} d^T H_5 d H_5 \right) \\
& + R \left(H_5 d d^T H_7 + H_7 d d^T H_5 + H_6 d d^T H_6 \right. \\
& \left. + \frac{1}{2} d^T H_5 d H_7 + \frac{1}{2} d^T H_7 d H_5 + \frac{1}{2} d^T H_6 d H_6 \right) \\
& + T \left(H_7 d d^T H_7 + \frac{1}{2} d^T H_7 d H_7 \right)
\end{aligned}$$

Appendix B: Critical Load Comparison for Constant Thickness Ratio and
Variable Arc Length

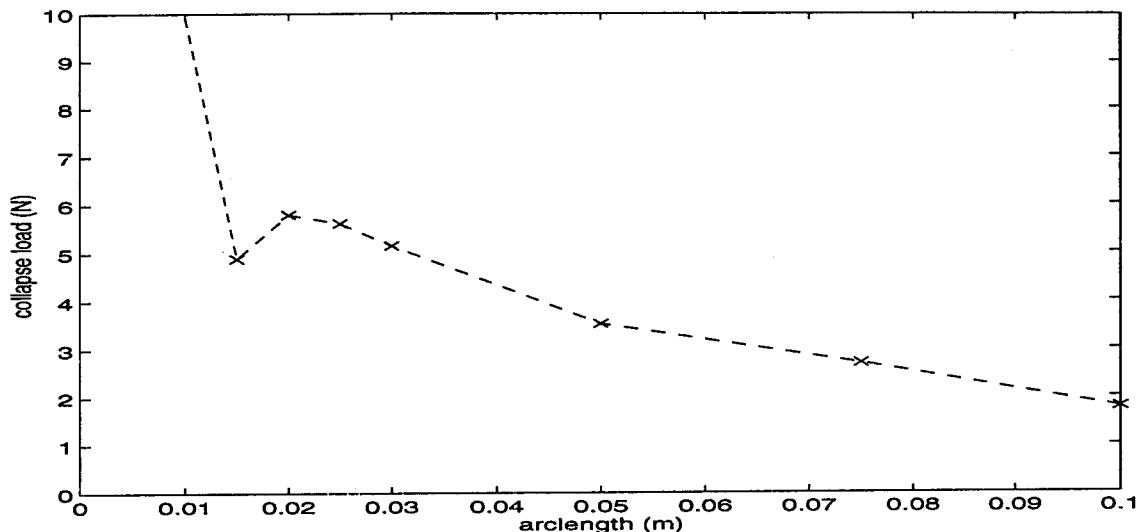


Figure B-1 Critical Load Comparison for $h/r = 0.05$

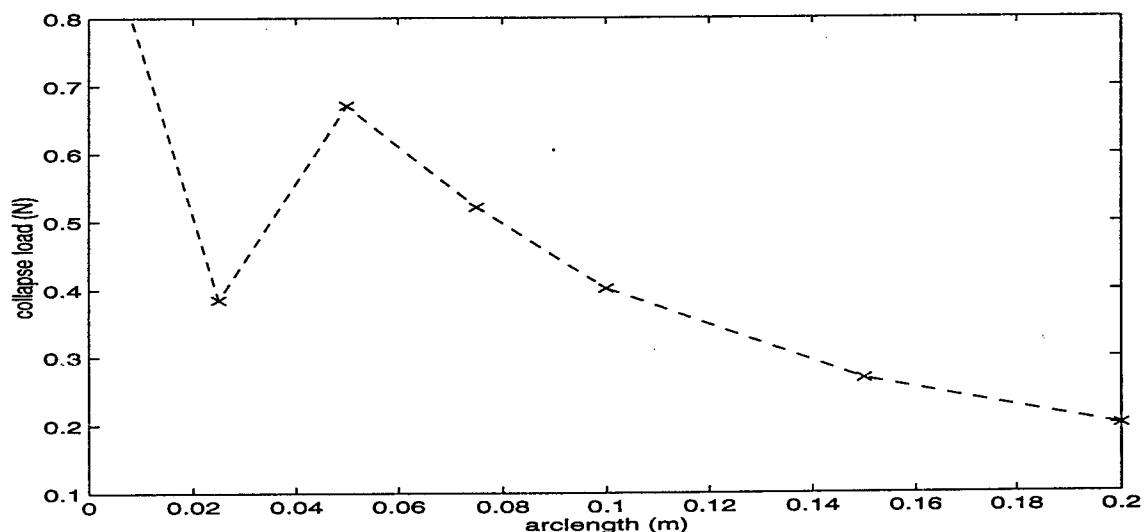


Figure B-2 Critical Load Comparison for $h/r = 0.01$

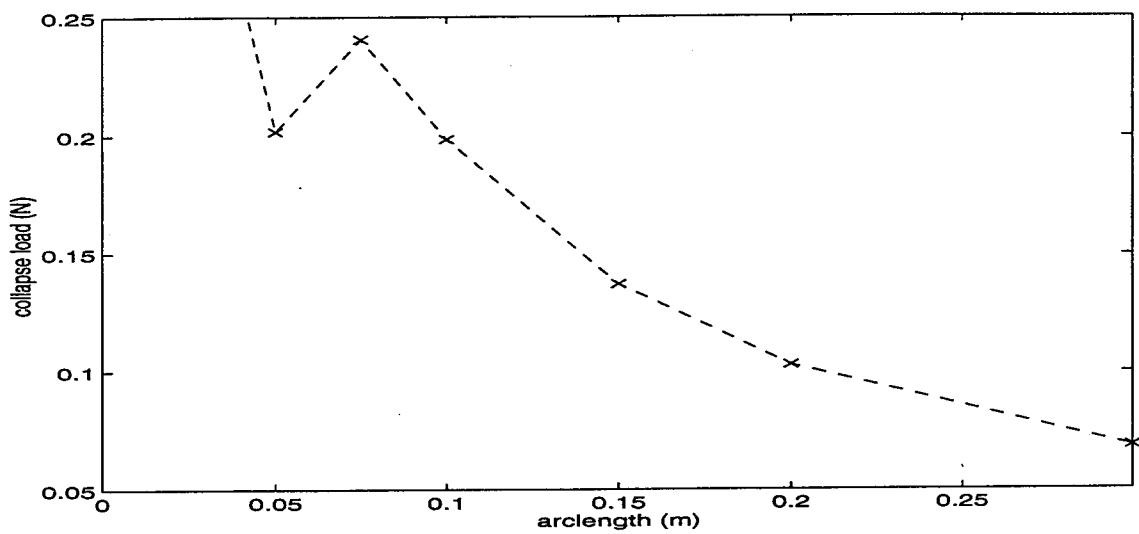


Figure B-3 Critical Load Comparison for $h/r = 0.005$

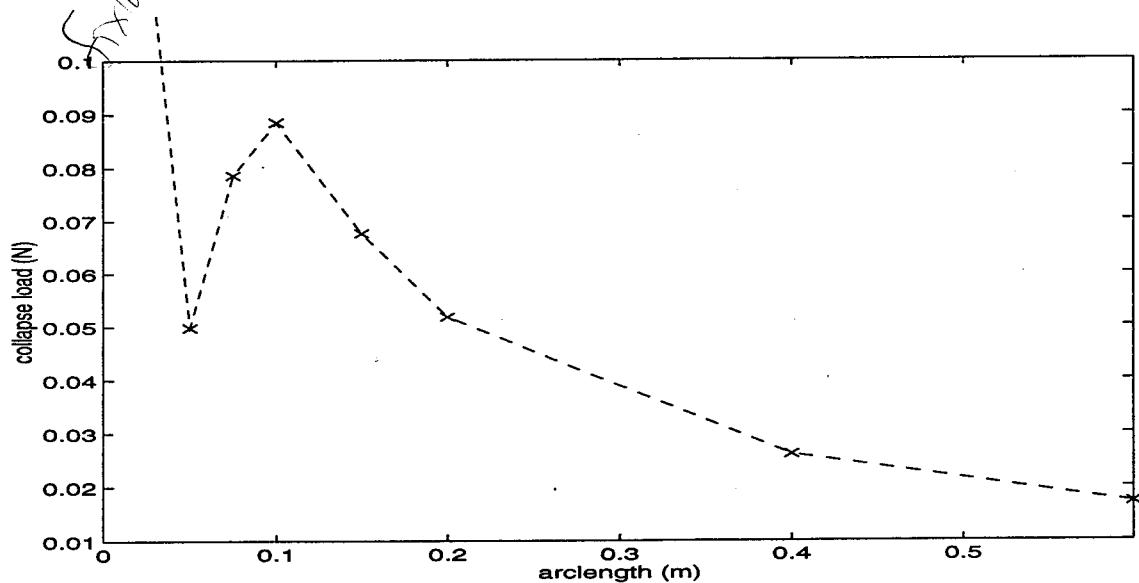


Figure B-4 Critical Load Comparison for $h/r = 0.0025$

Appendix C: Critical Load and Critical Displacement Comparisons for
Constant Arc Length and Variable Thickness Ratio

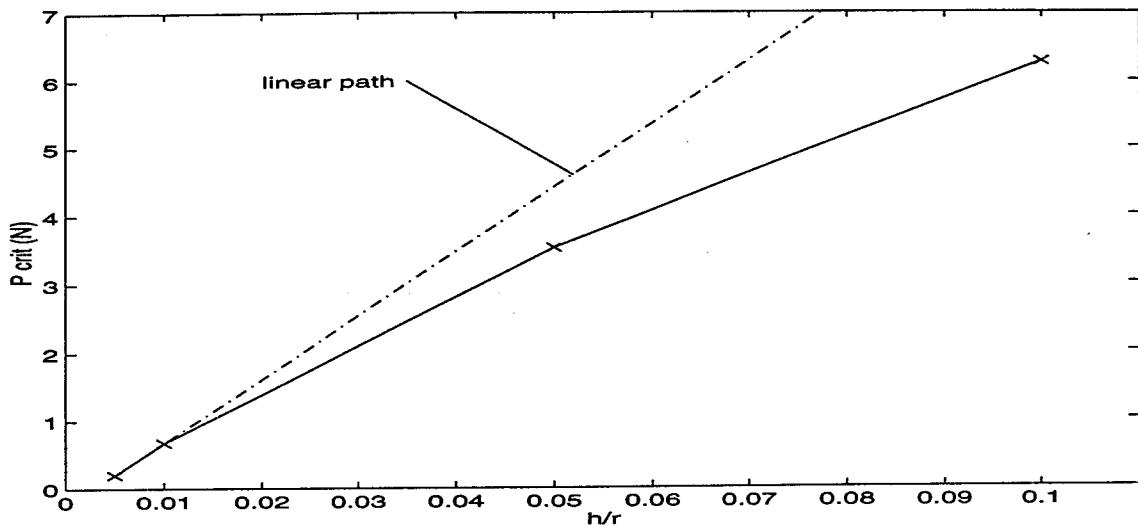


Figure C-1 P_c vs. h/r for $S = 0.05m$

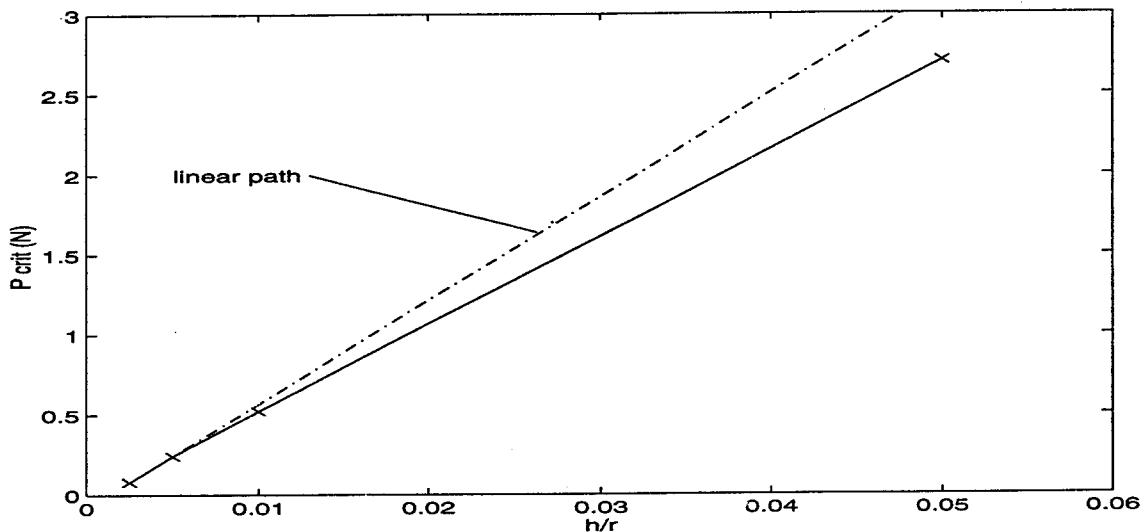


Figure C-2 P_c vs. h/r for $S = 0.075m$

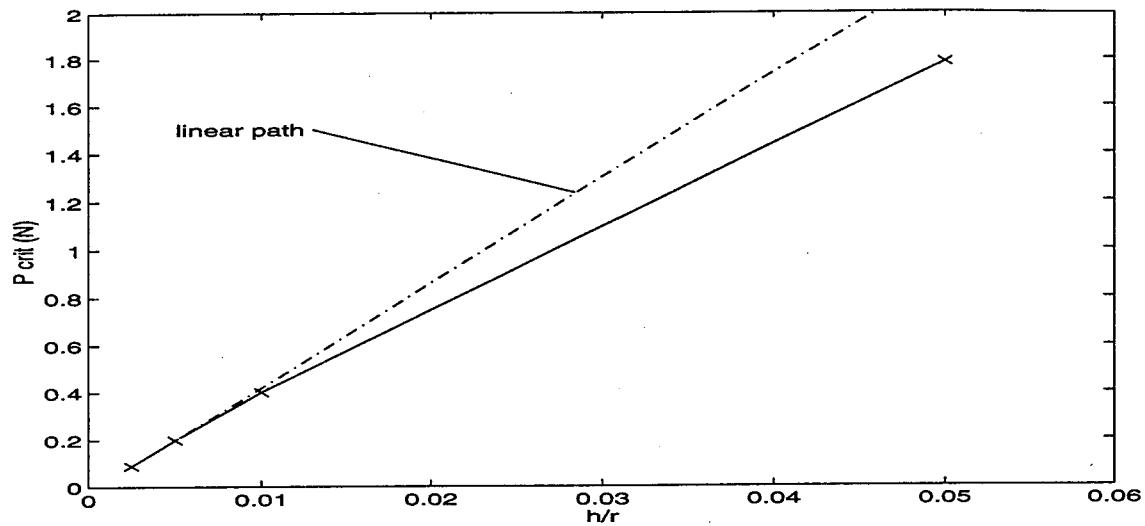


Figure C-3 P_{crit} vs. h/r for $S = 0.1m$

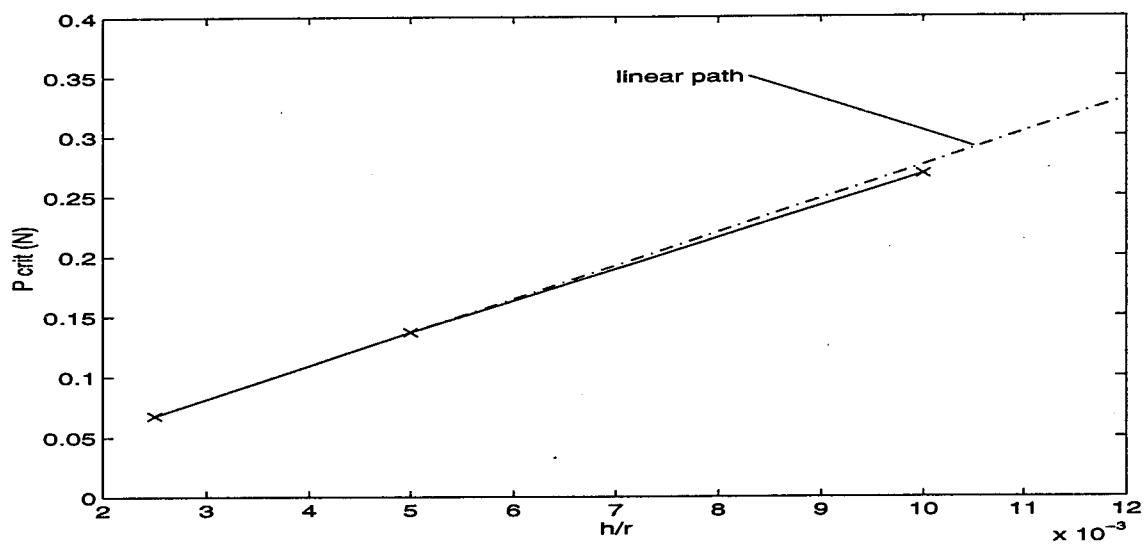


Figure C-4 P_{crit} vs. h/r for $S = 0.15m$

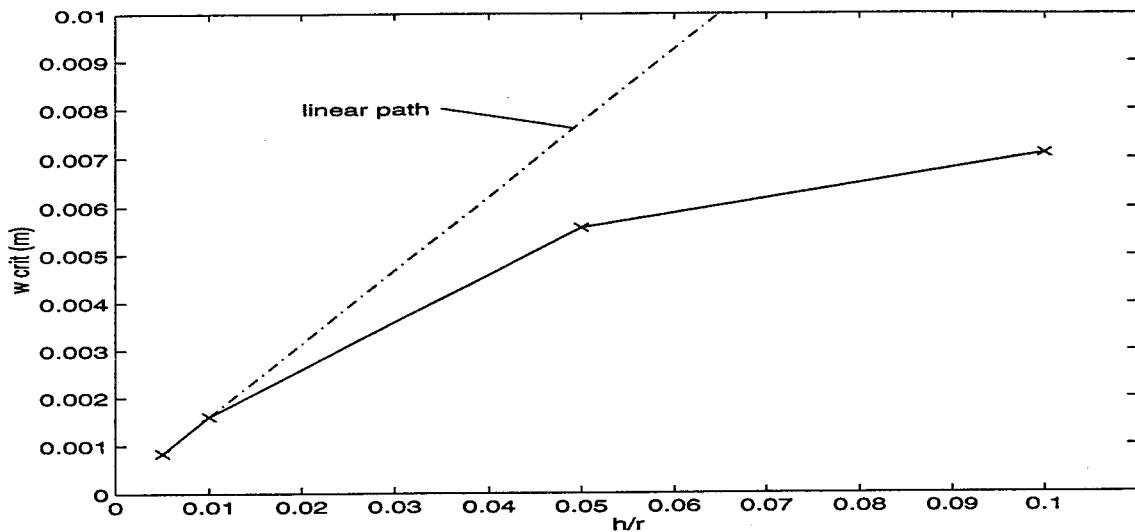


Figure C-5 w_c vs. h/r for $S = 0.05m$

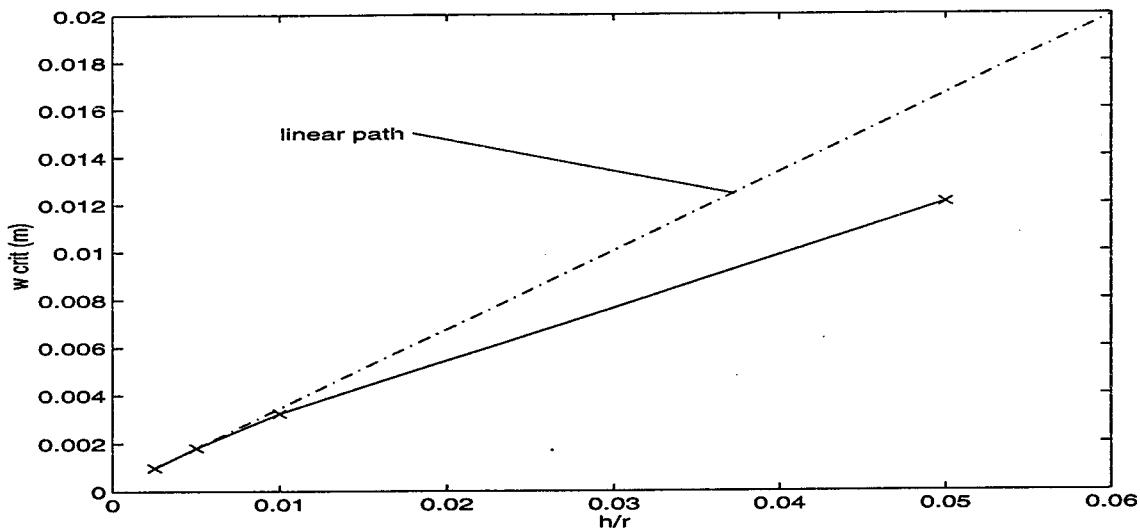


Figure C-6 w_c vs. h/r for $S = 0.075m$

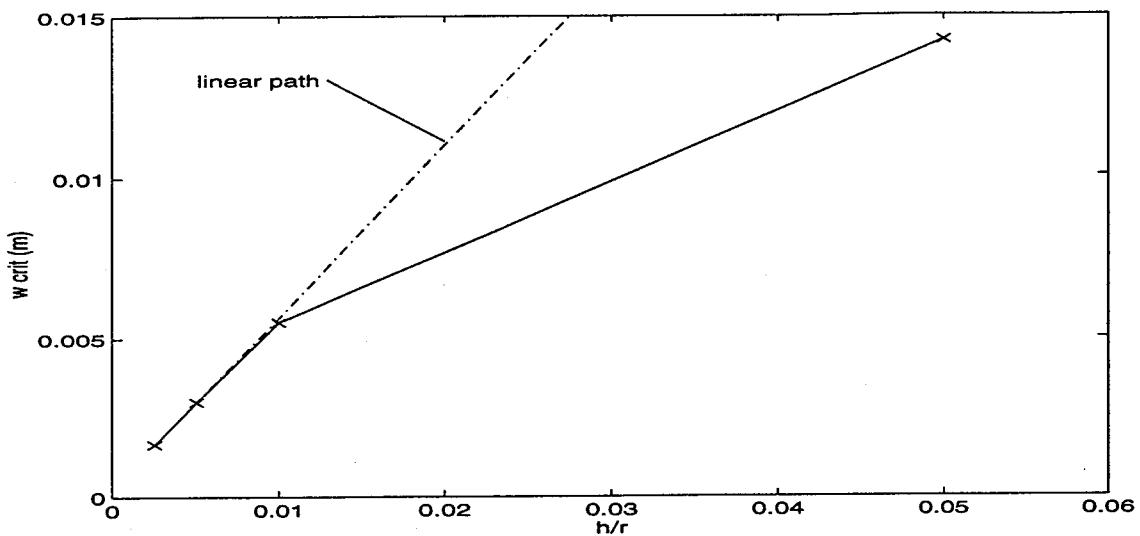


Figure C-7 w_c vs. h/r for $S = 0.1\text{m}$

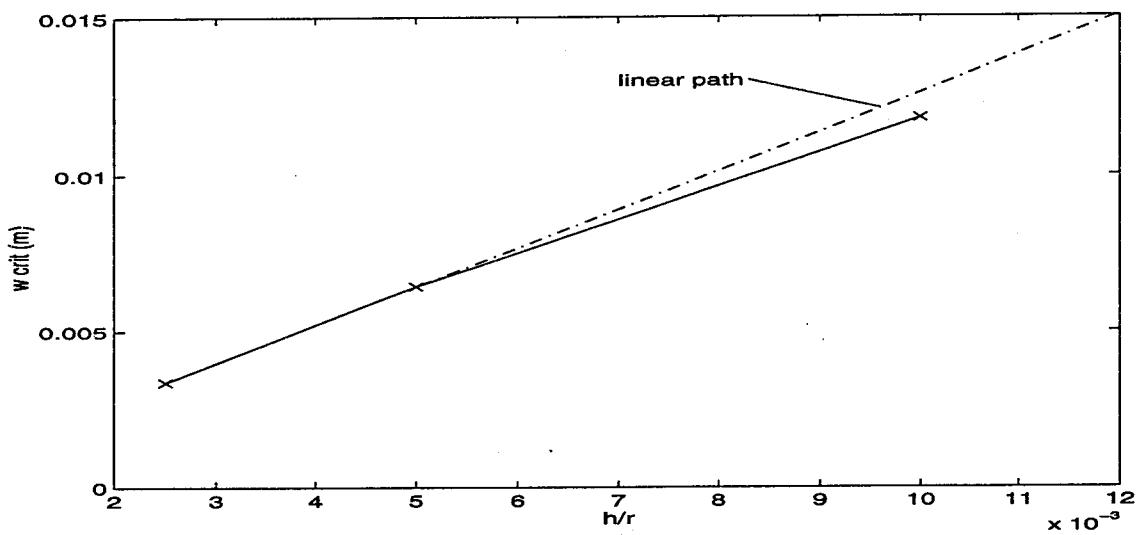


Figure C-8 w_c vs. h/r for $S = 0.15\text{m}$

Appendix D: FORTAN Program Description

D.1 Overview

The following program was initially developed by Creaghan and subsequently modified by Miller (3; 13). The attached listing does not include all of the subroutines, but only the ones modified for the current work. The majority of the modifications were done to incorporate the shape function changes presented in Chapter II. Some minor changes were also incorporated to facilitate multiple simultaneous simulations.

D.2 Data Input Format

This is a line by line description of the user constructed, problem specific data input file required by the program. Each section represents a line and each variable on a line is separated by a comma. Variables are identified by italics.

1. *title*: problem title text string
2. *linear, isotro, isarch, ishape*:
 - a. *linear*: 0 for nonlinear, 1 for linear
 - b. *isotro*: 0 for composite, 1 for isotropic
 - c. *isarch*: 0 for straight beam, 1 for circular arch
 - d. *ishape*: 0 if half a symmetric arch is modeled, 1 for a full arch model
3. *inctyp, ninc, imax, kupdate, tol*:
 - a. *inctyp*: 1 for displacement control, 2 for Riks method
 - b. *ninc*: number of increments (load or displacement) desired
 - c. *imax*: maximum number of iterations per increment

- d. *kupdate*: not used, fill with 0 or 1
- e. *tol*: convergence tolerance for an increment

4. *pincr, eiter, ttpi*: include only for Riks method nonlinear solutions

- a. *pincr*: initial load parameter, λ_0
- b. *eiter*: estimated number of iterations per increment, N_s not an integer
- c. *ttpi*: maximum load change for an iteration (λ_{max})

5. *table(ninc)*: include only for nonlinear displacement control; this is a list of *ninc* numbers used to attain the incremental global displacement

6. *nelem*: number of elements

7. *delem(nelem)*: list of element lengths; 1 for each element

8. *nbndry*: number of nodes with specified boundary condition (BC)

9. *nbound(nbndry, 6)(v, Ψ , Ψ_2 , w, w_2)*: one line for each node with a specified BC the first number is the node number; the next five numbers describe whether or not the dof for the specified node are free or specified in the order indicated; 1=prescribed, 0=free.

10. *vbond(ii)*: prescribed displacements for the fixed dof indicated on the previous line

11. *ldtyp, distld*: not used; fill with 0,0,0

12. *nconc*: number of concentrated loads or moments; Riks needs at least 1.

13. *inconc(nconc)*: use only if *nconc*>0; dof numbers for concentrated loads. middle node dof counts here (11 dof per element) in the following order:

$$v^1, \Psi^1, \Psi_2^1, w^1, w_2^1, v^3, v^2, \Psi^2, \Psi_2^2, w^2, w_2^2$$

14. *vconc*: use only if *nconc*>0; values of loads specified on line 13

15. *ey, nu, ht, width*: include for isotropic material

- a. *ey*: Young's modulus
- b. *nu*: Poisson's ratio
- c. *ht*: thickness
- d. *width*: beam or arch width

16. *e1, e2, g12, nu12, g13, g23, width*: include for composite materials

- a. *e1*: Young's modulus along fibers
- b. *e2*: Young's modulus transverse to fibers
- c. *g12*: shear modulus g_{12}
- d. *nu12*: Poisson's ratio ν_{12}
- e. *g13*: shear modulus g_{13}
- f. *g23*: shear modulus g_{23}
- g. *width*: beam or arch width

17. *nplies, pthick*: include for composite materials

- a. *nplies*: number of plies
- b. *pthick*: individual ply thickness; same for all

18. *theta(nplies)*: include for composites; list of ply orientation angles in degrees

19. *rad*: include for curved arch; arch radius of curvature

20. *nforc*: number of nodal resultant forces to be calculated

21. *iforce(nforce)*: include only for $nforce > 0$; dof numbers for force calculations, include middle node in calculation

22. *nstres*: not used; fill with 0

D.3 Program Listing

```
program beam
c
c      See bottom of file for variable and subroutine listing.
c      This version is n1 n2 updated to include tangent function and
c      hermitian shape functions for the rotation d.o.f.
c      remember -e compile option (will give 132 character lines)
c      this version is also prompts for plot and shape file names
c      implicit double precision (a-h,o-z)
c      character*64 gname, fname, pname, sname
c
c      common/chac/gname, fname
c
c      common/elas/ae, de, fe, he, ej, el, re, te, as, ds, fs
c      common/input/tol, table(250), delem(250), vbound(2500), distld,
c      . vconc(2500), ey, enu, ht, e1, e2, g12, enu12, enu21, g13, g23, pthick,
c      . rad, linear, isotro, isearch, ishape, inctyp, ninc, imax,
c      . nelem, nbndry, nbound(250, 6), ldtyp, nconc, iconc(2500),
c      . nplies, nforc, iforc(2500), nstres, istres(250), ibndry(2500),
c      . theta(20), idload(250), coord(251), width, nnod, pincr, eiter, ttpi
c
c      common/stf/stif(11,11), elp(11), eln(11,11), eld(11)
c
c      common/proc/gstif(2500,11), gn(2500,11), gf(2500), gd(2500), vperm(250
0),
c      . vpres(2500)
c
c      call rinput(pname, sname)
c      call elast
c      if(inctyp.eq.1)call proces(pname, sname)
c      if(inctyp.eq.2)call rikspr(pname, sname)
c*****
c
c      VARIABLES FOR BSHELL
c
c      fname input file
c      gname output file
c      ae, de,      elasticity terms
c      fe, he,      elasticity terms
c      ej, el,      elasticity terms
c      re, te,      elasticity terms
c      as, ds,      elasticity terms
c      fs      elasticity term
c      ey      Young's modulus for isotropic case
c      enu     Poisson's ratio for isotropic case
c      ht      thickness of beam for isotropic case
c      e1, e2,      laminate material properties
c      g12, enu12,  "
c      enu21, g13,  "
c      g23      "
c      pthick     laminate ply thickness
c      nplies     number of plies in laminate
c      theta(20)   ply orientation angles
c      tol       convergence tolerance, percent
c      table(250)  displacement increment multiplicative
c                  factors
c      delem(250)  element lengths
c      vbound(2500) values of prescribed displacement boundary
```

```

c           conditions
c distld    distributed load intensity
c vconc(2500) concentrated load values
c rad      arch radius of curvature
c linear    =1 for linear analysis, =0 for nonlinear
c isotro    =1 for isotropic, =0 for laminate
c isarch    =1 for arch, =0 for straight beam
c ishape    =1 to print x,y coordinates for each node at each
increment
c           when a full arch is represented output to file pname
c inclod    =1 to increment load(NA), =0 increment displacement
c ninc      total number of displacement increments
c imax      maximum number of iterations per increment
c nelem     total number of elements in model
c nbndry    number of nodes with specified boundary conditions
c nbound(250,6) array of node numbers followed by 1's for
c           fixed b.c.'s, zeros for unfixed
c ldtyp =1 for distributed load, =0 no distributed load
c nconc total number of concentrated loads input
c iconc(2500) DOF's for specified loads
c nforc number of forces(including moments)to be solved for
c iforc(2500) DOF's at which to calculate forces
c nstres    number of elements for stress calculation
c istres(250) element #'s for stress calculation
c ibndry(2500) DOF numbers for b.c.'s
c idload(250) elements with distributed load
c coord(251) coordinate of the nodes
c width     beam or arch width
c nnod      number of nodes
c ****
c ****
c
c SUBROUTINES FOR BSHELL
c
c rinput    reads in and echos input data
c elast     computes elasticity terms
c proces    drives the solution algorithm for displacement control
c rikspr(pname,sname) drives the solution algorithm for Riks method
c stiff     manages stiffness matrix computations
c shape     computes shape function array dsf
c beamk    computes constant stiffness array bmk
c beamn1   computes linear stiffness array bmn1
c beamn2   computes quadratic stiffness array bmn2
c bndy     applies displacement boundary conditions
c solve    solves simultaneous equations in banded array format
c converge  checks solutions for convergence
c postpr   computes nodal loads and sends to output file
c
c end
c
c
c subroutine rinput(pname,sname)
c
c character*64 fname,gname
c character*4 title
c dimension title(20)
c implicit double precision (a-h,o-z)
c
c common/chac/gname,fname

```

```

common/input/tol,table(250),delem(250),vbound(2500),distld,
. vconc(2500),ey,enu,ht,e1,e2,g12,enu12,g13,g23,pthick,
. rad,linear,isotro,isarch,ishape,inctyp,ninc,imax,
. nelem,nbndry,nbound(250,6),ldtyp,nconc,iconc(2500),
. nplies,nforc,iforc(2500),nstres,istres(250),ibndry(2500),
. theta(20),idload(250),coord(251),width,nnod,pincr,eiter,ttpi

c
write(*,1000)
read(*,1005) fname
write(*,1010)
read(*,1005) gname
write(*,1011)
read(*,1005) pname
write(*,1012)
read(*,1005) sname
open(5,file=fname)
open(6,file=gname,status='new')
read(5,1015) title
read(5,*) linear,isotro,isarch,ishape
read(5,*) inctyp,ninc,imax,kupdte,tol
if(linear.eq.0.and.inctyp.eq.2)read(5,*)pincr,eiter,ttpi
if(linear.eq.0.and.inctyp.eq.1)read(5,*)(table(i),i=1,ninc)
read(5,*)nelem
read(5,*)(delem(i),i=1,nelem)

c
c calculate nodal coordinates
c
nnod=nelem+1
coord(1)=0.0
do 5 ii=2,nnod
5 coord(ii)=coord(ii-1)+delem(ii-1)
read(5,*)nbndry
do 10 i=1,nbndry
10 read(5,*)(nbound(i,j),j=1,6)
ifdof=0

c
c ifdof=counter for enumber of fixed dof's
c
do 20 i=1,nbndry
do 20 j=2,6
if(nbound(i,j).eq.0)goto 20
ifdof=ifdof+1
ibndry(ifdof)=(nbound(i,1)-1)*6 + (j-1)
20 continue
read(5,*)(vbound(i),i=1,ifdof)
read(5,*)ldtyp,distld
if(ldtyp.eq.1)read(5,*)ndload
if(ldtyp.eq.1)read(5,*)(idload(i),i=1,ndload)
read(5,*)nconc
if(nconc.ne.0)read(5,*)(iconc(i),i=1,nconc)
if(nconc.ne.0)read(5,*)(vconc(i),i=1,nconc)
if(isotro.eq.1)read(5,*)ey,enu,ht,width
if(isotro.eq.0)read(5,*)e1,e2,g12,enu12,g13,g23,width
if(isotro.eq.0)read(5,*)nplies,pthick
if(isotro.eq.0)read(5,*)(theta(i),i=1,nplies)
if(isarch.eq.1)read(5,*)rad
read(5,*)nforc
if(nforc.ne.0)read(5,*)(iforc(i),i=1,nforc)
read(5,*)nstres
if(nstres.ne.0)read(5,*)(istres(i),i=1,nstres)

```

```

c
c Echo the input to the output file
c
write(6,1015)title
if(isarch.eq.1)write(6,1020)
if(isarch.eq.0)write(6,1025)
if(linear.eq.1)write(6,1030)
if(linear.eq.0)write(6,1035)
if(isotro.eq.1)write(6,1040)
if(isotro.eq.0)write(6,1045)
if(ishape.eq.1)write(6,1050)
if(inctyp.eq.1)write(6,1060)
if(inctyp.eq.2)write(6,1055)
write(6,1065)ninc
write(6,1070)imax
write(6,1075)tol
if(inctyp.eq.2)write(6,1076)pincr,eiter,ttpi
if(inctyp.eq.1)write(6,1078)
if(inctyp.eq.1)write(6,1080)(table(i),i=1,ninc)
write(6,1085)nelem
write(6,1090)
write(6,1095)(coord(i),i=1,nnod)
write(6,1100)
write(6,1105)
do 30 i=1,nbndry
30 write(6,1110)(nbond(i,j),j=1,6)
write(6,1115)ifdof
write(6,1120)(ibndry(i),i=1,ifdof)
write(6,1095)(vbound(i),i=1,ifdof)
if(ldtyp.eq.1)write(6,1125)distld
if(ldtyp.eq.1)write(6,1130)(idload(i),i=1,ndload)
if(nconc.ne.0)write(6,1135)
if(nconc.ne.0)write(6,1120)(iconc(i),i=1,nconc)
if(nconc.ne.0)write(6,1095)(vconc(i),i=1,nconc)
if(isotro.eq.1)write(6,1140)ey,enu,ht,width
if(isotro.eq.0)write(6,1145)e1,e2,g12,enu12,g13,g23,width
if(isotro.eq.0)write(6,1150)nplies,pthick
if(isotro.eq.0)write(6,1155)(theta(i),i=1,nplies)
if(isarch.eq.1)write(6,1160)rad
write(6,1165)(iforc(i),i=1,nforc)
write(6,1170)(istres(i),i=1,nstres)
close(5)
close(6)
c
c
c F O R M A T S
c
1000 format('Enter your input file name.')
1005 format(A)
1010 format('Enter your output file name.')
1011 format('Enter your plot file name.')
1012 format('Enter your shape file name.')
1015 format(20a4)
1020 format(//,1x,'Element type: arch')
1025 format(//,1x,'Element type: straight beam')
1030 format(//,1x,'Analysis type: linear')
1035 format(//,1x,'Analysis type: nonlinear')
1040 format(//,1x,'Material type: isotropic')
1045 format(//,1x,'Material type: laminate')
1050 format(//,1x,'Printout of nodal x,y coordinates requested')

```

```

1055 format(/,1x,'Riks method specified')
1060 format(/,1x,'Displacement control method specified')
1065 format(/,1x,'Increments specified:',2x,i3)
1070 format(/,1x,'Maximum iterations specified:',2x,i3)
1075 format(/,1x,'Percent convergence tolerance:',2x,d12.5)
1076 format(/,1x,'pincr=',2x,d12.5,2x,'eiter=',2x,d12.5,2x,
.          'ttpi=',2x,d12.5)
1078 format(/,1x,'Displacement Increment Table')
1080 format(8(2x,d12.5))
1085 format(/,1x,'Number of elements:',2x,i3)
1090 format(/,1x,'Nodal Coordinates:')
1095 format(8(2x,d12.5))
1100 format(/,1x,'DISPLACEMENT BOUNDARY CONDITIONS, 1=PRESCRIBED,
.          X0=FREE')
1105 format(/,4X,'NODE      V    PSI-S  PSI-SS    W    W-S   ')
1110 format(4x,i4,3x,5(i3,3x))
1115 format(/,1x,'NUMBER OF PRESCRIBED DISPLACEMENTS:',
.          i5,/,1x,'SPECIFIED DISPLACEMENT DOF AND THIER
.          VALUES FOLLOW:')
1120 format(16i5)

1125 format(/,1x,'Distributed Load Intensity:',2x,d12.5)
1130 format(/,1x,'Elements with distributed load:',//,1x,16i5)
1135 format(/,1x,'DOF and specified concentrated loads follow:')
1140 format(/,1x,'Isotropic material properties ey, enu, ht, width:'
.          ,/,1x,4d12.5)
1145 format(/,1x,'Composite material properties e1, e2, g12, enu12,
.          g13, g23, width://,1x,7d12.5)
1150 format(/,1x,'Number of plies:',2x,i3,2x,'Ply thickness:',2x,
.          d12.5)
1155 format(/,1x,'Ply orientation angles://,1x,8(2x,d12.5))
1160 format(/,1x,'Radius of curvature:',2x,d12.5)
1165 format(/,1x,'DOFs for equivalent load calculation://,
.          1x,16i5)
1170 format(/,1x,'Elements for stress calculation://,1x,16i5)
1175 format(/,1x,i5)
      return
      end

c
c
c
c      subroutine elast
c
c      implicit double precision (a-h,o-z)
c
c      dimension qbar(3,3),rtheta(20)
c
c
c      character*64 gname, fname
c      common/chac/gname, fname
c      common/elas/ae, de, fe, he, ej, el, re, te, as, ds, fs
c      common/input/tol, table(250), delem(250), vbound(2500), distld,
.      vconc(2500), ey, enu, ht, e1, e2, g12, enu12, enu21, g13, g23, pthick,
.      rad, linear, isotro, isearch, ishape, inctyp, ninc, imax,
.      nelem, nbndry, nbound(250,6), ldtyp, nconc, iconc(2500),
.      nplies, nforc, iforc(2500), nstres, istres(250), ibndry(2500),
.      theta(20), idload(250), coord(251), width, nnod, pincr, eiter, ttpi
c
c
c      Isotropic case

```

```

c
c      write(6,1000)e1,e2,g12,enu12,enu21,g13,g23,pthick
c      if(isotro.eq.0)goto 100
c      gs=ey/(2*(1+enu))
c          denom=1.-enu**2
c          q11=ey/denom
c      q12=enu*ey/denom
c      q22=q11
c      q2hat=q22-(q12**2/q11)
c      qs4=gs
c      ae=q2hat*ht
c          de=q2hat*ht**3/(3*2.**2)
c          fe=q2hat*ht**5/(5*2.**4)
c          he=q2hat*ht**7/(7*2.**6)
c          ej=q2hat*ht**9/(9*2.**8)
c          el=q2hat*ht**11/(11*2.**10)
c          re=q2hat*ht**13/(13*2.**12)
c      te=q2hat*ht**15/(15*2.**14)
c      as=qs4*ht
c      ds=qs4*ht**3/(3*2.**2)
c      fs=qs4*ht**5/(5*2.**4)
c      goto 200
c
c      Laminate case
c
100  ht=pthick*nplies
      enu21=e2*enu12/e1
      denom=1.-enu12*enu21
      q11=e1/denom
      q12=enu12*e2/denom
      q22=e2/denom
c
c***** calculate the elasticity matrices *****
c      calculate the elasticity matrices
c
c      remem that the z axis points down,
c      however, the first ply is the top ply, ie,
c      the ply with the most negative z !!!
c***** initialize elasticity terms *****
c
c      initialize elasticity terms
c
      ae=0.
      de=0.
      fe=0.
      he=0.
      ej=0.
      el=0.
      re=0.
      te=0.
      as=0.
      ds=0.
      fs=0.
      do 45 ii=1,nplies
45      rtheta(ii)=theta(ii)*3.14159265/180.
      do 50 kk=1,nplies
      qbar(1,1)=q11*(cos(rtheta(kk))**4)+2*q12*(sin(rtheta(kk))**2)*
.      (cos(rtheta(kk))**2)+q22*(sin(rtheta(kk))**4)

```

```

qbar(1,2)=(q11+q22)*(sin(rtheta(kk))**2)*(cos(rtheta(kk))**2) +
. q12*(sin(rtheta(kk))**4+cos(rtheta(kk))**4)

qbar(2,2)=q11*(sin(rtheta(kk))**4)+2*q12*(sin(rtheta(kk))**2)*
. (cos(rtheta(kk))**2)+q22*cos(rtheta(kk))**4
. qs4=g13*dcos(rtheta(kk))**2+g23*dsin(rtheta(kk))**2
q2hat=qbar(2,2)-(qbar(1,2)**2/qbar(1,1))
z1=(kk*1. - nplies*.5)*pthick
zu=z1-pthick
ae=ae + q2hat*pthick
de=de + q2hat*(z1**3-zu**3)/3.
fe=fe + q2hat*(z1**5-zu**5)/5.
he=he + q2hat*(z1**7-zu**7)/7.
ej=ej + q2hat*(z1**9-zu**9)/9.
el=el + q2hat*(z1**11-zu**11)/11.
re=re + q2hat*(z1**13-zu**13)/13.
te=te + q2hat*(z1**15-zu**15)/15.
as=as+qs4*pthick
ds=ds+qs4*(z1**3-zu**3)/3.
fs=fs+qs4*(z1**5-zu**5)/5.
50 continue
c 200 open(6,fiel=gname,status='old')
200 write(6,1000)ae,de,fe,he,ej,el,re,te,as,ds,fs
c  close(6)
1000 format(//,1x,'Elasticity terms:',//,1x,8(2x,d12.5))
      return
      end
c
c
c
      subroutine proces(pname,sname)
c
      implicit double precision (a-h,o-z)
c
      character*64 gname, fname
c
c
      common/chac/gname, fname
c
c
      common/elas/ae,de,fe,he,ej,el,re,te,as,ds,fs
c
      common/input/tol,table(250),delem(250),vbound(2500),distld,
. vconc(2500),ey,enu,ht,e1,e2,g12,enu12,enu21,g13,g23,pthick,
. rad,linear,isotro,isarch,ishape,inctyp,ninc,imax,
. nelem,nbndry,nbound(250,6),ldtyp,nconc,iconc(2500),
. nplies,nforc,iforc(2500),nstres,istres(250),ibndry(2500),
. theta(20),idload(250),coord(251),width,nnod,pincr,eiter,ttpi
      common/stf/stif(11,11),elp(11),eln(11,11),eld(11)
c
      common/proc/gstif(2500,11),gn(2500,11),gf(2500),gd(2500),vperm(250
0),
. vpres(2500)
c
      ndof=nnod*5+nelem
      ncount=1
      ictcount=1
      do 1 ii=1,ndof
1      gd(ii)=0.0d0
      do 2 ii=1,nbndry*6

```

```

        vpres(ii)=0.0d0
2      vperm(ii)=vbound(ii)
c
c      start new increment or iteration/
c      zero out global stiffness matrices and global force
c      vector
c
3      do 5 ii=1,ndof
        gf(ii)=0.0d0
        do 5 jj=1,11
          gstif(ii,jj)=0.0d0
5      gn(ii,jj)=0.0d0
        kcall=0
c
c      increment prescribed displacement for displacement control
c
        if(linear.eq.1)goto 9
        if(icount.ne.1)goto 9
        do 7 ii=1,nbndry*6
          if(ncount.eq.1)vbound(ii)=vperm(ii)*table(1)
7      if(ncount.gt.1)vbound(ii)=vperm(ii)*(table(ncount)-
              table(ncount-1))
c
c      loop over all elements for stiffness and forces
c
9      do 30 ielem=1,nelem
        do 10 ii=1,11
10      eld(ii)=gd(ii+(ielem-1)*6)
c
        kcall=kcall+1
        call stiff(ielem,icount,ncount,kcall)
c
c      Assemble global stiffness array, gstif, global equilibrium
c      stiffness, gn, in banded form. Half-bandwidth=11. Also
c      assemble global force vector, gf.
c
        nr=(ielem-1)*6 + 1
        do 30 jj=0,10
          gf(nr+jj)=gf(nr+jj)+elp(jj+1)
        do 30 kk=1,11-jj
          gstif(nr+jj,kk)=gstif(nr+jj,kk)+stif(jj+1,kk+jj)
        if(linear.eq.1)goto 30
        if(icount.eq.1 .and. ncount.eq.1)goto 30
        gn(nr+jj,kk)=gn(nr+jj,kk)+eln(jj+1,kk+jj)
30      continue
c
c      impose force boundary conditions
c      at this point, gf=R
c
        if(nconc.eq.0)goto 45
        do 40 ii=1,nconc
          nb=iconc(ii)
40      gf(nb)=gf(nb)+vconc(ii)
45      continue
c
c      calculate the residual force vector for nonlinear
c      analysis. -[gn]*{gd}+R=-[k+n1/2+n2/3]*{q}+R=gf
c
        if(icount.eq.1)goto 65
        do 60 ii=1,ndof

```

```

      add=0.
      do 50 kk=1,ii-1
      if(ii-kk+1 .gt. 11)goto 50
      add=add+gn(kk,ii-kk+1)*gd(kk)
      continue
      res=0.
      do 55 jj=1,11
      if(jj+ii-1 .gt. ndof)goto 55
      res=res + gn(ii,jj)*gd(jj+ii-1)
      55 continue

c
c      add to existing gf which already contains R
c
      gf(ii)=gf(ii)-res-add
  60 continue
  65 continue

c
c      impose displacement boundary conditions
c
      if(icount.eq.1)call bndy(ndof,gstif,gf,nbndry,ibndry,vbound)
      if(icount.gt.1)call bndy(ndof,gstif,gf,nbndry,ibndry,vpres)

c
c      solve system of equations, result in gf
c
      call solve(ndof,gstif,gf,0,detm,detm1)

c
c      update total displacement vector gd
c
      do 70 iei=1,ndof
  70 gd(iei)=gd(iei)+gf(iei)
      if(linear.eq.1)goto 80
      call converge(ndof,ncon,icount,tol,imax)

c
c      if no convergence (ncon=0) start next iteration
c
      if(ncon.eq.0)goto 3
  80      continue
      if(ncon.eq.1 .and. ncount.le.ninc)then
          call postpr(icount,ncount,kcall,ndof,pname,sname)
          if(ncount.eq.ninc)stop
          ncount=ncount+1
          icount=1
          goto 3
      endif
      return
      end

c
      subroutine bndy(ndof,s,s1,ndum,idum,vdum)
c
c      ..... .
c      subroutine used to impose boundary conditions on banded equations
c      ..... .

c
      implicit double precision (a-h,o-z)
      dimension s(2500,11),s1(2500)
      dimension idum(ndum*6),vdum(ndum*6)
      do 300 nb = 1, ndum*6
      ie = idum(nb)
      sval = vdum(nb)
      it=10

```

```

i=ie-11
do 100 ii=1,it
i=i+1
if (i .lt. 1)  go to 100
j=ie-i+1
s1(i)=s1(i)-s(i,j)*sval
s(i,j)=0.0
100 continue
s(ie,1)=1.0
s1(ie)=sval
i=ie
do 200 ii=2,11
i=i+1
if (i .gt. ndof)  go to 200
s1(i)=s1(i)-s(ie,ii)*sval
s(ie,ii)=0.0
200 continue
300 continue
return
end

c
c
c
c
c      subroutine converge(ndof,ncon,icount,tol,imax)
c.....c checks for convergnece using global displacement criterion
c.....c
c      implicit double precision (a-h,o-z)
c      common/proc/gstif(2500,11),gn(2500,11),gf(2500),gd(2500),vperm(250
0),
c           vpres(2500)
c
c      rcurr=0.
c      do 10 m=1,ndof
10      rcurr=rcurr + gd(m)*gd(m)
      if(icount.eq.1)rinit=rcurr
      if(icount.eq.1)ncon=0
      if(icount.eq.1)goto 20
c new criteria
      ratio=100. * abs(sqrt(rcurr)-sqrt(pvalue))/sqrt(rinit)
      if(ratio.le.tol)ncon=1
20      pvalue=rcurr
      write(*,100)ncon, ratio, rinit, rcurr
100     format(1x,'ncon= ',i3,3x,'ratio= ',d14.6,' rinit= ',d14.6,
      x           ' rcurr= ',d14.6)
      if(icount.eq.imax)write(6,200)
      if(icount.eq.imax)stop
200     format(1x,'icount equals imax')
      if(ncon.eq.0)icount=icount+1
      return
      end

c
c
c      subroutine rikspr(pname,sname)
c
c      implicit double precision (a-h,o-z)
c
c      character*64 gname, fname

```

```

c
c      common/chac/gname, fname
c
c      common/elas/ae, de, fe, he, ej, el, re, te, as, ds, fs
c
c      common/input/tol, table(250), delem(250), vbound(2500), distld,
c      . vconc(2500), ey, enu, ht, e1, e2, g12, enu12, enu21, g13, g23, pthick,
c      . rad, linear, isotro, isarch, ishape, inctyp, ninc, imax,
c      . nelem, nbndry, nbound(250, 6), ldtyp, nconc, iconc(2500),
c      . nplies, nforc, iforc(2500), nstres, istres(250), ibndry(2500),
c      . theta(20), idload(250), coord(251), width, nnod, pincr, eiter, ttpi
c      common/stf/stif(11,11), elp(11), eln(11,11), eld(11)
c
c      common/proc/gstif(2500,11), gn(2500,11), gf(2500), gd(2500), vperm(250
0),
c      . vpres(2500)
c
c      dimension
gld(2500), gld0(2500), gld1(2500), gdis(2500), gsti00(2500,11),
c      . gf0(2500), gd00(2500)
      ndof=nnod*5+nelem
      ncount=1
      ict=1
      iicut=0
      do 1 ii=1,ndof
1      gd(ii)=0.0d0
      do 2 ii=1,nbndry*6
      vpres(ii)=0.0d0
2      vperm(ii)=vbound(ii)
c
c      start new increment or iteration/
c      zero out global stiffness matrices and global force
c      vector
c
      tpincr=0.0
      if(ncount.eq.1)goto 2993
c
c      start new increment
c
3      if(iicut.eq.0)dss=dss*eiter/ict
2993  ict=1
      do 2992 ii=1,ndof
      gld0(ii)=0.0d0
2992  gd00(ii)=gd(ii)
c
c      start new iteration
c
4      do 5 ii=1,ndof
      gf0(ii)=0.0d0
      do 5 jj=1,11
      g stif(ii,jj)=0.0d0
5      gn(ii,jj)=0.0d0
      kcall=0
c
c      increment prescribed displacement for displacement control
c
c      if(linear.eq.1)goto 9
c      if(ict.ne.1)goto 9
c      do 7 ii=1,nbndry*6

```

```

c      if(ncount.eq.1.and.iicut.eq.0)vbound(ii)=vperm(ii)*table(1)
c 7  if(ncount.gt.1.or.iicut.gt.0)vbound(ii)=vperm(ii)*(table(ncount)-
c .      table(ncount-1))
c
c      loop over all elements for stiffness and forces
c
c 9  do 30 ielem=1,nelem
c      do 10 ii=1,11
c 10  eld(ii)=gd(ii+(ielem-1)*6)
c
c      kcall=kcall+1
c      call stiff(ielem,icount,ncount,kcall)
c
c      Assemble global stiffness array, gstif, global equilibrium
c      stiffness, gn, in banded form. Half-bandwidth=11. Also
c      assemble global force vector, gf.
c
c      nr=(ielem-1)*6 + 1
c      do 30 jj=0,10
c      gf0(nr+jj)=gf0(nr+jj)+elp(jj+1)
c      do 30 kk=1,11-jj
c      gstif(nr+jj,kk)=gstif(nr+jj,kk)+stif(jj+1,kk+jj)
c      if(linear.eq.1)goto 30
c      if(icount.eq.1 .and. ncount.eq.1 .and.iicut.eq.0)goto 30
c      gn(nr+jj,kk)=gn(nr+jj,kk)+eln(jj+1,kk+jj)
c 30  continue
c
c      impose force boundary conditions
c      at this point, gf=R
c
c      if(nconc.eq.0)goto 45
c      do 40 ii=1,nconc
c      nb=iconc(ii)
c 40  gf0(nb)=gf0(nb)+vconc(ii)
c 45  continue
c      do 47 ii=1,ndof
c 47  gf(ii)=gf0(ii)
c      do 48 ii=1,ndof
c      do 48 jj=1,11
c 48  gsti00(ii,jj)=gstif(ii,jj)
c      call bndy(ndof,gstif,gf,nbndry,ibndry,vbound)
c
c      call solve(ndof,gstif,gf,0,detm,detml)
c      dss0=0.0
c      do 49 ii=1,ndof
c      gdis(ii)=gf(ii)
c 49  dss0=dss0+gf(ii)*gf(ii)
c      if(icount.ne.1) go to 144
c          detm1=detm2
c          detm2=detm
c          if(ncount.eq.1.and.detm.lt.0.0 .and.iicut.eq.0) pincr=-pincr
c          if(ncount.eq.1.and.iicut.eq.0) dss=pincr*dsqrt(dss0)
c          if(ncount.ne.1.or.iicut.gt.0) pincr=
c          dss/dsqr(dss0)*detm*detml
c          *pincr1/dabs(pincr1)
c
c      attempt at offloading at bifurcation points
c
c      if(iicut.eq.1)pincr=-pincr
c

```

```

        pincr1=pincr

        prs=0.0
        do 142 ii=1,ndof
142      prs=prs+gf0(ii)*gld(ii)
        stifpa=pincr*prs
        do 143 ii=1,ndof
143      gld(ii)=pincr*gdis(ii)
144      continue

c      calculate the residual force vector for nonlinear
c      analysis. -[gn]*{gd}+R=-[k+n1/2+n2/3]*{q}+R=gf
c

        if(icount.eq.1)goto 69
          do 60 ii=1,ndof
            add=0.
            do 50 kk=1,ii-1
              if(ii-kk+1 .gt. 11)goto 50
              add=add+gn(kk,ii-kk+1)*gd(kk)
50      continue
            res=0.
            do 55 jj=1,11
              if(jj+ii-1 .gt. ndof)goto 55
              res=res + gn(ii,jj)*gd(jj+ii-1)
55      continue

c      add to existing gf which already contains R
c
        gf(ii)=gf0(ii)*(pincr+tpincr)-res-add
60      continue
65      continue

c      impose displacement boundary conditions
c
        call bndy(ndof,gsti00,gf,nbndry,ibndry,vbound)
c      if(icount.gt.1)call bndy(ndof,gstif,gf,nbndry,ibndry,vpres)
c
c      solve system of equations, result in gf
c
        call solve(ndof,gstif,gf,1,detm,detml)
c
c      through line 69 copied from Tsai's program
c
        a1=dss0
        a2=0.0
        a3=0.0
        do 147 ii=1,ndof
          a2=a2+(gld(ii)+gf(ii))*gdis(ii)
147      a3=a3+gf(ii)*(2.0*gld(ii)+gf(ii))
          d12=a2*a2-a1*a3
        write(6,*) d12,a1,a2,a3
c
c      if(d12.lt.0.0)then
c
c      deal with complex roots by cutting the search
c      radius (dss) in half
c
        do 2991 ii=1,ndof

```

```

2991 gd(ii)=gd00(ii)
      iicut=iicut + 1
      if(iicut.gt.10)then
          write(6,3000)
          stop
      endif
      dss=dss/2.0
      goto 3
      endif
      iicut=0
      dpinc1=(-a2+dsqrt(d12))/a1
      dpinc2=(-a2-dsqrt(d12))/a1
      theta1=0.0
      theta2=0.0
      do 148 ii=1,ndof
      gld0(ii)=gld(ii)
      gld(ii)=gld(ii)+gf(ii)+dpinc1*gdis(ii)
      gld1(ii)=gld0(ii)+gf(ii)+dpinc2*gdis(ii)
      theta1=theta1+gld0(ii)*gld(ii)
      theta2=theta2+gld0(ii)*gld1(ii)
148  continue
c      write(6,*) theta1,theta2
      thet12=theta1*theta2
      if(thet12.gt.0.0) go to 149
      dpincr=dpinc1
      if(theta2.gt.0.0) call chsign(gld,gld1,dpincr,dpinc2,ndof)
      go to 150
149  dpib=-a3/(a2*2.0)
      dpin1=dabs(dpib-dpinc1)
      dpin2=dabs(dpib-dpinc2)
      dpincr=dpinc1
      if(dpin2.lt.dpin1) call chsign(gld,gld1,dpincr,dpinc2,ndof)
150  pincr=pincr+dpincr
      69 continue
c
c      update total displacement vector gd
c
      do 70 ii=1,ndof
70  gd(ii)=gd(ii)+gld(ii)-gld0(ii)
      if(linear.eq.1)goto 80
      call converge(ndof,ncon,icount,tol,imax)
c
c      if no convergence (ncon=0) start next iteration
c
      if(ncon.eq.0)goto 4
80  continue
      if(ncon.eq.1 .and. ncount.le.ninc)then
          call postpr(icount,ncount,kcall,ndof,pname,sname)
          if(ncount.eq.ninc)stop
          ncount=ncount+1
          tpincr=tpincr+pincr
          goto 3
      endif
3000  format(1x,'More than 10 consecutive imaginary roots')
3010  format(1x,i2)
      return
      end
c
c
c

```

```

subroutine solve(ndof,band,rhs,ires,detm,detml)
c
c solve a banded symmetric system of equations
c
c
c implicit double precision (a-h,o-z)
dimension band(2500,11),rhs(2500)
meqns=ndof-1
if(ires.gt.0)goto 90
do 500 npiv=1,meqns
c      print*, 'npiv= ',npiv
npivot=npiv+1
lstsub=npiv+11-1
if(lstsub.gt.ndof) lstsub=ndof
do 400 nrow=npivot,lstsub
c      invert rows and columns for row factor
ncol=nrow-npiv+1
factor=band(npiv,ncol)/band(npiv,1)
do 200 ncol=nrow,lstsub
icol=ncol-nrow+1
jcol=ncol-npiv+1
200 band(nrow,icol)=band(nrow,icol)-factor*band(npiv,jcol)
400 rhs(nrow)=rhs(nrow)-factor*rhs(npiv)
500 continue
detm=1.0
detml=0.0
do 600 ii=1,ndof
c      write (*,*) 'BAND(' ,ii,'1)=' ,band(ii,1)
detml=detml+dlog10(dabs(band(ii,1)))
600 detm=detm*band(ii,1)/dabs(band(ii,1))
go to 101
90 do 100 npiv=1,meqns
npivot=npiv+1
lstsub=npiv+11-1
if(lstsub.gt.ndof) lstsub=ndof
do 110 nrow=npivot,lstsub
ncol=nrow-npiv+1
factor=band(npiv,ncol)/band(npiv,1)
110 rhs(nrow)=rhs(nrow)-factor*rhs(npiv)
100 continue
c      back substitution
101 do 800 ijk=2,ndof
npiv=ndof-ijk+2
rhs(npiv)=rhs(npiv)/band(npiv,1)
lstsub=npiv-11+1
if(lstsub.lt.1) lstsub=1
npivot=npiv-1
do 700 jki=lstsub,npivot
nrow=npivot-jki+lstsub
ncol=npiv-nrow+1
factor=band(nrow,ncol)
700 rhs(nrow)=rhs(nrow)-factor*rhs(npiv)
800 continue
rhs(1)=rhs(1)/band(1,1)
return
end
c
c
c
subroutine chsign(gld,gld1,dpincr,dpinc2,ndof)

```

```

implicit double precision (a-h,o-z)
dimension gld(2500),gld1(2500)
do 100 i=1,ndof
100 gld(i)=gld1(i)
dpincr=dpinc2
return
end

c
c
c
c subroutine stiff(ielem,icount,ncount,kcall)
c
c implicit double precision (a-h,o-z)
c character*64 gname, fname
c common/chac/gname, fname
c
c common/input/tol,table(250),delem(250),vbound(2500),distld,
c . vconc(2500),ey,enu,ht,e1,e2,g12,enu12,enu21,g13,g23,pthick,
c . rad,linear,isotro,isearch,ishape,inctyp,ninc,imax,
c . nelem,nbndry,nbound(250,6),ldtyp,nconc,iconc(2500),
c . nplies,nforc,iforc(2500),nstres,istres(250),ibndry(2500),
c . theta(20),idload(250),coord(251),width,nnod,pincr,eiter,ttpi
c
c common/elas/ae,de,fe,he,ej,el,re,te,as,ds,fs
c
c common/stf/stif(11,11),elp(11),eln(11,11),eld(11)
c
c common/shp/dsf(7,11)
c
c dimension bmk(7,7),bmn1(7,7),bmn2(7,7),
c . gauss4(4),wt4(4),gauss7(7),wt7(7),q(7),dsftr(11,7),
c . pkt(7,7),pkn(7,7),pktd(7,11),pknd(7,11),gauss5(5),wt5(5)
c
c data gauss4/0.8611363115d0,0.3399810435d0,-0.3399810435d0,
c . -0.8611363115d0/
c data wt4/0.3478548451d0,0.6521451548d0,0.6521451548d0,
c . 0.3478548451d0/
c data gauss5/0.9061798459d0,0.5384693101d0,0.0d0,-0.5384693101d0,
c . -0.9061798459d0/
c data wt5/0.2369268851d0,0.4786286705d0,0.5688888889d0,
c . 0.4786286705d0,0.2369268851d0/
c data gauss7/0.9491079123d0,0.7415311856d0,0.4058451513d0,
c . 0.0d0,-0.4058451513d0,-0.7415311856d0,-0.9491079123d0/
c data wt7/0.1294849662d0,0.2797053915d0,0.3818300505d0,
c . 0.4179591836d0,0.3818300505d0,0.2797053915d0,0.1294849662d0/
c
c initialize stiffness arrays and load array
c
c do 10 ii=1,11
c elp(ii)=0.0
c do 10 jj=1,11
c stif(ii,jj)=0.0
10 eln(ii,jj)=0.0
c
c set number of gauss points for interpolation
c
c ngp=5
c if(ncount.eq.1 .and. icount.eq.1)ngp=4
c if(linear.eq.1)ngp=4
c

```

```

ek1=-4./(3.*ht**2)
if (isearch.eq.1) p1=1./rad
if(ncount.eq.1 .and. icanth.eq.1 .and. kcall.eq.1 .and.
isearch.eq.1)
    .    call beamk(bmk,ek1,p1)
    if(ncount.eq.1 .and. icanth.eq.1 .and. kcall.eq.1 .and.
isearch.eq.0)
    .    call sbeamk(bmk,ek1)
c
c    loop over gauss points
c
do 100 ii=1,ngp
if(ngp.eq.4)eta=gauss4(ii)
if(ngp.eq.5)eta=gauss5(ii)
if(ngp.eq.7)eta=gauss7(ii)
call shape(eta,ielem,aa)

c
c    multiply element displacement vector, eld (this is 'q'
c    in the thesis) by the shape function matrix, dsf, to get
c    the displacement gradient vector(d(s) in thesis, q here)
c
do 20 kk=1,7
20 q(kk)=0.0
c
do 30 jj=1,7
do 30 kk=1,11
30 q(jj)=q(jj)+dsf(jj,kk)*eld(kk)

c
c    initialize bmn1, bmn2
c
do 35 kk=1,7
do 35 jj=1,7
bmn1(jj,kk)=0.0d0
35 bmn2(jj,kk)=0.0d0

c
c    skip bmn1 and bmn2 comps first time through
c
if(icanth .eq. 1 .and. ncount .eq. 1)goto 37
c
if(isearch.eq.1)call beamn1(q,bmn1,ek1,p1)
if(isearch.eq.1)call beamn2(q,bmn2,ek1,p1)
if(isearch.eq.0)call sbmn1(q,bmn1,ek1)
if(isearch.eq.0)call sbmn2(q,bmn2,ek1)
37    continue
c
c    transpose the shape function matrix
c
do 40 jj=1,7
do 40 kk=1,11
40 dsftr(kk,jj)=dsf(jj,kk)

c
c    create element independent incremental stiffness array,
c    pkt, and element ind. equilibrium stiffness array, pkn
c
do 50 jj=1,7
do 50 kk=1,7
pkt(jj,kk)=bmk(jj,kk)+bmn1(jj,kk)+bmn2(jj,kk)
50 pkn(jj,kk)=bmk(jj,kk)+bmn1(jj,kk)/2.+bmn2(jj,kk)/3.

c

```

```

c      post-multiply each array by the shape function matrix
c
c      do 60 jj=1,7
c      do 60 kk=1,11
c      pktd(jj,kk)=0.0
c      pknd(jj,kk)=0.0
c      do 60 ll=1,7
c      pktd(jj,kk)=pktd(jj,kk) + aa*pkt(jj,ll)*dsf(ll,kk)
c 60  pknd(jj,kk)=pknd(jj,kk) + aa*pkn(jj,ll)*dsf(ll,kk)

c      Finally, pre-multiply these new arrays by the transpose
c      of the shape function matrix to get the element incremental
c      stiffness, stif, and element equilibrium stiffness, eln.
c      Also multiply by the weighting factor for this particular
c      gauss point. Note that these arrays are zeroed outside the
c      loop over the gauss points since they accumulate (integrate)
c      data over all the gauss points.
c
c      if/ngp.eq.4)wt=wt4(ii)
c      if/ngp.eq.5)wt=wt5(ii)
c      if/ngp.eq.7)wt=wt7(ii)
c      do 70 jj=1,11
c      do 70 kk=1,11
c      do 70 ll=1,7
c      stif(jj,kk)=stif(jj,kk)+wt*width*dsftr(jj,ll)*pktd(ll,kk)
c      eln(jj,kk)=eln(jj,kk)+wt*width*dsftr(jj,ll)*pknd(ll,kk)
c 70  continue
c 100  continue
c      write(6,1010)ielem
c      write(6,1005)
c      do 900 ii=1,11
c900  write(6,1000) (stif(ii,jj),jj=1,11)
c1000 format(9(2x,d12.5))
c1005 format(/, 'stif')
c1010 format(i4)
c      return
c      end

c
c
c      subroutine shape(eta,ielem,aa)
c
c      implicit double precision (a-h,o-z)
c
c
c      common/shp/dsf(7,11)
c
c
c      common/input/tol,table(250),delem(250),vbound(2500),distld,
c      . vconc(2500),ey,enu,ht,e1,e2,g12,enu12,enu21,g13,g23,pthick,
c      . rad,linear,isotro,isarch,ishape,inctyp,ninc,imax,
c      . nelem,nbndry,nbound(250,6),ldtyp,nconc,iconc(2500),
c      . nplies,nforc,iforc(2500),nstres,istres(250),ibndry(2500),
c      . theta(20),idload(250),coord(251),width,nnod,pincr,eiter,ttpi
c
c      initialize shape function matrix
c
c      do 10 ii=1,7
c      do 10 jj=1,11
c 10  dsf(ii,jj)=0.0

```

```

c
c      aa=(coord(ielem+1)-coord(ielem))*0.5
c
c      enter values into dsf
c      these include jacobian terms
c
c
c      Q1,Q3,Q2 and derivatives
c
dsf(1,1)=0.5*(eta**2-eta)
dsf(1,6)=1.0-eta**2
dsf(1,7)=0.5*(eta**2+eta)
dsf(2,1)=(eta-0.5)/aa
dsf(2,6)=-2.0*eta/aa
dsf(2,7)=(eta+0.5)/aa
dsf(3,4)=0.25*(2.0-3.0*eta+eta**3)
dsf(3,5)=0.25*aa*(1.0-eta-eta**2+eta**3)
dsf(3,10)=0.25*(2.0+3.0*eta-eta**3)
dsf(3,11)=0.25*aa*(-1.0-eta+eta**2+eta**3)
dsf(4,4)=0.25*(-3.0+3.0*eta**2)/aa
dsf(4,5)=0.25*(-1.0-2.0*eta+3.0*eta**2)
dsf(4,10)=0.25*(3.0-3.0*eta**2)/aa
dsf(4,11)=0.25*(-1.0+2.0*eta+3.0*eta**2)
dsf(5,4)=0.25*6.0*eta/aa**2
dsf(5,5)=0.25*(-2.0+6.0*eta)/aa
dsf(5,10)=-0.25*6.0*eta/aa**2
dsf(5,11)=0.25*(2.0+6.0*eta)/aa
dsf(6,2)=0.25*(2.0-3.0*eta+eta**3)
dsf(6,3)=0.25*aa*(1.0-eta-eta**2+eta**3)
dsf(6,8)=0.25*(2.0+3.0*eta-eta**3)
dsf(6,9)=0.25*aa*(-1.0-eta+eta**2+eta**3)
dsf(7,2)=0.25*(-3.0+3.0*eta**2)/aa
dsf(7,3)=0.25*(-1.0-2.0*eta+3.0*eta**2)
dsf(7,8)=0.25*(3.0-3.0*eta**2)/aa
dsf(7,9)=0.25*(-1.0+2.0*eta+3.0*eta**2)

c
c      temporary print
c
c      do 100 ii=1,7
c 100  write(6,1000) (dsf(ii,jj),jj=1,11)
c 1000      format (9(2x,d12.5))
      return
      end
c
c
c      subroutine postpr(icount,ncount,kcall,ndof,pname,sname)
c
c      implicit double precision (a-h,o-z)
c
c      character*64 gname,fname
c
c
c      common/chac/gname,fname
c
c      common/elas/ae,de,fe,he,ej,el,re,te,as,ds,fs
c
c      common/input/tol,table(250),delem(250),vbound(2500),distld,
c      . vconc(2500),ey,enu,ht,e1,e2,g12,enu12,enu21,g13,g23,pthick,
c      . rad,linear,isotro,isearch,ishape,inctyp,ninc,imax,

```

```

. nelem,nbndry,nbound(250,6),ldtyp,nconc,iconc(2500),
. nplies,nforc,iforc(2500),nstres,istres(250),ibndry(2500),
. theta(20),idload(250),coord(251),width,nnod,pincr,eiter,ttpi
c
c common/stf/stif(11,11),elp(11),eln(11,11),eld(11)
c
c common/proc/gstif(2500,11),gn(2500,11),gf(2500),gd(2500),vperm(250
0),
. vpres(2500)
c
c dimension vforc(2500),xcoord(251),ycoord(251)
pi=3.14159
c
c if ishape=1 print global x,y coords to file bshape
c if ishape=2 figure out symmetric coords as well
c
c if (isarch.eq.0) then
c     go to 423
c end if
if(ishape.eq.1.or.ishape.eq.2.and.ncount.eq.1.and.isarch.eq.1)then
    open(8,file=sname,status='new')
    do 1 ii=1,nnod
        if(ishape.eq.2)then
            xcoord(nnod-1+ii)=rad*cos(pi/2.0-coord(ii)/rad)
            xcoord(nnod+1-ii)=-rad*cos(pi/2.0-coord(ii)/rad)
            ycoord(nnod-1+ii)=rad*sin(pi/2.0-coord(ii)/rad)
            ycoord(nnod+1-ii)=ycoord(nnod-1+ii)
        else
            xcoord(ii)=-rad*cos(pi/2.0+coord(ii)/rad-
coord(nnod)/(2*rad))
            ycoord(ii)=rad*sin(pi/2.0+coord(ii)/rad-coord(nnod)/(2*rad))
        endif
    1 continue
    do 100 ii=1,nnod
        write(8,2000)xcoord(ii),ycoord(ii)
        if(ishape.eq.2)then
            do 110 ii=2,nnod
                write(8,2000)xcoord(nnod+ii-1),ycoord(nnod+ii-1)
            endif
        110 write(8,2010)
        endif
c
c global displacements for straight beams
c
423 if(ishape.eq.1.and.ncount.eq.1.and.isarch.eq.0)then
    open(8,file=sname,status='new')
    do 2 ii=1,nnod
        xcoord(ii)= coord(nnod)-coord(ii)
        ycoord(ii)=0.0
    2 write(8,2000)xcoord(ii),ycoord(ii)
    write(8,2010)
    endif
c
c print out global displacements
c
    write(6,1000)
    write(6,1010)
    write(6,1020)ncount,iconc
    write(6,1030)
    do 90 ii=0,nnod-1

```

```

c
c x,y for arches
c
if(ishape.eq.1.and.isarch.eq.1)then
if(ishape.eq.1.and.isarch.eq.1.and.ii.eq.0)write(8,2020)ncount
  xcoord(ii+1)=-((rad-gd(6*ii+4))*  

  .  cos(pi/2.0+coord(ii+1)/rad-coord(nnod)/(2*rad)+  

  .  gd(6*ii+1)/rad)
  ycoord(ii+1)=((rad-gd(6*ii+4))*  

  .  sin(pi/2.0+coord(ii+1)/rad-coord(nnod)/(2*rad)+  

  .  gd(6*ii+1)/rad)
  write(8,2000)xcoord(ii+1),ycoord(ii+1)
endif
if(ishape.eq.2.and.isarch.eq.1)then
if(ii.eq.0)write(8,2020)ncount
  xcoord(nnod+ii)=((rad-gd(6*ii+4))*  

  .  cos(pi/2.0-coord(ii+1)/rad+gd(6*ii+1)/rad)
  xcoord(nnod-ii)=-xcoord(nnod+ii)
  ycoord(nnod+ii)=((rad-gd(6*ii+4))*  

  .  sin(pi/2.0-coord(ii+1)/rad+gd(6*ii+1)/rad)
  ycoord(nnod-ii)=ycoord(nnod+ii)
endif
c
c x,y for straight beams
c
if(ishape.eq.1.and.isarch.eq.0)then
if(ishape.eq.1.and.isarch.eq.0.and.ii.eq.0)write(8,2020)ncount
  xcoord(ii+1)=coord(nnod)-coord(ii+1)-gd(6*ii+1)
  ycoord(ii+1)=-gd(6*ii+4)
  write(8,2000)xcoord(ii+1),ycoord(ii+1)
endif
write(6,1040)ii+1,(gd(6*ii+jj),jj=1,5)
90 write(6,1050)gd(6*ii+6)
if(ishape.eq.2.and.isarch.eq.1)then
  do 95 ii=1,2*nnod-1
95  write(8,2000)xcoord(ii),ycoord(ii)
endif
if(ishape.ge.1)write(8,2010)
c
c compute equivalent forces requested
c
3  do 5 ii=1,ndof
  gf(ii)=0.0d0
  do 5 jj=1,11
  gstif(ii,jj)=0.0d0
5   gn(ii,jj)=0.0d0
c
c loop over all elements for stiffness and forces
c
9  do 30 ielem=1,nelem
  do 10 ii=1,11
10  eld(ii)=gd(ii+(ielem-1)*6)
c
  call stiff(ielem,icount,ncount,kcall)
c
c Assemble global stiffness array, gstif, global equilibrium
c stiffness, gn, in banded form. Half-bandwidth=11. Also
c assemble global force vector, gf.
c
  nr=(ielem-1)*6 + 1

```

```

do 30 jj=0,10
gf(nr+jj)=gf(nr+jj)+elp(jj+1)
do 30 kk=1,11-jj
gstif(nr+jj,kk)=gstif(nr+jj,kk)+stif(jj+1,kk+jj)
if(linear.eq.1)goto 30
gn(nr+jj,kk)=gn(nr+jj,kk)+eln(jj+1,kk+jj)
30 continue
c
c calculate the residual force vector for nonlinear
c analysis. -[gn]*{gd}+R=-[k+n1/2+n2/3]*{q}+R=gf
c
      do 60 jj=1,nforc
ii=iforc(jj)
add=0.
do 50 kk=1,ii-1
if(ii-kk+1 .gt. 11)goto 50
add=add+gn(kk,ii-kk+1)*gd(kk)
50 continue
res=0.
do 55 ll=1,11
if(ll+ii-1 .gt. ndof)goto 55
res=res + gn(ii,ll)*gd(ll+ii-1)
55 continue
c
c compute nodal force
c
vforc(jj)=res+add
60 continue
c
c print nodal forces and create plot file
c
open(7,file=pname,status='new')
if(ncount.eq.1)write(7,1065)0.0,0.0,0.0
write(6,1060)
do 70 ii=1,nforc
write(7,1065)gd(iforc(ii)),gd(iforc(ii)-3),vforc(ii)
70 write(6,1070)iforc(ii),vforc(ii)
1000 format(/)
1010 format(1x,'Results of nonlinear analysis')
1020 format(1x,'increment=',i3,' iteration=',i3)
1030 format(1x,'Node',7x,'V',12x,'Psi-s',9x,'Psi-ss',9x,'W',9x,'W-s')
1040 format(1x,i4,5(2x,d12.5))
1050 format(1x,'Midnode v:',3x,d12.5)
1060 format(1x,/, 'Equivalent nodal forces:')
1065 format(1x,d12.5,2x,d12.5,2x,d12.5)
1070 format(1x,'DOF no:',i4,2x,'Force:',1x,d12.5)
2000 format(1x,f12.5,2x,f12.5)
2010      format(/)
2020 format(/,1x,i4)
      return
      end

```

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